

# Visualization of a Laminar Necklace Vortex System in Front of a Vertical Rectangular Plate on a Ground Wall

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## Abstract

Various properties of a necklace vortex system formed around a rectangular plate standing vertically on a flat ground wall were investigated by visualizations produced by injecting fluorescent dye into a water channel flow. As a result, it was found that the necklace vortex pattern had three steady systems depending on the Reynolds number  $Re_h$ , the relative height  $h/\delta$  of the rectangular plate compared with the laminar boundary layer thickness and the aspect ratio  $w/h$ . As it is expected that the aspect ratio of the rectangular plate will typify the projected area configuration of various three-dimensional bodies, the aspect ratio is varied widely from 0.5 to 7.0. The transitional boundaries of  $Re_h$  and  $h/\delta$  in the 2-vortex, 4-vortex and 6-vortex systems for each aspect ratio decreased when  $w/h < 3.0$ , and increased when  $w/h \geq 3.0$  as  $w/h$  increased. The x-direction length of the main vortex position  $X_{V1}/h$  was almost constant when  $w/h < 3.0$ , and decreased when  $w/h \geq 3.0$  as  $Re_h$  increased. Then, the separation length  $r_1/h$  increased when  $w/h < 3.0$  and  $Re_h < 1000$ , and became almost constant when  $w/h \geq 3.0$  and  $Re_h \geq 1000$  as  $w/h$  and  $Re_h$  increased. Moreover, the increase of  $r_1/h$  is notably rapid when  $w/h < 3.0$ .

## Keywords

Flow Visualization, Necklace Vortex System, Rectangular Plate, Laminar Boundary Layer

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## 1. Introduction

It is well known that the necklace vortex system is formed around the three-dimensional bluff body of various geometrical shapes such as a circular cylinder [1]-[5], a square cylinder [6]-[9], a vertical plate [10]-[12] and others [13] [14] on a flat ground wall. The necklace vortex system also appears in the juncture flow field caused

between a three-dimensional bluff body and the flat ground wall. Such necklace vortices would be expected to control the flow field near the bluff body and to promote heat transfer between the fluid and wall surfaces.

Baker [1] showed that a steady necklace vortex system around a short cylinder whose diameter to height was 2 was classified into three patterns: a 2-vortex system, a 4-vortex system and a 6-vortex system, based on its Reynolds number and ratio of cylinder diameter to laminar boundary layer thickness by using a critical point concept. Moreover, the unsteady behavior of the necklace vortices was examined using smoke visualization and measurements of vortex frequency by Baker [2] [3] and Thomas [4]. Visbal [5] obtained streamlines of the steady necklace vortices and the unsteady process through the use of numerical calculations. Seal *et al.* [6] [7] studied a breakaway necklace vortex system in front of a long rectangular cylinder which protrudes through the surface of water. Therefore, the flow field was different from a three-dimensional flow field which flows over a bluff body. Subsequently, Lin *et al.* investigated the characteristics of a necklace vortex around a slender rectangular cylinder [8] and a slender rectangular plate [10] whose width to height in their projected area was from 0.25 to 2.0. They paid attention to the relationship between the primary vortex and the down flow in front of the juncture between the cylinder and the ground wall over a wide range of Reynolds numbers and boundary layer thicknesses, and also discussed the structure of the oscillation and breakaway vortex system in detail. Matsuguchi *et al.* [9] displayed a classification map of the necklace vortex system in front of a rectangular cylinder with a width-to-height ratio  $w/h = 2$  in the projected area. Wei *et al.* [13] [14] experimentally observed the steady necklace vortex and unsteady oscillating vortex in front of a long cylinder with various cross section shapes including circles, rectangles and prisms. On the other hand, Okamoto *et al.* [11] showed the existence of a necklace vortex at a high Reynolds number through wind tunnel testing of the pressure distributions and oil film patterns observed on the ground wall surface around rectangular plates with width-to-height ratios 2, 4 and 7. Moreover, Sakamoto *et al.* [12] measured the surface pressure distribution on rectangular plates with width-to-height ratios from 0.5 to 10 submerged in turbulent boundary layer flows, and suggested the existence of a turbulent necklace vortex. However, in many traditional investigations, the characteristics of the necklace vortex system over the wide range of width-to-height ratios in projected areas of three-dimensional bluff bodies have largely remained unrevealed. Then, if a rectangular plate is employed as the typical shape of three-dimensional bluff bodies, the width-to-height ratio in the projected area may be replaced with the aspect ratio of the plate.

Therefore, the aim of this study is to clarify the effect of the aspect ratio, the Reynolds number and the relative height concerning with the rectangular plate on the characteristics of the laminar necklace vortices. A thin rectangular plate with a sharp edge standing vertically on a flat ground wall was selected as the basic bluff body and was mounted on the wall. The shape of the rectangular plate could be described by using only the aspect ratio  $w/h$  defined as a ratio of the plate width  $w$  to the plate height  $h$ . It may be expected that the characteristics of the necklace vortex system, such as the shape and locations of the separation line around the three-dimensional bluff body on the ground wall and the number and locations of the necklace vortices, strongly depend on the aspect ratio. Therefore, the aspect ratio was varied widely. The overall effect of the aspect ratio  $w/h$ , the relative height  $h/\delta$  defined as the ratio of a plate height  $h$  to the laminar boundary layer thickness  $\delta$  and the Reynolds number  $Re_h$  based on the plate height  $h$  and the nominal tip velocity  $U_h$ , on the characteristics of necklace vortices was investigated by using visualization techniques.

## 2. Experimental Apparatus and Procedure

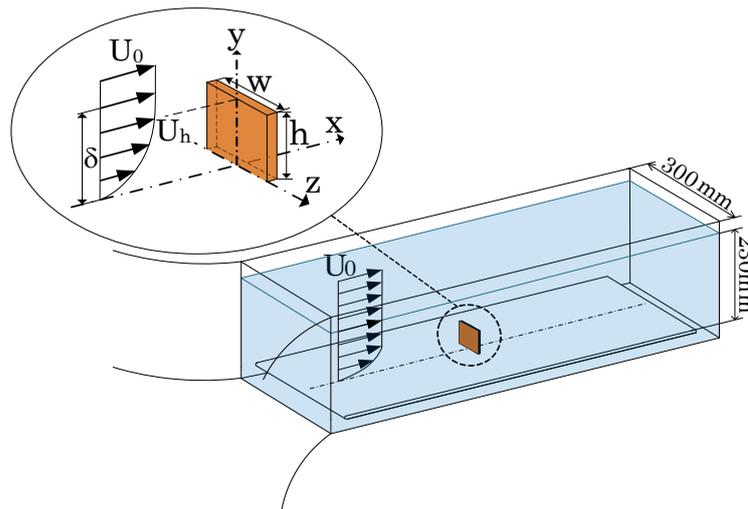
In this experiment, a water channel with an open test section with a width of 300 mm, a height of 300 mm and a length of 1100 mm was used, as shown in **Figure 1**. An inlet nozzle with a contraction area ratio 0.5 was installed upstream of the test section. A perforated plate, a honeycomb and four screens were installed upstream of the nozzle. The water depth in the test section was maintained at 280 mm. The transparent acrylic sidewalls and bottom of the test section allowed for the visualization of flow. A new laminar boundary layer was formed on a flat plate called the ground wall which was submerged in a position 50 mm from the water test section floor. To make the various necklace vortices, rectangular plates with various sizes were mounted vertically upon the ground wall. Details of the rectangular plate with a thickness of 1.5 mm are shown in **Table 1**. The maximum value of the rectangular plate width was restricted to 70 mm in consideration of the width of the test section. The blockage ratio, which was defined as the ratio of the projected area of each rectangular plate to the cross section of the water channel flow, was smaller than 4.4%.

The experimental Reynolds number  $Re_h (=U_h \cdot h/\nu)$  was defined based on the plate height  $h$  and the nominal tip velocity  $U_h$  at the same  $y$ -position as the upper tip of the plate in the approaching flow. The Reynolds number  $Re_h$  was in the range of 100 - 1800. However, because the unsteady vortex system was observed in the region of about  $Re_h > 500$  in the case of  $w/h = 3.0 - 7.0$ , the experiment was hardly carried out under these conditions, as shown in **Figure 5**. The nominal tip velocity  $U_h$  and the laminar boundary layer thickness  $\delta_{99}$  was estimated based on Blasius' solution by using free stream velocity  $U_0$  and the length from the leading edge of the ground wall to the rectangular plate position.  $U_0$  was determined by measuring 7 times dye moving speeds in the  $x$ -distance of 500 mm. The patterns of the necklace vortex system on a symmetrical plane were observed as the streak lines of the fluorescent dye injected from a small tracer tube set in various positions away from the ground wall, and in the separation line around the rectangular plate being observed as a curve line formed by dye injected from small holes of 0.8 mm diameter along the center line on the ground wall. The behavior of the dye was illuminated by a metal halide light source with slit width of 2 mm. The visualization images were taken by a film still camera with a shutter speed of (1/15) second, and were digitized into a BMP format by using a scanner with resolutions of 9 - 26 pixel/mm.

### 3. Experimental Results and Discussions

#### 3.1. Visualization and Classification of Necklace Vortex System

To clarify the characteristics of the necklace vortex system produced in front of the rectangular plate, the influence of the Reynolds number  $Re_h$ , relative height  $h/\delta$  and aspect ratio  $w/h$  upon the pattern of necklace vortices formed on the symmetrical plane was discussed. **Figure 2** exhibits the pattern of the necklace vortex system on the symmetrical plane upstream of a rectangular plate with the aspect ratios of  $w/h = 3.0, 5.0$  and  $7.0$ , respectively. The Reynolds number and relative height are kept at about  $Re_h = 400$  and  $h/\delta = 1.1$ . It is clear that the



**Figure 1.** Schematic diagram of experimental apparatus and coordinate system.

**Table 1.** Variation of blockage ratio for rectangular plates used in this experiment.

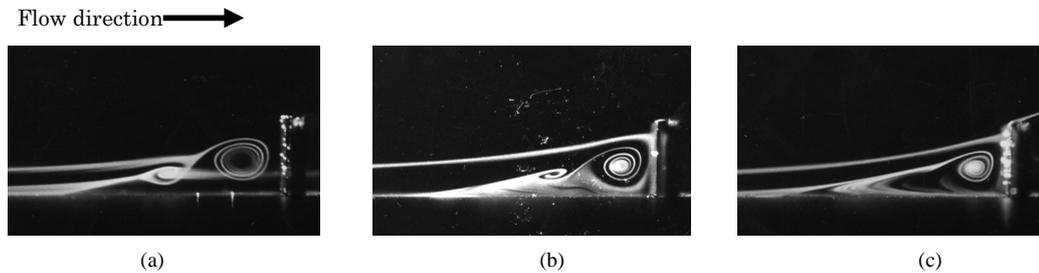
Aspect ratio $w/h$	Plate height $h$					
	10 mm	15 mm	20 mm	25 mm	35 mm	55 mm
0.5	0.07%	-	0.29%	-	0.89%	-
1.0	0.14%	-	0.58%	-	1.78%	4.38%
2.0	0.29%	0.65%	1.16%	1.81%	3.55%	-
3.0	0.43%	-	1.74%	-	-	-
5.0	0.72%	-	-	-	-	-
7.0	1.01%	-	-	-	-	-

different necklace vortex systems are formed depending on the conditions of  $Re_h$ ,  $w/h$  and  $h/\delta$ . Once the values of  $Re_h$ ,  $w/h$  and  $h/\delta$  are determined, the only steady and laminar necklace vortex is formed, and other vortex system is not formed on one's way to the only vortex system. The formed vortex system is not varied with the lapse of time. As shown in **Figure 2(a)**, in the case of  $w/h = 3.0$  there are two obvious clockwise vortices. From the viewpoint of flow pattern topology, it is predicted that two counter-clockwise vortices must exist in the triangular region that adjoins upstream of each clockwise vortex, therefore a small clockwise vortex must exist in front of these clockwise vortices, and another counter-clockwise vortex exists in the small corner between the plate and the ground wall. Here, the typical flow pattern of a 6-vortex system based on topology is shown in **Figure 3**. In this case, there exists three clockwise vortices called  $V_1$ ,  $V_2$  and  $V_3$ . Therefore, the flow pattern shown in **Figure 2(a)** is considered to be almost a 6-vortex system, but as the most upstream clockwise vortex  $V_3$  is extremely small and not visualized, then it may be a vortex pattern existing between the 6-vortex and the 4-vortex systems.

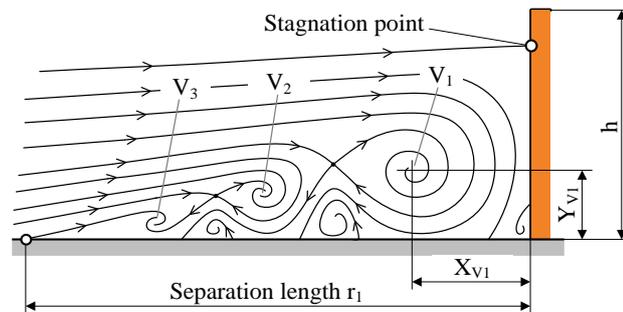
In the case of  $w/h = 5.0$ , the flow separating from the ground wall forms a second clockwise vortex  $V_2$  as shown in **Figure 2(b)**. This signifies that the system clearly becomes a 4-vortex system. In the case of  $w/h = 7.0$ , a large clockwise vortex clearly entrains the separating flow upstream of the plate as shown in **Figure 2(c)**. In this case, since a small counter-clockwise vortex must exist in the small corner between the plate and the ground wall, this flow pattern is called a 2-vortex system. The result above indicates that as the aspect ratio increases, the number of necklace vortices decreases.

**Figure 4** shows the necklace vortex system with aspect ratio  $w/h = 3.0$  and  $7.0$ . Though the relative height is also considered to be close to that of **Figure 2**, the Reynolds number is slightly larger than that of **Figure 2**. It was noticed that the 4-vortex or 6-vortex systems in **Figure 2(a)** has transformed into the 6-vortex system in **Figure 4(a)**, and the 2-vortex system in **Figure 2(c)** has become the 4-vortex system in **Figure 4(b)**. The comparison of **Figure 2** and **Figure 4** shows that even if the aspect ratio is the same, the vortex pattern has a tendency to change into larger vortex systems as the Reynolds number increases.

For the next step, the variation of the vortex system was systematically examined for a wide range of aspect ratios  $w/h$ , Reynolds numbers  $Re_h$  and relative heights  $h/\delta$ . **Figures 5(a)-(f)** show the classifications of the necklace vortex systems in terms of  $Re_h$  and  $h/\delta$  for aspect ratios  $w/h = 0.5, 1.0, 2.0, 3.0, 5.0$  and  $7.0$ , respectively. The necklace vortex pattern is classified into the 2-steady-vortex, 4-steady-vortex and 6-steady-vortex systems. The stationarity of the vortex systems were determined by the maintenance of the dye trajectory which exhibits the shape of the vortex pattern. The unsteady vortex pattern in the case of high Reynolds number and



**Figure 2.** Pattern of necklace vortex system on symmetrical plane in front of a rectangular plate: (a)  $w/h = 3.0$ ,  $Re_h = 410$ ; (b)  $w/h = 5.0$ ,  $Re_h = 420$ ; (c)  $w/h = 7.0$ ,  $Re_h = 440$ .



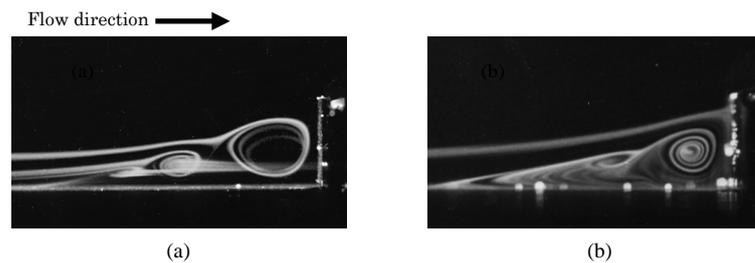
**Figure 3.** Typical flow pattern of the 6-vortex system ( $w/h = 2.0$ ,  $Re_h = 800$ ,  $h/\delta = 2.0$ ).

the boundary between the steady and unsteady vortex patterns is not handled in this study. By and large, it was noted that the necklace vortex system occurs depending on the Reynolds number  $Re_h$ , the relative height  $h/\delta$  and the aspect ratio  $w/h$ . The vortex system becomes larger for each  $w/h$  with increasing  $Re_h$  and  $h/\delta$ . In addition, it is understood that the effect of  $Re_h$  and  $h/\delta$  on the transitional boundary between the different vortex systems is approximately similar for each aspect ratio  $w/h$ . In the present experimental range, the 2-vortex system was not observed in the aspect ratio  $w/h = 2.0$  and  $3.0$ . The boundary between the steady and the unsteady regimes also shifts to small values of  $Re_h$  and  $h/\delta$  in the same aspect ratio condition. On the other hand, in the case of aspect ratio  $w/h \geq 3.0$ , the 2-vortex system is observed again in the region of low Reynolds number.

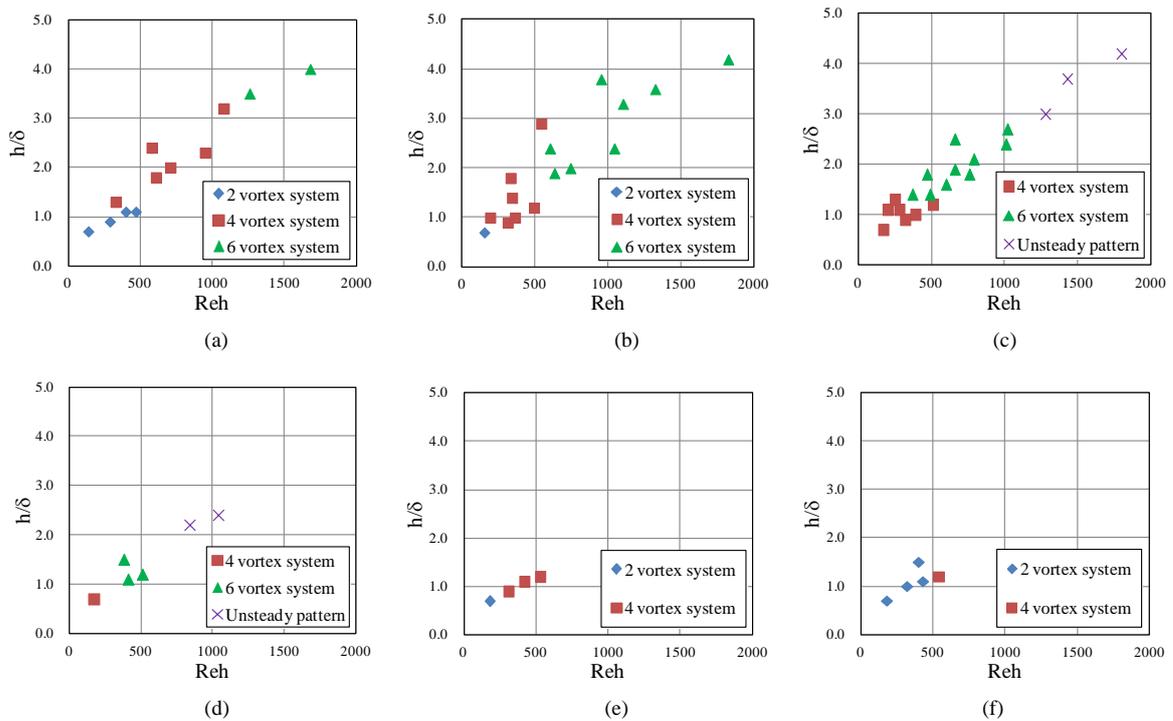
**Figure 6** summarizes the transitional boundaries of  $Re_h$  and  $h/\delta$  in the 2-vortex, 4-vortex and 6-vortex systems for each aspect ratio  $w/h$ . In the case of about  $w/h \leq 3.0$ , the transitional boundaries of  $Re_h$  and  $h/\delta$  shown in dotted lines decrease as  $w/h$  increases. In the case of about  $w/h \geq 3.0$ , their transitional boundaries increase as  $w/h$  increases. Therefore, it seems that the necklace vortex system forming in front of the rectangular plate changes the formation mechanism around the aspect ratio of about  $w/h = 3.0$ .

### 3.2. Variation of Necklace Vortex Position

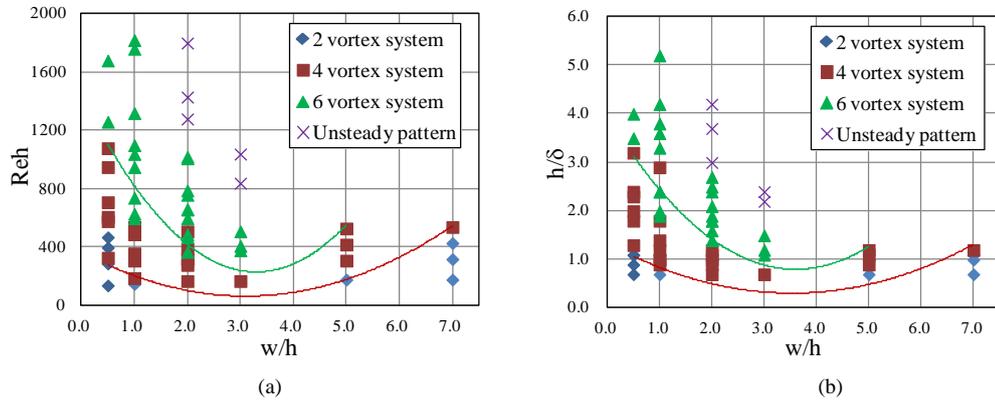
**Figure 7** shows  $X_{V1}/h$  for Reynolds number  $Re_h$  at each aspect ratio  $w/h$ .  $X_{V1}$  is the x-direction vortex position



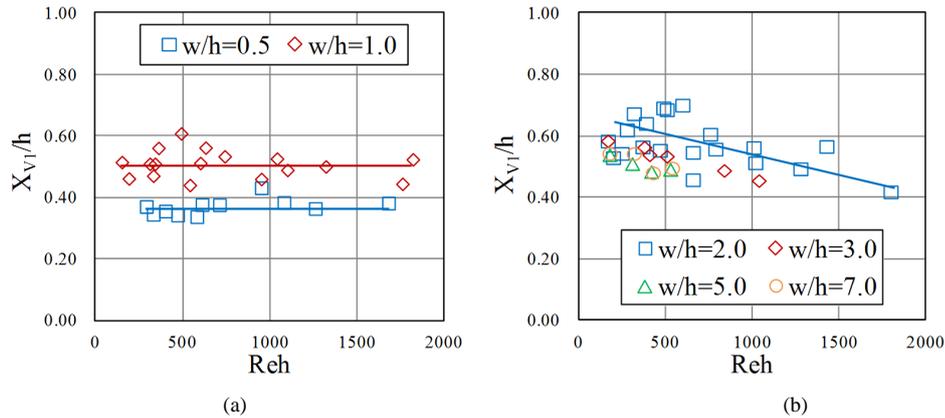
**Figure 4.** Pattern of necklace vortex system on symmetrical plane in front of a rectangular plate: (a)  $w/h = 3.0$ ,  $Re_h = 510$ ; (b)  $w/h = 7.0$ ,  $Re_h = 530$ .



**Figure 5.** Classification map of necklace vortex system for the combination of  $Re_h$  and  $h/\delta$  at each aspect ratio: (a)  $w/h = 0.5$ ; (b)  $w/h = 1.0$ ; (c)  $w/h = 2.0$ ; (d)  $w/h = 3.0$ ; (e)  $w/h = 5.0$ ; (f)  $w/h = 7.0$ .



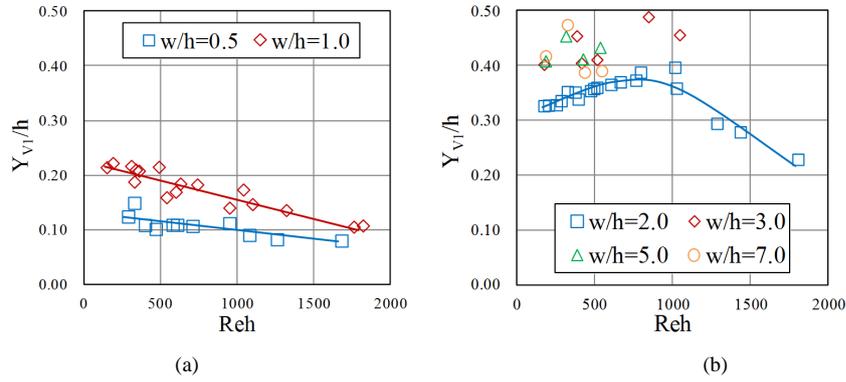
**Figure 6.** Classification map of necklace vortex system for various aspect ratios: (a) Reynolds number  $Re_h$  range; (b) Relative height  $h/\delta$  range.



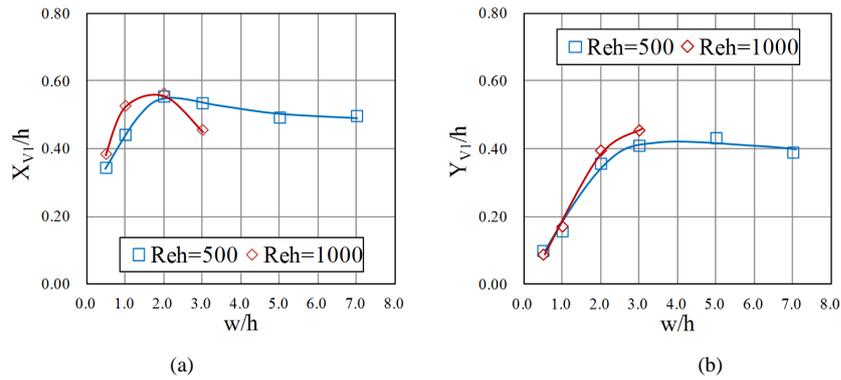
**Figure 7.** The x-direction vortex position length  $X_{V_1}$  for Reynolds number  $Re_h$ : (a)  $w/h = 0.5$  and  $1.0$ ; (b)  $w/h \geq 2.0$ .

length from the rectangular plate to the center position of main necklace vortex  $V_1$ . A center position of the vortices is defined as the center of the fluorescent dye that was rotating in the vortices on a symmetrical plane. In the case of  $w/h \leq 1.0$ ,  $X_{V_1}/h$  is constant over a wide range in terms of the Reynolds number  $Re_h$ , but gradually decreases as  $Re_h$  increases in the case of  $w/h \geq 2.0$ . In addition,  $X_{V_1}/h$  decreases with  $w/h$  decreasing under  $w/h < 2.0$ . The flow approaching the plate forms a stagnation point on the plate front surface. A down-reverse flow from the stagnation point pushes the necklace vortices upstream, and then causes the formation of the necklace vortex system. By decreasing  $w/h$ , the flow in the plate side direction becomes dominant, and the down-reverse flow becomes relatively weak. Therefore, it is thought that  $X_{V_1}/h$  is low value with  $w/h$  decreasing. On the other hand,  $X_{V_1}/h$  is slightly decreased in the case of  $w/h > 2.0$ , because the vortex tube is stretched in the plate width direction with increasing  $w/h$ . It seems to indicate that the formation mechanics of necklace vortices has gradually altered in the case of  $w/h > 2.0$ . The tendency that  $X_{V_1}/h$  shifts in the case of  $w/h = 2.0$  is clearly observed in **Figure 9(a)** which shows the variations of  $X_{V_1}/h$  at  $Re_h = 500$  and  $1000$ .

**Figure 8** shows  $Y_{V_1}/h$  for Reynolds number  $Re_h$  at each aspect ratio  $w/h$ .  $Y_{V_1}$  is the y-direction vortex position length from the ground wall to the center position of main necklace vortex  $V_1$ .  $Y_{V_1}/h$  has a tendency to alter the variation of  $Re_h$ . In the case of  $w/h = 0.5$  and  $1.0$ ,  $Y_{V_1}/h$  decreases as  $Re_h$  increases. On the other hand, in the case of  $w/h = 2.0$ ,  $Y_{V_1}/h$  increases and peaks at about  $Re_h = 800$ , and then decreases as  $Re_h$  increases. Further variation of  $Y_{V_1}/h$  seems to produce a similar tendency for the aspect ratios of  $w/h \geq 2.0$ . **Figure 9(b)** shows  $Y_{V_1}/h$  for the various  $w/h$  at  $Re_h = 500$  and  $1000$ .  $Y_{V_1}/h$  rapidly increases with increasing  $w/h$  in the case of  $w/h \leq 2.0$ , and becomes gradually constant in the case of about  $w/h \geq 2.0$ . These results show that the variation in the necklace vortex formation position is light for the Reynolds number and rapid for small aspect ratios as shown in **Figure 8** and **Figure 9**, and suggest that the transition of the vortex system hardly affects the formation position and scale of the main necklace vortex.



**Figure 8.** The y-direction vortex position length  $Y_{V1}$  for Reynolds number  $Re_h$ : (a)  $w/h = 0.5$  and  $1.0$ ; (b)  $w/h \geq 2.0$ .



**Figure 9.** Vortex position length for various aspect ratios at  $Re_h = 500$  and  $1000$ : (a)  $X_{V1}/h$ ; (b)  $Y_{V1}/h$ .

### 3.3. Variation of Separation Line

The spread of the necklace vortices region can be known by observing a separation line on the ground wall. **Figure 10** shows the top view of a separation line around the rectangular plate ( $w/h = 1.0$ ). The visualization dye was injected from a small hole just downstream of the separation point on the wall surface along  $x$ -axis. It moves once upstream of the hole, and surrounds the rectangular plate on the wall surface. In this study, the shape of the separation line is determined by separation lengths  $r_1$  and  $r_2$  from the coordinate origin to the separation point, measured as the outer edge of the dye in the  $x$  and  $z$ -directions, and the separation length  $r_3$  from the side edge of the plate to the separation point in the  $z$ -direction.

**Figure 11** shows the relative separation length  $r_1/h$  in the  $x$ -direction for Reynolds number  $Re_h$  in various aspect ratios  $w/h$ . In the case of  $w/h = 0.5$  and  $1.0$ ,  $r_1/h$  gradually increases with increasing  $Re_h$ , and becomes almost constant under  $Re_h \geq 1000$ . In the case of  $w/h = 2.0$   $r_1/h$  becomes large and has a gentle peak at about  $Re_h = 800$ , and then in the case of  $w/h \geq 2.0$  it seems to have the same tendency as the case of  $w/h = 2.0$ . This tendency of  $r_1/h$  is similar to that of  $Y_{V1}/h$  of decreasing as  $Re_h$  increases. The rapid increase of  $r_1/h$  in  $w/h = 2.0$  is due to the increase in the rate of the down-reverse flow as the plate width increases. **Figure 12** shows the relative separation length  $r_3/h$  in the  $z$ -direction for each  $Re_h$ . The separation length  $r_3/h$  increases with increasing  $Re_h$ , and becomes almost constant in the region of about  $Re_h \geq 1000$ . **Figure 13** shows the separation lengths  $r_1/h$  and  $r_3/h$  for each aspect ratio at  $Re_h = 500$  and  $1000$ . Both  $r_1/h$  and  $r_3/h$  show the same variation patterns that increases with increasing  $w/h$  in the case of  $w/h < 3.0$  and approach constant values in the case of  $w/h \geq 3.0$ . The value of  $r_1/h$  and  $r_3/h$  is almost equal in the case of  $w/h < 1.0$ , but  $r_3/h$  is always smaller than  $r_1/h$  in the case of  $w/h \geq 1.0$ . By and large, it is noted that variation of  $r_1/h$  and  $r_3/h$  for  $w/h$  is similar to that of  $X_{V1}/h$  and  $Y_{V1}/h$ .

It is considered that the separation length  $r_1$  and the main vortex position length  $X_{V1}$  are closely related. The relationship between  $r_1$  and  $X_{V1}$  was examined in terms of the Reynolds number  $Re_h$  for each aspect ratio as shown in **Figure 14**, and was examined in terms of the aspect ratio  $w/h$  in the case of  $Re_h = 500$  and  $1000$  as shown in **Figure 15**. The variation of  $X_{V1}/r_1$  for  $Re_h$  has the same tendency regardless of  $w/h$ . That is to say,

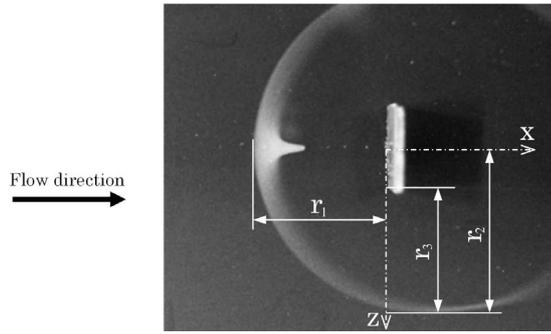


Figure 10. Top view of the separation line on the ground wall ( $w/h = 1.0$ ).

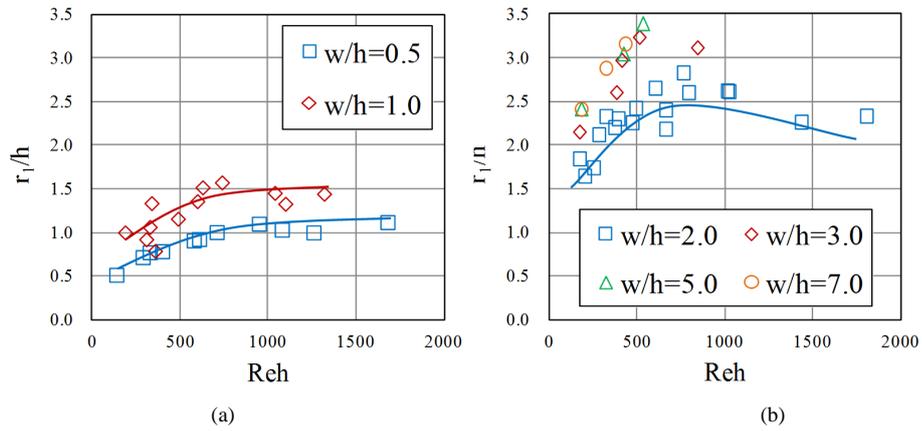


Figure 11. The x-direction separation length  $r_1$  for Reynolds number  $Re_h$ : (a)  $w/h = 0.5$  and  $1.0$ ; (b)  $w/h \geq 2.0$ .

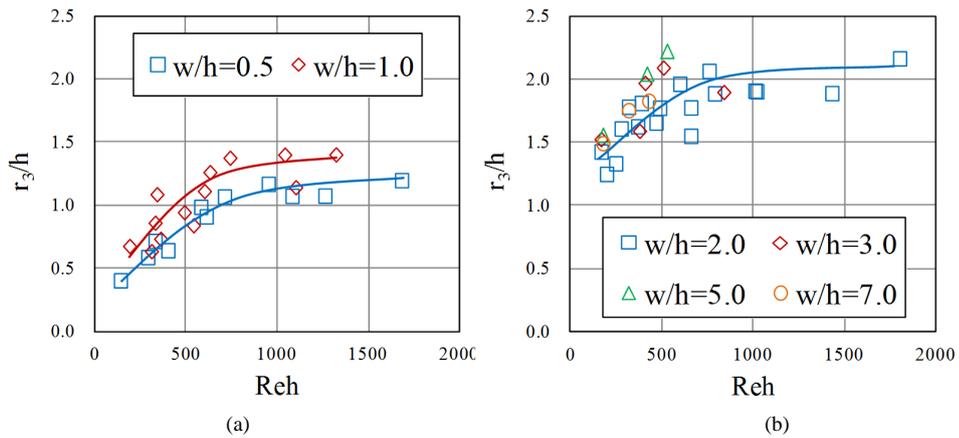


Figure 12. The z-direction separation length  $r_3$  for Reynolds number  $Re_h$ : (a)  $w/h = 0.5$  and  $1.0$ ; (b)  $w/h \geq 2.0$ .

$X_{v1}/r_1$  decreases slightly with increasing  $Re_h$  in the region of  $Re_h < 1000$  and becomes almost constant at about  $Re_h \geq 1000$ . Moreover,  $X_{v1}/r_1$  decreases slightly with increasing  $w/h$  in the case of about  $w/h < 3.0$  and approaches a constant value of about  $0.15$  in the case of  $w/h \geq 3.0$ .

#### 4. Conclusions

The characteristics of the necklace vortex system produced around a rectangular plate vertically standing on a ground wall forming a laminar boundary layer were studied by using a visualization method. As a result, the following insights were gained:

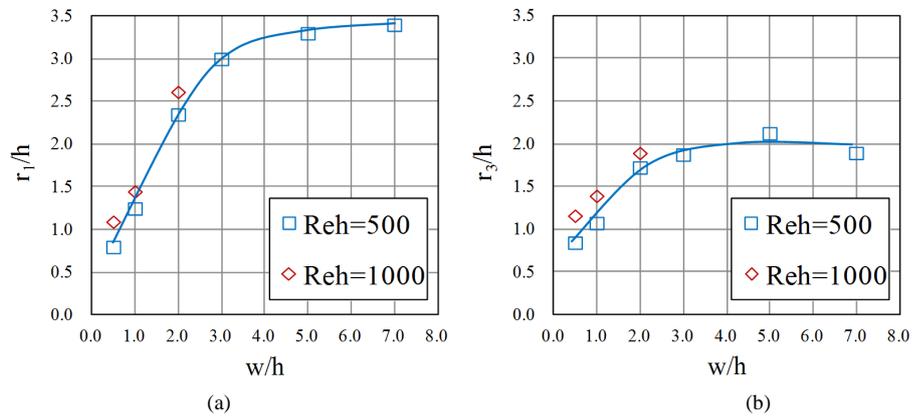


Figure 13. The x-direction separation length  $r_1$  for various aspect ratios at  $Re_h = 500$  and  $1000$ : (a)  $r_1/h$ ; (b)  $r_3/h$ .

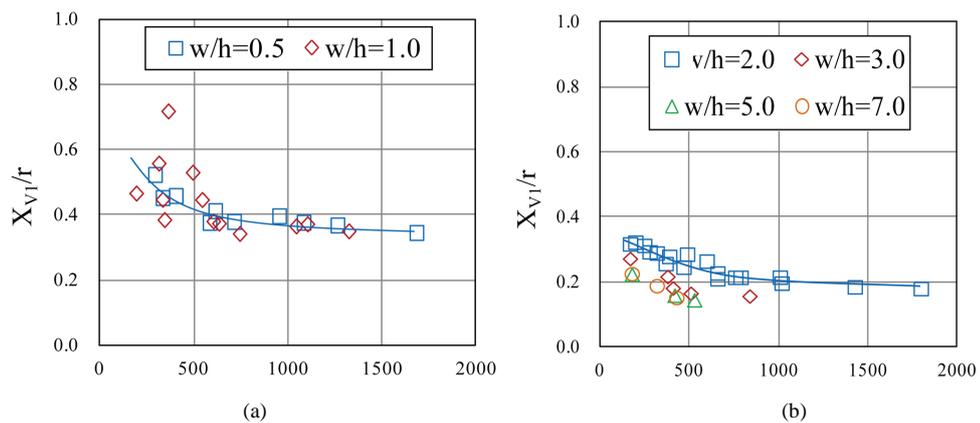


Figure 14. Relationship between the vortex position length and the separation length in x-direction for Reynolds number  $Re_h$ : (a)  $w/h = 0.5$  and  $1.0$ ; (b)  $w/h \geq 2.0$ .

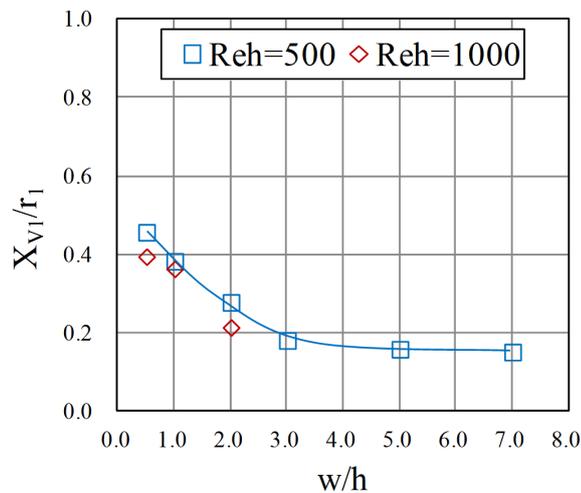


Figure 15. Relationship between the vortex position length and the separation length in x-direction for various aspect ratios at  $Re_h = 500$  and  $1000$ .

1) The necklace vortex pattern is classified into either 2-steady-vortex, 4-steady-vortex and 6-steady-vortex systems depending on the Reynolds number  $Re_h$ , the relative height  $h/\delta$  and the aspect ratio  $w/h$ , and the vortex

system is large at each  $w/h$  with increasing  $Re_h$  and  $h/\delta$ .

2) The transitional boundaries of  $Re_h$  and  $h/\delta$  forming the 2-vortex, 4-vortex and 6-vortex systems for each aspect ratio decrease in the case of  $w/h < 3.0$ , and increase in the case of  $w/h \geq 3.0$  as  $w/h$  increases.

3) The x-direction main vortex position length  $X_{v1}/h$  is almost constant in the case of  $w/h < 3.0$ , and decreases in the case of  $w/h \geq 3.0$  with increasing  $Re_h$ .

4) The x-direction separation length  $r_1/h$  increases in the case of  $w/h < 3.0$  as  $w/h$  increases, and becomes almost constant in the case of  $w/h \geq 3.0$ . Moreover,  $r_1/h$  increases in the case of  $Re_h < 1000$  as  $Re_h$  increases, and becomes almost constant in the case of  $Re_h \geq 1000$ .

5) The ratio  $X_{v1}/r_1$  of the main vortex position length to the separation length gradually approaches a constant value of 0.15 in the case of  $w/h \geq 3.0$  and  $Re_h \geq 1000$ .

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## References

- [1] Baker, C.J. (1979) The Laminar Horseshoe Vortex. *Journal of Fluid Mechanics*, **95**, 347-367. <http://dx.doi.org/10.1017/S0022112079001506>
- [2] Baker, C.J. (1980) The Turbulent Horseshoe Vortex. *Journal of Wind Engineering Industrial Aerodynamics*, **6**, 9-23. [http://dx.doi.org/10.1016/0167-6105\(80\)90018-5](http://dx.doi.org/10.1016/0167-6105(80)90018-5)
- [3] Baker, C.J. (1991) The Oscillation of Horseshoe Vortex Systems. *Journal of Fluids Engineering*, **113**, 489-495. <http://dx.doi.org/10.1115/1.2909523>
- [4] Thomas, A.S.W. (1987) The Unsteady Characteristics of Laminar Juncture Flow. *Physics of Fluids*, **30**, 283-285. <http://dx.doi.org/10.1063/1.866374>
- [5] Visbal, M.R. (1991) Structure of Laminar Juncture Flows. *AIAA Journal*, **29**, 1273-1281. <http://dx.doi.org/10.2514/3.10732>
- [6] Seal, C.V., Smith, C.R., Akin, O. and Rockwell, D. (1995) Quantitative Characteristics of a Laminar Necklace Vortex System at a Rectangular Block-Flat Plate Juncture. *Journal of Fluid Mechanics*, **286**, 117-135. <http://dx.doi.org/10.1017/S002211209500067X>
- [7] Seal, C.V., Smith, C.R. and Rockwell, D. (1997) Dynamics of Vorticity Distribution in Endwall Junctures. *AIAA Journal*, **35**, 1041-1047. <http://dx.doi.org/10.2514/2.192>
- [8] Lin, C., Ho, T.C. and Dey, S. (2008) Characteristics of Steady Horseshoe Vortex System near Junction of Square Cylinder and Base Plate. *Journal of Engineering Mechanics*, **134**, 184-197. [http://dx.doi.org/10.1061/\(ASCE\)0733-9399\(2008\)134:2\(184\)](http://dx.doi.org/10.1061/(ASCE)0733-9399(2008)134:2(184))
- [9] Matsuguchi, A., Kagawa, N., Tsuda, N. and Tsuruno, S. (1999) Study on Transition Process of Horse-Shoe Vortex. *Transaction of the Japan Society of Mechanical Engineers, Series B*, **65**, 28-35. (In Japanese) <http://dx.doi.org/10.1299/kikaib.65.28>
- [10] Lin, C., Chiu, P.H. and Shieh, S.J. (2002) Characteristics of Horseshoe Vortex System near a Vertical Plate-Base Plate Juncture. *Experimental Thermal and Fluid Science*, **27**, 25-46. [http://dx.doi.org/10.1016/S0894-1777\(02\)00215-7](http://dx.doi.org/10.1016/S0894-1777(02)00215-7)
- [11] Okamoto, S., Kobayashi, M., Kadono, T., Kagaya, H. and Shimane, A. (2005) Flow Past a Plate of Finite Width Placed Vertically on a Ground Plane. *Transaction of the Japan Society of Mechanical Engineers, Series B*, **71**, 2663-2670. (In Japanese) <http://dx.doi.org/10.1299/kikaib.71.2663>
- [12] Sakamoto, H., Moriya, M., Taniguchi, S. and Arie, M. (1981) Flow around a Normal Plate of Finite Width Immersed in Turbulent Boundary Layer: 1st Report, Pressure and Drag Acting on Plate. *Transaction of the Japan Society of Mechanical Engineers, Series B*, **48**, 1674-1682. (In Japanese) <http://dx.doi.org/10.1299/kikaib.48.1674>
- [13] Wei, Q.D., Chen, G. and Du, X.D. (2001) An Experimental Study on the Structure of Juncture Flows. *Journal of Visualization*, **3**, 341-348. <http://dx.doi.org/10.1007/BF03181728>
- [14] Wei, Q.D., Wang, J.M., Chen, G., Lu, Z.B. and Bi, W.T. (2008) Modification of Junction Flows by Altering the Section Shapes of the Cylinders. *Journal of Visualization*, **11**, 115-124. <http://dx.doi.org/10.1007/BF03181926>

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## Nomenclatures

$h$ :	Height of rectangular plate [m]
$h/\delta$ :	Relative height of rectangular plate
Reh:	Experimental Reynolds number
$r_1$ and $r_2$ :	x- and z-direction separation lengths from the origin to the separation points [m]
$r_3$ :	z-direction separation length from the rectangular plate side edge to the separation point [m]
$U_h$ :	Nominal tip velocity [m/s]
$U_0$ :	Free stream velocity [m/s]
$w$ :	Width of rectangular plate [m]
$w/h$ :	Aspect ratio of rectangular plate
(x, y, z):	Cartesian coordinate system
$X_{V_1}$ and $Y_{V_1}$ :	x-direction and y-direction lengths from the origin to the center position of main vortex $V_1$ [m]
$\delta$ :	Laminar boundary layer thickness(= $\delta_{99}$ ) [m]
$\nu$ :	Kinematic viscosity [ $m^2/s$ ]