

Energetic Balance for the Flow Induced by a Constantly Accelerating Plate in a Second Grade Fluid

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Abstract

Dissipation, power due to the shear stress at the wall and the boundary layer thickness corresponding to the unsteady flow of a second grade fluid, due to a constantly accelerating plate, are established in exact and approximate forms. The changing of the kinetic energy with time is also determined from the energetic balance. Exact expressions of the same entities for Newtonian fluids are recovered as limiting cases of general results.

Keywords: Second Grade Fluid, Dissipation, Energetic Balance, Power, Kinetic Energy, Constantly Accelerating Plate

1. Introduction

In recent years, the interest for flows of non-Newtonian fluids has considerably increased, while the energetic aspects regarding these motions are scarcely met in the literature. To the best of our knowledge, the first results of this kind seem to be those of Bühler and Zierep [1] concerning the Rayleigh-Stokes problem for Newtonian fluids. These results have been recently extended to second grade and Maxwell fluids by Zierep and Fetecau [2,3]. In these works, the authors also determine the boundary layer thickness and use it in order to obtain series solutions for the velocity field and the shear stress.

The aim of this note is to provide the energetic study for the unsteady flow of a second grade fluid, driven by the transversal motion of a constantly accelerating plate, and to compare the obtained results with those corresponding to the flow induced by a suddenly moved plate (Stokes' first problem). Of special interest is the energetic balance of the three terms: Changing of the kinetic energy with time, Dissipation and the Power due to the shear stress at the wall. The last term describes the input energy that is necessary to keep the medium running. A decisive question is whether these terms are larger or smaller than in the Newtonian case.

2. Statement of the Problem

The flows to be here considered have the velocity field

[1-3]

$$\mathbf{v} = \mathbf{v}(y, t) = u(y, t) \mathbf{i}, \quad (1)$$

where \mathbf{i} is the unit vector along the x-direction of the Cartesian coordinate system x , y and z . For such motions, the constraint of incompressibility is automatically satisfied while the governing equations, in the absence of body forces and a pressure gradient in the flow direction, are [2,4]

$$\begin{aligned} \partial_t u(y, t) &= (\nu + \alpha \partial_t) \partial_y^2 u(y, t), \\ \tau(y, t) &= (\mu + \alpha_1 \partial_t) \partial_y u(y, t), \end{aligned} \quad (2)$$

where μ is the dynamic viscosity, α_1 is a material constant, $\nu = \mu / \rho$ is the kinematic viscosity of the fluid (ρ being its constant density), $\alpha = \alpha_1 / \rho$ and $\tau = S_{xy}$ is the non-trivial shear stress.

The appropriate initial and boundary conditions, corresponding to the motion over an infinite constantly accelerating plate, are given by [4-6]

$$\begin{aligned} u(y, 0) &= 0, \quad y > 0; \quad u(0, t) = \\ At, \quad t \geq 0; \quad u(y, t), \quad \partial_y u(y, t) &\rightarrow 0 \text{ as } y \rightarrow \infty, \end{aligned} \quad (3)$$

where A is a constant.

The energetic balance for an infinite volume of rectangular cross-section with $x \in [0, \ell]$ and $z \in [0, 1]$, as it results from [1-3], is given by the identity

$$\frac{d}{dt} E_{kin} + L + \Phi = 0, \quad (4)$$

where E_{kin} is the kinetic energy and

$$L = -\ell u(0,t)\tau(0,t) = -\ell u_w(t)\tau_w(t), \Phi = \int_0^\infty \tau(y,t) \frac{\partial u(y,t)}{\partial y} dy, \quad (5)$$

are the power of the shear stress at the wall and the dissipation.

The boundary layer thickness δ , as it results from [7], is given by

$$\delta = \frac{1}{u(0,t)} \int_0^\infty u(y,t) dy \quad (6)$$

and represents the thickness of the fluid layer moved with the plate by friction. One measure of the boundary layer thickness is the distance from the wall where the velocity of the fluid differs by 1% from the external velocity.

3. Exact Results

Let us consider an incompressible second grade fluid at rest over an infinitely extended flat plate situated in the (x, z) -plane. After time $t = 0$ the plate is subjected to a transversal motion, in its plane, of velocity $u(0,t) = At$ in the x -direction. Owing to the shear, the fluid over the plate is gradually moved, its velocity being of the form (1). The governing equation is given by Equation (2) while the corresponding initial and boundary conditions are of the form (3).

The velocity field $u(y,t)$ and the adequate shear stress $\tau(y,t)$ corresponding to this problem, as it results from [8] (see Equations (3) and (4)), are given by (see also [5], Equation (5) with $V(t) = At$ or [6], Equation (7) for the velocity field only)

$$u(y,t) = At - \frac{2A}{\nu\pi} \int_0^\infty \left[1 - \exp\left(-\frac{\nu t \xi^2}{1 + \alpha \xi^2}\right) \right] \frac{\sin(y\xi)}{\xi^3} d\xi, \quad (7)$$

$$\tau(y,t) = -\frac{2\rho A}{\pi} \int_0^\infty \left[1 - \frac{1}{1 + \alpha \xi^2} \exp\left(-\frac{\nu t \xi^2}{1 + \alpha \xi^2}\right) \right] \frac{\cos(y\xi)}{\xi^2} d\xi. \quad (8)$$

Introducing (7) and (8) into (5) and (6), we find for L , Φ and δ the next exact expressions

$$L = -\frac{2\rho \ell A^2 t}{\pi} \int_0^\infty \left[1 - \frac{1}{1 + \alpha \xi^2} \exp\left(-\frac{\nu t \xi^2}{1 + \alpha \xi^2}\right) \right] \frac{1}{\xi^2} d\xi, \quad (9)$$

$$\Phi = \frac{4\rho \ell A^2}{\nu\pi^2} \int_0^\infty \left\{ \int_0^\infty \left[1 - \exp\left(-\frac{\nu t \xi^2}{1 + \alpha \xi^2}\right) \right] \frac{\cos(y\xi)}{\xi^2} d\xi \right. \\ \left. \times \int_0^\infty \left[1 - \frac{1}{1 + \alpha \xi^2} \exp\left(-\frac{\nu t \xi^2}{1 + \alpha \xi^2}\right) \right] \frac{\cos(y\xi)}{\xi^2} d\xi \right\} dy, \quad (10)$$

$$\delta = \int_0^\infty \left\{ 1 - \frac{2}{\nu\pi t} \int_0^\infty \left[1 - \exp\left(-\frac{\nu t \xi^2}{1 + \alpha \xi^2}\right) \right] \frac{\sin(y\xi)}{\xi^3} d\xi \right\} dy. \quad (11)$$

By making α_1 and thereby $\alpha \rightarrow 0$ into Equations (7)-(11), we attain to the similar expressions

$$u_N(y,t) = At - \frac{2A}{\nu\pi} \int_0^\infty \left(1 - e^{-\nu t \xi^2} \right) \frac{\sin(y\xi)}{\xi^3} d\xi, \\ \tau_N(y,t) = -\frac{2\rho A}{\pi} \int_0^\infty \left(1 - e^{-\nu t \xi^2} \right) \frac{\cos(y\xi)}{\xi^2} d\xi, \quad (12)$$

$$L_N = -\frac{2\rho \ell A^2 t}{\pi} \int_0^\infty \left(1 - e^{-\nu t \xi^2} \right) \frac{1}{\xi^2} d\xi, \\ \Phi_N = \frac{4\rho \ell A^2}{\nu\pi^2} \int_0^\infty \left[\int_0^\infty \left(1 - e^{-\nu t \xi^2} \right) \frac{\cos(y\xi)}{\xi^2} d\xi \right]^2 dy, \quad (13)$$

$$\delta_N = \int_0^\infty \left[1 - \frac{2}{\nu\pi t} \int_0^\infty \left(1 - e^{-\nu t \xi^2} \right) \frac{\sin(y\xi)}{\xi^3} d\xi \right] dy, \quad (14)$$

corresponding to a Newtonian fluid performing the same motion. The integrals in (12)-(14) can be evaluated to give the classical solutions

$$u_N(y,t) = 4At^2 \operatorname{Erfc}\left(\frac{y}{2\sqrt{\nu t}}\right), \tau_N(y,t) = -2\rho A\sqrt{\nu t} \operatorname{iErfc}\left(\frac{y}{2\sqrt{\nu t}}\right), \quad (15)$$

for velocity and shear stress and the simple expressions [9, Equations (13) and (14)]

$$L_N = -2\rho \ell A^2 t \sqrt{\frac{\nu t}{\pi}}, \Phi_N = \frac{8(\sqrt{2}-1)}{3} \rho \ell A^2 t \sqrt{\frac{\nu t}{\pi}}, \delta_N = \frac{4}{3} \sqrt{\frac{\nu t}{\pi}}, \\ \frac{d}{dt} E_{kin}^N = \frac{2(7-4\sqrt{2})}{3} \rho \ell A^2 t \sqrt{\frac{\nu t}{\pi}}, \quad (16)$$

for L_N , Φ_N , δ_N and dE_{kin}^N/dt . Into Equation (15)

$$i^n \operatorname{Erfc}(x) = \int_x^\infty i^{n-1} \operatorname{Erfc}(\xi) d\xi \text{ with } n =$$

$$0, 1, 2, \dots; i^0 \operatorname{Erfc}(x) = \operatorname{Erfc}(x),$$

are the integrals of the complementary error function $\operatorname{Erfc}(\cdot)$.

4. Asymptotic approximations for $(\alpha / vt) \ll 1$

Let us now consider the case when the material module α_1 is small enough, so that the dimensionless memory variable $\alpha / (vt)$ is much less than one. Simple computations show that the next approximations

$$\Phi = \Phi_N + \alpha \frac{4\rho l A^2}{v\pi^2} \int_0^\infty \int_0^\infty \frac{1 - e^{-vt\xi^2}}{\xi^2} \cos(y\xi) d\xi \int_0^\infty (2vt\xi^2 - 1) e^{-vt\xi^2} \cos(y\xi) d\xi dy + O(\alpha^2), \quad (20)$$

$$\delta = \delta_N + \alpha \frac{2}{\pi} \int_0^\infty \int_0^\infty \xi e^{-vt\xi^2} \sin(y\xi) d\xi dy + O(\alpha^2). \quad (21)$$

Evaluating all integrals, as well as in [2] and [3], we attain to the following approximate expressions

$$L = L_N \left\{ 1 + \frac{1}{4} \frac{\alpha}{vt} + O\left[\left(\frac{\alpha}{vt}\right)^2\right] \right\},$$

$$\delta = \delta_N \left\{ 1 + \frac{3}{4} \frac{\alpha}{vt} + O\left[\left(\frac{\alpha}{vt}\right)^2\right] \right\}. \quad (22)$$

$$\Phi = \Phi_N \left\{ 1 + \frac{3(2-\sqrt{2})}{8} \frac{\alpha}{vt} + O\left[\left(\frac{\alpha}{vt}\right)^2\right] \right\}, \quad (23)$$

Equations (4), (16), (22) and (23), also imply

$$\frac{d}{dt} E_{kin} = \frac{d}{dt} E_{kin}^N \left\{ 1 + \frac{9(5-2\sqrt{2})}{68} \frac{\alpha}{vt} + O\left[\left(\frac{\alpha}{vt}\right)^2\right] \right\}. \quad (24)$$

The results corresponding to L , Φ and dE_{kin} / dt are in accordance with those obtained in [2] for the motion due to a suddenly moved plate. More exactly, into both motions, L , Φ and dE_{kin} / dt increase for second grade fluids in comparison with Newtonian fluids. On the other hand the boundary layer thickness δ increases for the present motion and decreases for the first problem of Stokes.

5. Analogy to the Teipel Series Expansion

As early as 1981, Teipel [7] studied the first problem of Stokes for second grade fluids and provided a series so-

$$\exp\left(-\frac{vt\xi^2}{1+\alpha\xi^2}\right) = e^{-vt\xi^2} \exp\left(\frac{\alpha vt\xi^4}{1+\alpha\xi^2}\right) = e^{-vt\xi^2} [1 + \alpha vt\xi^4 + O(\alpha^2)], \quad (17)$$

$$\frac{1}{1+\alpha\xi^2} \exp\left(-\frac{vt\xi^2}{1+\alpha\xi^2}\right) = e^{-vt\xi^2} [1 + \alpha\xi^2(vt\xi^2 - 1) + O(\alpha^2)], \quad (18)$$

are valid for each ξ and t greater zero.

Introducing Equations (17) and (18) into (9)-(11) and having in mind Equation (16), we can easily prove that

$$L = L_N + \alpha \frac{2\rho l A^2 t}{\pi} \int_0^\infty (vt\xi^2 - 1) e^{-vt\xi^2} d\xi + O(\alpha^2), \quad (19)$$

lution. This series has been determined until an arbitrary constant, although all initial and boundary conditions have been fulfilled. In order to determine that constant, Teipel used a new initial condition, namely $\delta(0) = 0$. Later, in order to determine this constant, Erdogan and Imrak [10] used the condition that the volume flux is zero at time $t = 0$.

Following the Teipel's idea [7], and having in mind Equation (3), we are looking for a velocity field of the form

$$u(y, t) = At \left\{ f_0(\eta) + \frac{\alpha}{vt} f_1(\eta) + O\left[\left(\frac{\alpha}{vt}\right)^2\right] \right\}; \quad \eta = \frac{y}{2\sqrt{vt}}, \quad (25)$$

for the flow induced by a constantly accelerating plate in a second grade fluid. Introducing (25) into the governing Equation (2) and identifying the coefficients of the same powers, we find that

$$f_0''(\eta) + 2\eta f_0'(\eta) - 4f_0(\eta) = 0 \text{ and } f_1''(\eta) + 2\eta f_1'(\eta) = \frac{\eta}{2} f_0'''(\eta). \quad (26)$$

The appropriate boundary conditions being

$$f_0(0) = 1, f_0(\infty) = 0 \text{ and } f_1(0) = f_1(\infty) = 0, \quad (27)$$

it results that, in contrast with the results of Teipel, the functions $f_0(\cdot)$ and $f_1(\cdot)$ are completely determined and have the forms

$$f_0(\eta) = 4i^2 \operatorname{Erfc}(\eta), \quad f_1(\eta) = \frac{\eta}{\sqrt{\pi}} e^{-\eta^2} \quad (28)$$

Consequently, an approximate solution for the velocity field is

$$u(y,t) = 4At^2 \operatorname{Erfc}\left(\frac{y}{2\sqrt{vt}}\right) + \frac{\alpha}{vt} \frac{Ay}{2} \sqrt{\frac{t}{v\pi}} \exp\left(-\frac{y^2}{4vt}\right) + O\left[\left(\frac{\alpha}{vt}\right)^2\right]. \tag{29}$$

Introducing (29) into (2) and keeping the terms of the first order into $\alpha/(vt)$, we get for the shear stress the approximate expression

$$\tau(y,t) = -2\rho A\sqrt{vt} \operatorname{Erfc}\left(\frac{y}{2\sqrt{vt}}\right) - \frac{\alpha}{vt} \frac{\rho A}{2} \sqrt{\frac{vt}{\pi}} \left(1 + \frac{y^2}{2vt}\right) \exp\left(-\frac{y^2}{4vt}\right) + O\left[\left(\frac{\alpha}{vt}\right)^2\right]. \tag{30}$$

6. Conclusions

In this note, the power due to the shear stress at the wall, dissipation and the boundary layer thickness corresponding to the unsteady flow of a second grade fluid induced by a constantly accelerating plate, have been determined in exact and approximate forms. Their exact expressions corresponding to Newtonian fluids, performing the same motion, are recovered as limiting cases of general results. As a consequence, the changing of the kinetic energy with time is obtained from the energetic balance. Finally, series solutions are established both for the velocity field $u(y,t)$ and the shear stress $\tau(y,t)$.

In conclusion, from Equations (22)-(24) it clearly results that L , Φ , δ and dE_{kin}/dt increase for second grade fluids in comparison with Newtonian fluids. Furthermore, the results corresponding to L , Φ and dE_{kin}/dt are in accordance with those obtained in [2] for the motion due to a suddenly moved plate. As regards the boundary layer thickness, the present result is opposite to that from [2], where δ decreases for second grade fluids in comparison with the Newtonian fluids.

Finally, in order to reveal some relevant physical aspects of the obtained results, the diagrams of the velocity $u(y,t)$ and the shear stress $\tau(y,t)$ against y are presented for different values of the time t . **Figures 1 and 2** clearly show that the approximate solutions (29) and (30) give a very good approximation for the exact solutions (7) and (8). The variations of the power due to the shear stress at the wall $L(t)$ and the boundary layer thickness $\delta(t)$ are also depicted in **Figures 3 and 4** for different values of the material constant α . These graphs show that $L(t)$ and $\delta(t)$ are increasing functions with respect to t and decreasing ones with regard to α . Furthermore, as expected, for $\alpha \rightarrow 0$ they are going to L_N and δ_N corresponding to Newtonian fluids.

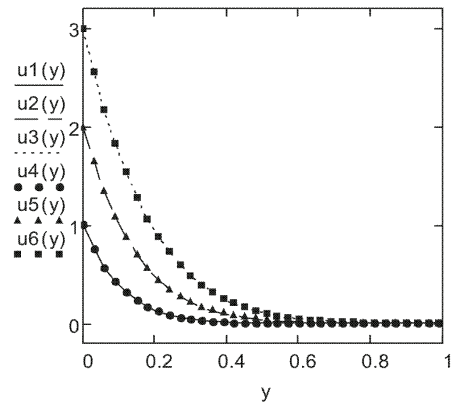


Figure 1. Comparison between the exact solution (7)—curves $u1(y)$, $u2(y)$, $u3(y)$ and the asymptotic solution (29)—curves $u3(y)$, $u4(y)$, $u5(y)$, for $A = 0.2$, $\nu = 0.003$, $\rho = 972$, $\alpha = 0.002$ and different values of t .

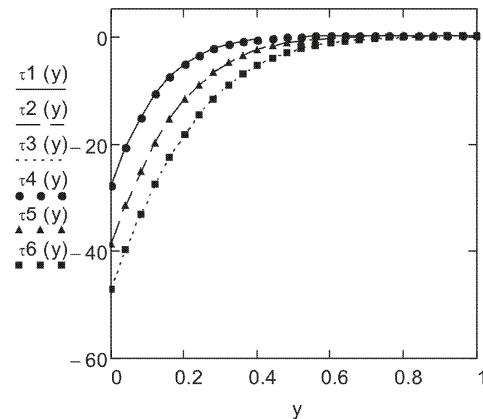


Figure 2. Comparison between the exact solution (8)—curves $\tau 1(y)$, $\tau 2(y)$, $\tau 4(y)$ and the asymptotic solution (29)—curves $\tau 3(y)$, $\tau 4(y)$, $\tau 5(y)$, for $A = 0.2$, $\nu = 0.003$, $\rho = 972$, $\alpha = 0.002$ and different values of t .

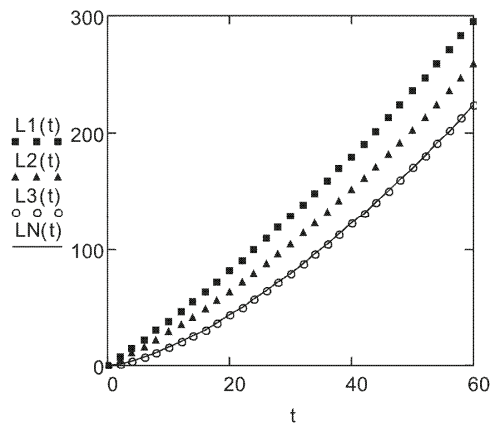


Figure 3. Power of the shear at the wall $L(t)$, corresponding to Newtonian fluid—curve $LN(t)$ and second grade fluid—curves $L1(t)$, $L2(t)$, $L3(t)$, for $A = 0.2$, $\nu = 0.003$, $\rho = 972$, and different values of α .

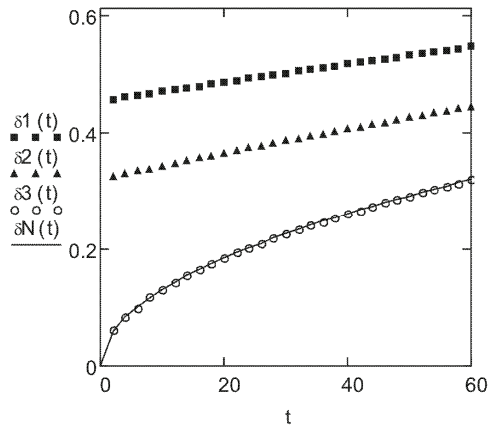


Figure 4. The boundary layer thickness $\delta(t)$, corresponding to Newtonian fluid—curve $\delta N(t)$ and second grade fluid—curves $\delta 1(t)$, $\delta 2(t)$, $\delta 3(t)$, for $\nu = 0.003$ and different values of α .

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