

Location Results for Brace Spacing in Gap Tubular “K” Joints Using FEM

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ABSTRACT

The 8-node iso-parametric thin shell element was employed in the study of stress concentrations in the welded tubular “K” joint. Element equilibrium equations were derived using isoparametric formulation based on thin shell theory. After assembly, the resulting system equations were solved using existing fortran programs. Numerical experiments were conducted to isolate and locate ideal gap (positions) for the two braces of the “K” joint. The nominal stresses were calculated from which stress concentration factors were obtained. The resulting stress concentration factors were presented both as tables and as figures. A good agreement between our solutions and those for model joints in the literature is good and acceptable. It was found that the wider apart the brace spacing is, the weaker the strength of the joint. It was also found that the best location for the braces occurs when the stress level changes sign either from positive to negative or vice versa at a critical sampling point.

Keywords: Tubular Joint; Finite Element; Computer Simulation; Structural Joints; Effect of Bracing; Stress Concentration; Stress Distribution

1. Introduction

Lapped joints generally, and lapped tubular joints in particular are known to be stronger than gapped joints following a heuristic reasoning. However, for the purpose of bracing, it is not always feasible or even possible to brace a structural system with lapped joint throughout. Hence a strong case arises from using gapped tubular joints which are in many cases welded on site due to ease of construction and transportation. As usual, one of the problems with welded steel construction is the development of stress concentration or ‘hot spot’ stresses. The hot spots affect the strength of both lapped and gapped joints.

The problem of hot spot stresses or stress concentrations has been investigated by Kuang *et al.* [1] in their work on stress concentrations in tubular joints. Potuin *et al.* [2] also investigated stress concentrations in tubular joints. Gibstein [3] used a parametric study to investigate stress concentrations in T joints. Wordsworth and Smedly [4] investigated stress concentrations in offshore tubular joints without stiffeners. However, joints of hollow square, rectangular and tubular sections are very often used in very important constructions such as large space roofs, offshore platforms [4] and trussed bridges [5,6] instead of their solid counterparts to reduce dead loads due to self weights. However, failure of these systems due to joint failures still pose engineering challenges. In order

to understand why these joints still fail, So and So [7] carried out a study of “hot spot” stresses in a tubular “K” joint with combined in-plane and out-of-plane loadings. May be due to combined in-plane and out-of-plane loading, very high “hot spot” stresses were obtained from their model. Although they studied the effect of load variation, they did not study the effect of varying the gap between braces, on the strength of joint they studied. In order to understand the behaviour of these joints further, more studies were initiated by Paul *et al.* [8] to study the ultimate resistance of un-stiffened multi-planar tubular “T” and “K-K” joints. Parker [9] proposed the use of concrete filled hollow square section (“HSS”) joints to improve the strength of such joints. The results of the finite element study on square hollow “K” joints by Jiki [10] seem to represent more realistically the behaviour of the joints but it was realised that the sharp corner edges of the section increased stress concentrations. This effect of geometry was also observed by Jiki [11] in the study of a rectangular hollow (RH) “K” joints. For the purpose of comparison using the finite element method to study the effect of section geometry on the distribution of stress concentrations in hollow joints, the same 8-node iso-parametric thin shell element is used to model these sections with different geometries. In all joints modelled by the finite element method, welded joint construction and the 8 node iso-parametric shell element were used through

out to facilitate room for comparison.

It can be seen from the above review that the effect of stress concentrations on strength of tubular joints has been found to be a problem generally [5,6].

However, the particular problem of stress concentrations in lapped and gapped tubular joints with respect to effective location of the spacing of the braces has not hitherto been properly addressed.

The purpose of this study is to investigate the effect of effective location of braces (brace spacing) on the strength of the gapped tubular “K” joints. The effect of section geometry on the distribution of stress concentration factors in hollow section (HS) or tubular joints is also investigated. The finite element method of analysis was used in the investigation of the static stress concentrations.

2. Equilibrium Equations

The usual curved thin shell iso-parametric element was used for the present investigation (e.g. see Hinton and Owen [12]). The shape of the element is shown in **Figure 1**. Using potential energy formulation and the Rayleigh-Ritz process, the equilibrium equations were derived as follows:

The element total potential energy functional is given as :

$$\pi^e = U^e - W^e \quad (15)$$

where U^e is the strain energy of the element W^e is the external work done by the element during deformation.

In a compact formulation Equation (15) can be written as:

$$\pi^e = \iint_A [B]^T \{\delta\}^T [D][B][\delta] dA - \int_b q \{\delta\} db \quad (16)$$

$$= \frac{1}{2} \{\delta\}^T [K^e] \{\delta\} - Q^e \{\delta\} \quad (17)$$

In which $[K^e]$ is the element stiffness matrix Q^e is the element boundary loads, such that:

$$[K^e] = \int_{-1}^1 \int_{-1}^1 [B]^T [D][B] \det \det [J] d\xi d\beta \quad (18)$$

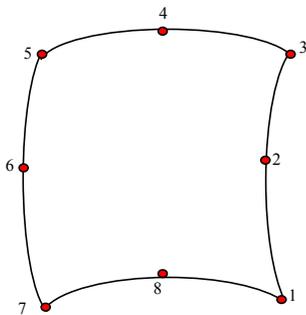


Figure 1. An 8-node isoparametric element.

$$Q^e = \sum_{i=1}^n q \quad (19)$$

After assembly we solve the system equations of Equation (20) for displacements.

$$[K^s] \{\Delta\} - \{Q\} = \{0\} \quad (20)$$

The finite element stresses are calculated from

$$\sigma_F = [D][B]\{\delta\} \quad (21)$$

in which

$$D = \frac{E}{(1-\nu^2)} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & \frac{1-\nu}{2} \end{bmatrix} \forall \text{ plane stress.} \quad (22)$$

3. Solution of System Equilibrium Equations

The system equilibrium equations for the present problem were solved after application of boundary conditions using the frontal solution algorithm and code developed by Hinton and Owen [12]. To check the validity of the program used we compare our results with those of So and So [7] for the case of in-plane loads as shown in **Table 1** herein. As we can see from that table our results or findings closely agree with theirs and is therefore acceptable. We have also investigated the effect of increasing the gap between braces on the stress concentration factors and our findings in this regard are shown in **Tables 2** and **3**. Having been satisfied with the performance of our program as a tool for the analysis, we have now studied the effect of variable brace spacing or gap on the strength of gap tubular “K” joints in details as presented in the tables and figures in this work.

4. Investigation of Stress Concentrations

The finite element discretization and loading of the joint for purpose of stress analysis is shown in **Figure 2**. Due to symmetry only half of the “K” joint used for the present study. After assembly we solve Equation (20) for nodal displacements and use Equation (21) to calculate stresses using Fortran code developed by Hinton and Owen [12]. After the finite element stresses were calculated, we also calculated nominal stresses using the usual expression for combined bending and axial effect as:

$$\sigma_N = \frac{P}{A} + \frac{My}{I} \quad (24)$$

and the nominal shear stresses are calculated from:

$$\tau_N = \frac{Q}{2\pi Rt} \quad (25)$$

Table 1. Comparison of S_{CF} s for “K” tubular joints subjected to inplane loads.

Load Type	S_{CF}					
	Obtained by Authors	Efthymiou and Durkin [7]	Obtained by So and So [7]	Authors	Efthymiou and Durkin [7]	Obtained by So and So [7]
In-plane bending	2.795	2.810	2.678	3.158	3.243	3.127

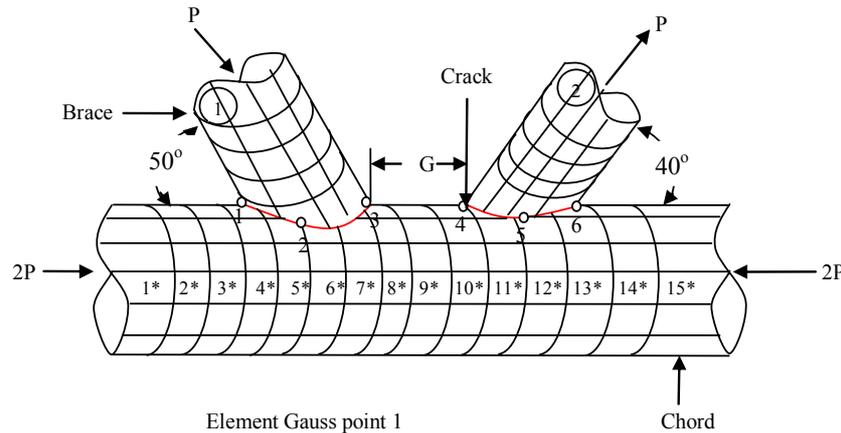


Figure 2. A finite element mesh for the K joint.

Table 2. Effect of increasing brace spacing (gap) on S_{CF} at corner of top chord (Brace angles = 50/40 deg).

Sampling point	50 mm gap	60 mm gap	70 mm gap
1	-0.99	-0.99	-0.83
2	-1.18	-1.18	-1.02
3	-0.70	-0.80	-1.92
4	-0.63	-0.86	-0.54
5	-1.22	1.73	+4.19
6	-0.69	-0.58	+2.59
7	-1.35	-0.96	-0.44
8	-0.88	-1.74	-0.99
9	-1.02	-0.83	-1.63
Average = -0.962 Average = -0.960 Average = -0.066			

Table 3. Effect of increasing brace spacing (gap) on S_{CF} at corner of top chord (Brace angles = 50/50 deg).

Sampling point	50 mm gap	60 mm gap	70 mm gap
1	1.19	1.17	-1.10
2	1.10	1.07	-1.15
3	0.72	0.73	-1.45
4	0.71	0.65	-0.64
5	1.26	1.10	+6.32
6	0.65	0.42	-1.15
7	1.39	1.44	-4.70
8	0.75	0.83	-1.16
9	0.98	1.12	-0.48
Average = +0.972 Average = +0.950 Average = +0.612			

in which Q is applied shear force, R is shell radius and t is shell thickness. The gap between the two braces in the first case was kept constant at 50 mm and stress concentration factor S_{CF} was obtained as

$$S_{CF} = \frac{\sigma_F}{\sigma_N} \nabla \text{ flexure} \tag{26}$$

or

$$S_{CF} = \frac{\tau_F}{\tau_N} \nabla \text{ shear} \tag{27}$$

where σ_F is finite element stress in the longitudinal axis of the chord τ_F is shear stress in the chord. σ_N is

$$\text{nominal stress given as } \sigma_N = \frac{P}{A} + \frac{My}{I}$$

5. Applications

Geometric parameters for the “K” joint.

Example 1.

For the purpose of comparison with our solution the dimensions of the “K” joint studied by So and So [7] are used here:

$$D = 232.85 \text{ mm}, d = 168.30 \text{ mm},$$

$$T = t = 12.7 \text{ mm}, \theta = 45^\circ.$$

We have also:

t/T = Brace of chord thickness ratio = 1.0,
 D/T = Chord diameter to thickness ratio = 25.50,
 d/D = Brace of chord diameter ratio = 0.52,
 L/D = Chord length to diameter ratio = 6.18,
 g/D = Brace separation to chord diameter ratio = 0.015
 From g/D = ratio of 0.015, we calculate g to be $0.013 \times 323.85 = 4.86$ mm, say =5 mm.

And $L/D = 6.18$, we calculate L used by So and So [7]: $L = 6.18 \times 323.85 = 2001.4$ mm, say 2000 mm.

We next subject the joint with the above details to in-plane loading for the purpose of bending stress analysis. The result of analysis from our finite element code is compared with those of So and So [7] and Efthymiou and Durkin.

Our result for the chord is 0.53% lower than that of Efthymiou and Durkin and 4.37% higher than that of So and So [7], while for the chord; it is 2.37% lower that of Efthymiou and Durkin and 0.98% higher than that So and So [7]. In this case, we conclude that the comparison between our solution and that of the two published authors is overall less than 5% difference and is good and acceptable.

Example 2.

In this example we investigate the effect of damage and brace spacing/ separation on the strength of the “K” tubular joint. The geometric details of the joint is as follows:

Chord:

$D = 350$ mm, $t = T = 6.30$ mm, $L = 650$ mm; Load = 5T.

Brace:

$d = 250$ mm, $t = T = 6.30$ mm, $L = 650$ mm; Load = 2T.

The result of the peak (adverse) stress concentrations obtained mainly from top chord of the “K” joint are presented here in **Tables 2** and **3**.

6. Discussion of Results

Bracings in structural systems provide strength to the systems. However, there is no universal method of selecting locations for a selected bracing system. Usually the designer selects brace separations or gaps based on previous experience or heuristic reasoning. In this work we have used the finite element method as a tool to carry out some numerical experiments on locations of effective brace separations or spacings rather than use usual rule of thumb to select the desired spacings or gap between braces. By varying the gap between two braces and making computer runs produce a set of results. A close look at the results produced shows a trend in degradation of stresses as the gap is increased. This can give the analyst or the designer some idea of which brace separation to choose in the design.

In **Tables 2** and **3** we have presented the results of our study for three brace separations e.g. 50 mm, 60 mm and

70 mm respectively. We see in the tables that from 50 mm to 60 mm spacing/separation there is no alternating stress concentration factors S_{CFs} i.e. from positive to negative; that is, we have monotonic stress pattern and there is no stress reversal. However, as we increase the separation gap to 70 mm, we see the alternating pattern of stress concentration factors. This marks the limit of brace separation for the joint details above.

7. Conclusions

From our study of gapped tubular “K” joints we conclude as follows:

1) Brace separation distance for “K” tubular joint should be carefully selected so as to reduce over stress concentrations.

2) For best result, the joint should be lapped.

3) However, if the joint must be spaced, then based on our findings in **Tables 2** and **3** and **Figures 3** and **4**, 50 mm gap should be the limit of the spacing or gap. This is because after the 50 mm brace spacing or gap, adverse stress concentrations are observed when the spacing is increased beyond the 50 mm mark. In any case, the geometric parameters of the joint should be considered in order to avoid joint over stress.

4) We recommend or propose here that the finite element method be used as an ideal tool for the location of best brace spacing or gap (g).

5) Using our example joint studied here, we propose as a guide in the location result, that the ratio of the spacing

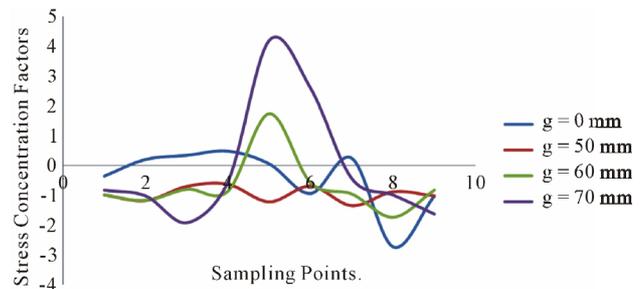


Figure 3. Effect of gap (g) on S_{CF} of top chord (centre).

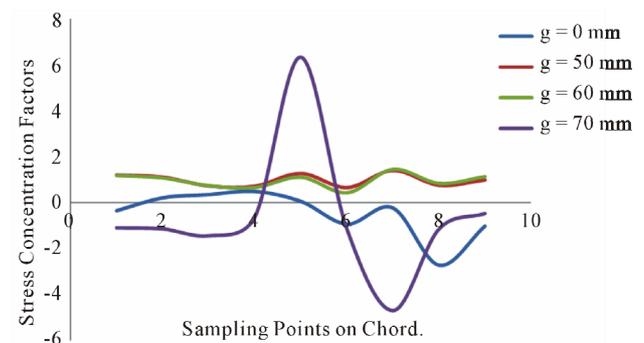


Figure 4. Effect of gap (g) on S_{CF} of top Chord (edge).

or gap g to the chord diameter D should not be greater than 1, that is, $g/D \leq 1$.

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