

# Cold-state Experimental Study on Flow Characteristics of Multi-nozzle in Natural Gas Reburning Burner

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## ABSTRACT

Based on the prototypes of a 130 t/h boiler, constant proportional cold-state test bench is established, flow characteristics of multi-nozzle in natural gas reburning burner and its influence on the covering effect for the upflow in the furnace are researched. Numerical simulations of this process are also made with standard  $k-\varepsilon$  turbulence model. The results show that air flow fullness in furnace is better in the case of the reburning zone with 8 nozzles compared to 4 nozzles and also coverage effect of the reburning flow for the updraft gas in the furnace is better. In the condition each nozzle airflow velocity is constant, the effect of reburning flow on coverage of up-secondary air is best when the incident angle for four corners is  $14.17^\circ$ , while Center of the furnace wall is  $84.57^\circ$ . And while the best incident angle is invariable, the effect of reburning flow on coverage of up-secondary air is best when the speed of reburning gas in the corners of furnace is 51 m/s, the same to the center of the furnace wall's.

**Keywords:** Reburning Burner; Multi-nozzle; Cold-state Flow Field; Numerical Simulation; Covering Effect

## 1. Introduction

Natural gas reburning technology is a promising method for denitrification [1]. In recent years, study on mechanism of reduce NO<sub>x</sub> with natural gas reburning in many countries found that compounds containing nitrogen HCN in reburning fuel has important influence on the reburning process. Patry and Engel [2] think that the product of reactions for nitrogen oxide and methane is HCN, water and hydrogen after study, also found that about 70% of the NO<sub>x</sub> has been transformed in a short time. One of the key technologies is the mixing properties of reburning flow and flue gas in the furnace [3]. So, study on reburning gas flow field is important for investigation on gas mixing characteristics in furnace and its effect on the reduction of NO<sub>x</sub>.

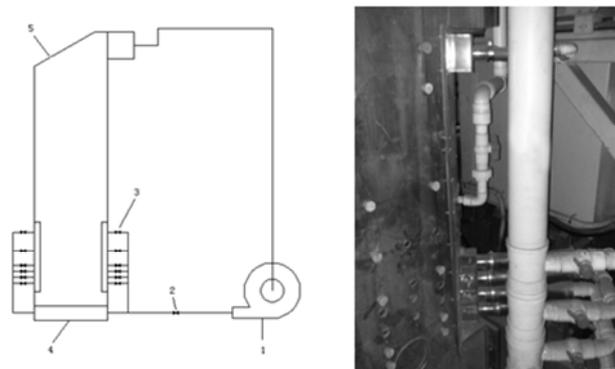
In this paper, compared with only 4 reburning nozzles in four corners, another 4 reburning nozzles is added in the center of the furnace wall in order to improve air flow fullness inside the furnace, achieving the purpose of effective reduction NO<sub>x</sub>. Similarity simulation principle is applied with velocity characterization of concentration and the two phase flow processing for the single phase fluid. The influence of reburning nozzle numbers, incident angle and airflow speed on the flow field inside the furnace will be studied with the method of numerical simulation and cold-state test in order to optimize the natural gas reburning technology and provide reference

for engineering application.

## 2. Introduction of Cold-state Experiment

### 2.1. Experiment Device

Cold-state test bench is built in the proportion of 1:5 according to the prototype of a 130 t/h boiler which consists of three parts: boiler furnace, the reburning burner and air distribution piping, and the bench system diagram is shown in **Figure 1**. The whole furnace is made of organic glass, the burner components is placed bottom-up in turn: lower primary air nozzle, lower secondary air nozzle, upper primary air nozzle, upper secondary air



1. Fan; 2. Valve; 3. Burner; 4. Air Chamber; 5. Furnace.

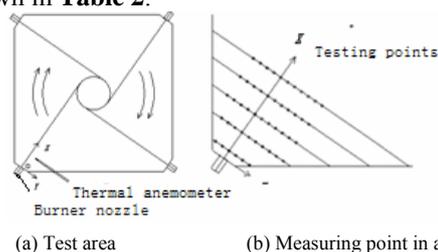
**Figure 1. Cold-state test bench system.**

nozzle, reburning nozzle and OFA nozzle. Working medium is air while the measuring instrument is thermal anemometer in the experiment. Test area and measuring point distribution in a corner of the furnace is shown in **Figure 2**: take X as the nozzle jet axis with the origin at the center of the nozzle, while Y is the horizontal axis along the nozzle, O point is the center of the nozzle.

### 2.2. The Experimental Parameters

According to similarity theory, cold modeling calculation is made based on operation parameters of the boiler, outlet air velocity of reburning nozzle in the experiment can be got. Experimental parameters and the actual operation parameters is shown in **Table 1**.

According to the research content, 6 kinds of operating modes are arranged in the experiment, which parameters are shown in **Table 2**.



**Figure 2. Test area and the measuring points in corner.**

**Table 1. The parameters of the boiler and experiment.**

Parameters	The boiler operation (m·s-1)	The experiment (m·s-1)
Primary air velocity	22	11.4
Secondary air velocity	50	22.6
Reburning air velocity in corners	100	51
	80	40.8
Reburning air velocity in the center of the furnace wall	100	51
	120	61.2
Actual velocity of OFA	64.2	29

**Table 2. Reburning nozzles parameters of 1 ~ 6 operating modes in the test.**

item	1	2	3	4	5	6
incident angle in corners $\alpha(^{\circ})$	41.17	41.17	41.17	41.17	41.17	41.17
air velocity in corners $v/(m \cdot s^{-1})$	51	51	51	51	51	51
Incident angle in the furnace wall $\beta(^{\circ})$	/	90°	84.57	74.57	84.57	84.57
air velocity in the furnace wall $v/(m \cdot s^{-1})$	/	51	51	51	40.8	61.2

### 3. Theory of Experiment and Numerical Simulation

#### 3.1. The Experiment Principle

According to the theory of similarity modeling, cold-state experiment must adhere to the following principles [4]: (1) geometric similarity between the model and the actual object is needed. (2) Under the corresponding conditions, air movement of model and the actual object must be in the automatic modeling area. (3) The momentum ratio each shared airflow of model keeps in equal with the actual objects.

Due to primary air and secondary air momentum ratio of the model and the actual object is equal [5, 6], as follows:

$$\frac{m_{1M}\omega_{1M}}{m_{2M}\omega_{2M}} = \frac{m_{1O}\omega_{1O} + m_P\omega_P}{m_{1O}\omega_{1O}(1+k\mu)} = \frac{m_{2O}\omega_{2O}}{m_{2O}\omega_{2O}} \quad (1)$$

In the formula, M-the model; O-the actual object; 1-the primary air; 2- secondary air; m-the mass flow rate;  $\omega$  - average velocity of the flowing nozzles; U -the pulverized coal concentration of the primary air(0.36), kg/kg; k- 0. 8 (consider different coefficient of pulverized coal flow rate and wind speed).

Secondary air velocity of the model can be calculated according to the euler number is equal between the model and the actual object.

$$\omega_{2M} = \omega_{2O} \sqrt{\frac{\rho_{2O}}{\rho_{2M}} * \frac{\Delta P_{2M}}{\Delta P_{2O}}} \quad (2)$$

In the formula,  $\Delta P_{2M}/\Delta P_{2O}$  - flow resistance of model and the actual object, 2;  $\rho$  – density of flow.

Take the temperature of each nozzle working medium as equal, the flowing formula can be obtained using formula (1) and (2):

$$\frac{\omega_{1M}}{\omega_{2M}} = \frac{\omega_{1O}}{\omega_{2O}} \sqrt{\frac{\rho_{2M}}{\rho_{1M}} \cdot \frac{\rho_{1O}(1+k\mu)}{\rho_{2O}} \cdot \frac{f_{2M}}{f_{1M}} \cdot \frac{f_{1O}}{f_{2O}}} \quad (3)$$

Each nozzle parameter of the cold model is obtained according to the operation parameters of the boiler.

#### 3.2. Mathematical Model of Numerical Simulation

Gas flow in the furnace is considered as the three-dimensional turbulent flow, standard turbulence model has good adaptability with large numbers of studies. The gas phase turbulent flow control equations can be obtained under the three-dimensional rectangular coordinate system, according to the N- S equation, the general form as follows: [7]

$$\frac{\partial}{\partial x}(\rho U \phi) + \frac{\partial}{\partial y}(\rho V \phi) + \frac{\partial}{\partial z}(\rho W \phi) = \frac{\partial}{\partial x} \left[ \Gamma_{\phi} \frac{\partial \phi}{\partial x} \right] + \frac{\partial}{\partial y} \left[ \Gamma_{\phi} \frac{\partial \phi}{\partial y} \right] + \frac{\partial}{\partial z} \left[ \Gamma_{\phi} \frac{\partial \phi}{\partial z} \right] + S_{\phi} \quad (4)$$

Inlet boundary condition: the primary air  $V_1 - 11.4$  m/s, the secondary air  $V_2 - 22.6$  m/s, Reburning air  $V_3 - 51$  m/s, OFA  $V_4 - 29$  m/s;

Outlet boundary conditions: the negative pressure of export of furnace(- 50 )Pa;

Wall boundary conditions: no velocity slip and no quality penetration, the border of the turbulent kinetic

energy-  $k = \frac{3}{2} \left( I u' \right)^2$  Turbulent kinetic energy dissipation

rate-  $\varepsilon = 0.09^{3/4} \frac{k^{3/2}}{l}$ , Turbulence intensity is  $I = 0.16 Re^{-1/8}$ ,

turbulence length is  $l = 0.007L$ .

The SIMPLE algorithm is adopted for the numerical simulation in the paper, using the algorithm to iterative calculation the equations of each variable [8]. The method of “speculation – revised” is mainly used in SIMPLE algorithm, computing pressure field on the basis of staggered grid in order to achieve the purpose of solving the momentum equations.

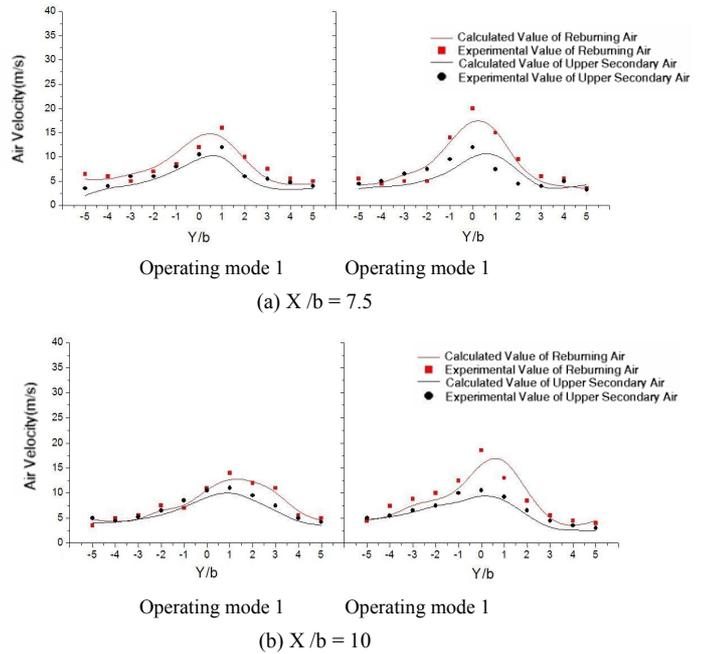
### 4. Analysis of Results

Under the conditions of 1~6 operating modes, air speed of 5 set of dimensionless distance ( $X/b=2.5, 7.5, 5, 10, b$  is 2 cm) is measured, which are distributed On the cross section of the reburning nozzle and the upper secondary air nozzle. The measurement results are compared with the results of numerical simulation, flow field characteristics of the furnace center ( $X/b = 7.5, 10$ ) is mainly analysed in this paper.

#### 4.1. The Influence of Reburning Nozzle Number on Covering Effect in Reburn Zone

Numerous of studies have shown that There are similarities between the distribution of gas concentration profile and the distribution of velocity profile in the furnace along the horizontal cross section of a furnace [9], so the speed distribution will represent the concentration distribution in this experiment [10].

As shown in **Figure 3**, in the operating mode 1 and 2, gas flow injection into the furnace with high speed. The direction of reburning airflow is coincide with the central axis of up secondary air roughly due to the large wind speed and airflow strong rigidity in the nozzle front end ( $X /, 2.5 b = 1, 5$ ). Reburning air velocity is higher than the upper secondary air in range of  $- 2 < y/b < 2$ , so up-drafts at the bottom of the furnace can be effectively covered by reburning airflow.



**Figure 3. Velocity distribution of reburning air in four corners and upper secondary air for operating mode 1,2.**

As shown in **Figure 3(a)**, reburning Air flow to the boiler furnace wall side deflected under the Impact of rotating airflow in furnace due to reburning air rate attenuation in mode 1 near the furnace center ( $X/b = 7.5$ ), the maximum its speed is near the  $Y/b = 1$ . Air deflection ratio is smaller compared to mode 2. as shown in **Figure 3(b)**, maximum speed of reburning flow under the mode 1 is 13.1 m/s, which cannot be significantly higher than the secondary air speed, so its coverage effect is poorer., under the mode 2, because 4 reburning nozzles have been added in the center of the furnace wall, enough momentum of the reburning air in the four horns is made to be shot into the rotating airflow of furnace center through the surrounding air entrainment effect, and coverage effect for the secondary air is higher.

#### 4.2. The Influence of the Incident Angle in Side Walls on Covering Effect in Reburn Zone

For coal-fired boiler, tangential diameter is an important characteristic of air flow in the furnace [8], the ideal flow state inside the furnace is forming right circular rotating in the center of the furnace flame. [11, 12]

Keep the velocity of reburning air and the upper secondary air unchanged, gradually adjusted the incident angle in the center of furnace wall, from  $90^\circ$  to  $74.57^\circ$ . As is shown in **Figure 4(a)**, under the mode 2,3 and 4, with the reduce of the incident angle, the maximum velocity in the center of reborn gas flow occur in  $y/b = 0.5, 1, 1.5$  or so, far away from the nozzle position ( $x/b = 7.5$ ). The phenomenon shows that due to defend incident

angle, the gas mixing intensity of the four corners and the furnace wall is different, which causes the difference of the four horns side gas flow deflection slope again, finally form the different diameter of tangential under the different incident angles.

Maximum speed of reburning gas flow by the center points occur in  $y/b = 1.5$  in operating mode 4, then gas flow deflection slope. The largest in the furnace form bigger tangential, because tangential diameter is too large, the gas flow is easy to stick on the wall, which causes the water wall slugging in the actual operation.

According to the **Figure 4(b)**, reburning air velocity is higher than the upper secondary air speed in  $-2 < y/b < 2$  range, and there is good coincidence on the center axis between reburning air and secondary air in mode 2, 3. Maximum reburning air speed is smaller at the  $x/b = 10$  in mode 3 compared with mode 2, and larger deflection. It means mode 3 form larger reactions in the furnace tangential, and its covering effect is better.

### 4.3. The Influence of Flow Velocity in Side Walls on Covering Effect in Reburn Zone

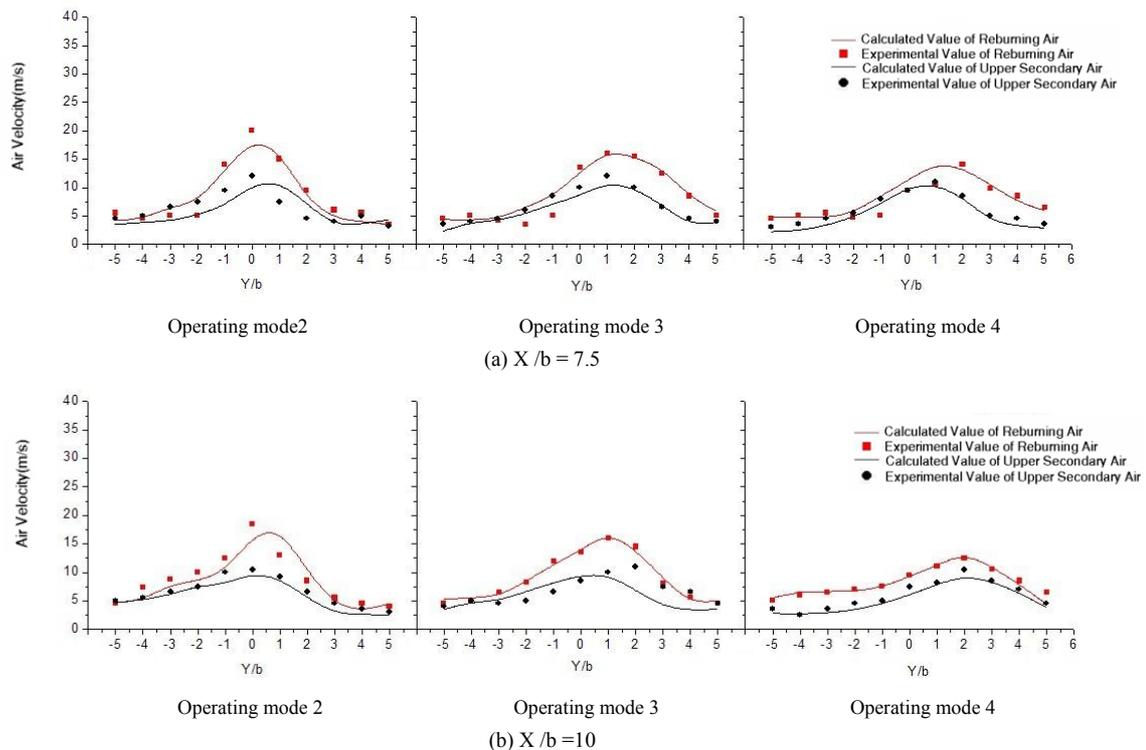
By studying the influence of different incident angles on the gas flow in furnace, reburning air has good cover effect on the upper secondary air when the incident angle of the four horns  $s$  is  $41.17^\circ$  and the incident angle in the center of side wall is  $84.57^\circ$ . The results are as follows when change the velocity of nozzle inlet with the inci-

dent, angle keep the same as above.

As is shown in **Figure 5**, with the high-speed and Strong rigidity of reburning air at  $X/b = 1, 5$ , calculated value and the simulation values coincide well under the mode 5. In the center of the furnace ( $x/b = 7.5, 10$ ), speed of reburning air can not be completely obvious higher than the second blast velocity. In actual operation,  $NO_x$  content is relatively high in the heat of the furnace flame, so if the reburning air in the region cannot be completely surrounded rotating airflow in furnace, generation efficiency will be poor.

In the operating mode 3, the highest speed central axis of the nozzle in corners and the nozzle in side wall roughly Keep consistent at the nozzle outlet ( $x/b = 1, 5, 2.5$ ). And in  $-2 < y/b < 2$  range, velocity in four corners is greater than the upper secondary air, due to the deflection is far stronger than which in mode 5 and air entrainment effect of up flow in the center of the furnace, the reburning air can be mixed with rotating airflow in the furnace, which has a better gentrification efficiency.

In the operating mode 6, the area of high-speed reborn gas cover on the secondary air is almost at low speed area. In the real operation, a lot of  $NO_x$  pollutants produced in the region of high-speed flue gas in the furnace, so in mode 6, reburning air and rotating air in furnace cannot happen strong disturbance and hybrid, which led to a decline in gentrification efficiency



**Figure 4. Velocity distribution of reburning air in four corners and upper secondary air for operating mode 2,3,4.**

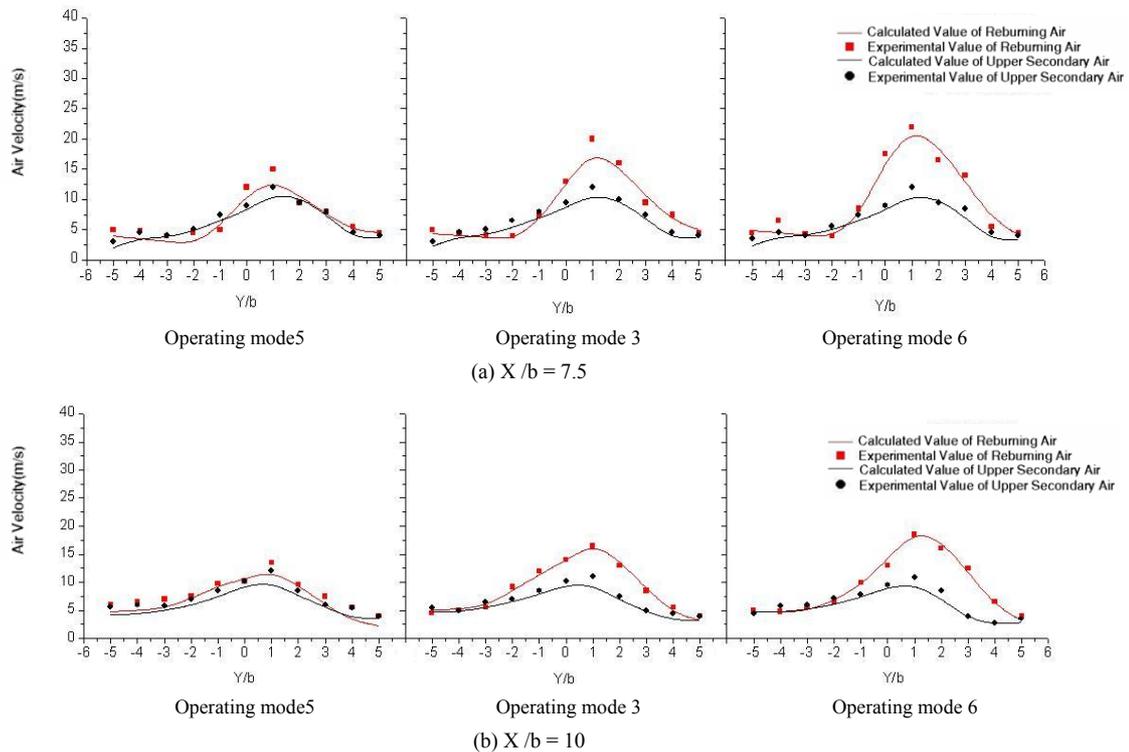


Figure 5. Velocity distribution of reburning air in four corners and upper secondary air for operating mode 3,5,6.

## 5. Conclusions

1) Effect on reduction NO<sub>x</sub> is better when eight nozzles are arranged in the four corners of the furnace and the center of side wall, compared with only arranges four nozzles in the four corners of the furnace.

The mutil-nozzles setting can decrease the NO<sub>x</sub> more efficiently for have better effect on the coverage of the updraft in the furnace, which is advantageous for improving the generation efficiency.

2) When eight nozzles are set in the reburner, the incident angle of the nozzle has a huge impact on the coverage effect of the updraft in the furnace. In the condition that the velocity of the reburning air is constant, the reburning flow has the best coverage quality for overfire air with the incident angle in corners of 41.17° and the incident angle in the furnace wall of 84.57°.

3) The speed of the reburning flow also has an effect on the decreasing of NO<sub>x</sub>. In the condition of maintaining the incident angle of four-corner nozzles with 41.17° and the incident angle of side wall nozzles with 84.57°, the reburning air has the best coverage quality for the upper secondary air when the speed of reburning air in corner is 51m/s and so as to the speed of reburning air in the center of the furnace wall.

## REFERENCES

- [1] J. Mereb and J. O. L. Wendt, "Reburning Mechanism in pulverized Coal Combustor," Twenty-third Symposium (international) on Combustion Institute, 1990, pp. 1273-1304.
- [2] M. Patry and G. Engel, "Formation of HCN by the Action of Nitric Oxide on Methane Atmospheric Pressure, General Conditions of Formation. Compt. Rend, Vol. 231, 1950, pp. 1302-1304
- [3] X. D. Yao, Z. X. Zhang, L. L. Qiu, *et al.*, "Analysis on the Kinetic Mechanism and Key Parameters of NO<sub>x</sub> Reduction with Natural Gas Reburning," *Journal of University of Shanghai For Science and Technology*, Vol. 26, No. 1,2004, pp. 62-65.
- [4] Z. G. Li, "Similar and Modeling," Beijing: National Defence Industry Press,1982.
- [5] K. F. Cen, "research method and measurement technology for Boiler combustion experimental [M]. Water Conservancy and Electric Power Press,1987.
- [6] Q. Guo, W. J. Ma and R. Sun, "Experimental Study on Furnace Air Flow Field of a 2000t/h Tangentially Fired Boiler," *Power System Engineering*, 2010.
- [7] W. Q. Tan, "Numerical Heat Transfer(version 2)," Xi'an: Xi'an Jiaotong University Press, 2001.
- [8] Z. H. Tuo, Z. Hang, X. Zhong, X. J. Wu, *et al.*, "Study on Flow Characteristics of Nozzle in Gas-reburning burner," *Journal of University of Shanghai For Science and Technology*, Vol. 29, No. 2, 2007, pp. 137-141.
- [9] Z. Y. Dong, "Jet Theory of Fluid Mechanics, beijing: Science press, 2005
- [10] S. Y. Wu, Y. R. Li, X. F. Lu, *et al.*, "Cold State Analysis

on the Jet of Single Nozzle Using Gas-reburning for NO<sub>x</sub> Reduction," *Industrial Heating*, Vol. 6, 2001, pp. 4-7.

- [11] Y. Wang, Y. K. Qin, S. H. Wu, *et al.*, "Numerical Simulation and Analysis of the Effect of an Infurnace Flow Field on the High Temperature Corrosion of Water Walls in a Tangentially Fired Boiler Furnace," *Journal of Engineering for Thermal Energy and Power*, Vol. 15, No. 87, 2000, pp. 284-329.
- [12] T. Zhu, W. D. Fan, *et al.*, "A Numerical Simulation Study of Aerodynamic Field Characteristics in a Double Furnace," *Journal of Engineering for Thermal Energy and Power*, Vol. 12, No. 06, 1997, pp. 401-477.