

Unsteady Behavior of Necklace Vortex Produced by a Square Plate Protrusion

Keita Toda¹, Tsukasa Haraoka¹, Teruyoshi Sadahiro², Hidemi Yamada¹

¹Department of Mechanical and Energy Systems Engineering, Oita University, Oita, Japan ²Department of Mechanical Engineering, National Institute of Technology, Sasebo College, Sasebo, Japan Email: yamada@oita-u.ac.jp

How to cite this paper: Toda, K., Haraoka, T., Sadahiro, T. and Yamada, H. (2018) Unsteady Behavior of Necklace Vortex Produced by a Square Plate Protrusion. *Open Journal of Fluid Dynamics*, **8**, 59-72. https://doi.org/10.4236/ojfd.2018.81005

Received: January 30, 2018 **Accepted:** March 26, 2018 **Published:** March 29, 2018

Copyright © 2018 by authors and Scientific Research Publishing Inc. This work is licensed under the Creative Commons Attribution International License (CC BY 4.0).

http://creativecommons.org/licenses/by/4.0/

CC O Open Access

Abstract

In this study, the unsteady behavior of necklace vortices formed in front of a square flat plate was experimentally investigated by flow visualization and PIV analysis by using a water channel. As a result, the necklace vortices started to oscillate when the Reynolds number increased beyond approximately 2000. Then, an amalgamation behavior of the necklace vortices took place when the Reynolds number increased beyond approximately 2650. Furthermore, in the same Reynolds number range, a breakaway behavior appeared when the relative height of the square plate was beyond approximately $h/\delta = 4.0$. The features of the necklace vortices behaviors in the oscillation, amalgamation and breakaway states were explained by observing the time-series image of path lines and by analyzing the frequency of velocity fluctuation.

Keywords

Necklace Vortex, Unsteady Flow, Square Plate, Visualization, PIV Analysis

1. Introduction

It is well known that the necklace vortex system is formed around a three-dimensional bluff body. Regarding the steady necklace vortex, a circular cylinder, a rectangular cylinder or a flat plate was mainly selected as a three-dimensional bluff body. The steady necklace vortex around a circular cylinder was experimentally studied by Baker [1]. The rectangular cylinder was dealt with by Seal *et al.* [2]. The flat plate was dealt with by Lin *et al.* [3] and Na-kahara *et al.* [4]. In particular, the characteristics of the unsteady necklace vortex are shown below. Baker experimentally studied an unsteady necklace vortex system formed around a short circular cylinder with a diameter-to-height ratio of 2. Then, he showed that nature of complex oscillatory behavior was described by

measuring the velocity distribution [5]. Visbal showed streamlines of the steady necklace vortices and the unsteady processes by using numerical calculations [6]. Seal *et al.* studied a breakaway behavior of the necklace vortices formed by a long rectangular cylinder which protrudes through the water surface [7]. Tsuruno *et al.* and Matsuguchi *et al.* studied the transition process of an unsteady necklace vortex system in front of a short square cylinder with a width-to-height ratio of 2 [8] [9] [10]. Lin *et al.* investigated the oscillation and breakaway processes of a necklace vortex upstream from a vertical rectangular plate (h/w = 0.5 - 4.0) and a short square cylinder (h/w = 1.08) on a ground wall. They explained that the boundary layer thickness is an important length scale controlling the cycle in the oscillation and breakaway processes of the necklace vortex [3] [11].

In studies on the necklace vortex, various three-dimensional bluff body shapes have been used, as mentioned above. Among them, a thin plate can be regarded as a typical bluff body, because there is no reattachment to itself downstream of the separation on the plate edge. However, there are few reports on unsteady necklace vortices produced by the thin square flat plate, except the reports of Lin *et al.* Lin *et al.* did not distinguish between flat plates having different aspect ratios, w/h, in his experiment results. The necklace vortex produced by the thin square flat plate should be researched in more details, because it is thought to be the most basic shape among vertical thin plates.

Therefore, we focused on the unsteady necklace vortices produced in front of the thin square flat plate protrusion perpendicular to the ground wall on which the laminar boundary layer exists. The purpose of this paper is to clarify the influence of the Reynolds number $R_{\rm e}$ and the reference laminar boundary layer thickness δ on the behavior of the unsteady necklace vortices produced by the thin square plate on the ground wall. On this account, flow visualization and PIV analysis were carried out while an unsteady necklace vortex state appeared in the Reynolds numbers between 2000 and 3250.

2. Experimental Apparatus and Procedure

Figure 1 shows an open test section of a water channel used in this experiment. The test section is made of transparent acrylic walls, and has a width of 300 mm, a height of 300 mm and a length of 1100 mm. The coordinate system and nomenclature also are shown in the same figure. The contraction area ratio of the inlet nozzle is 0.5. A perforated plate, a honeycomb and four woven wire screens are installed upstream of the inlet nozzle. A ground wall, which let a square plate protrusion stand, is an acrylic plate with a width of 300 mm, a length of 1000 mm and a thickness of 5 mm. The ground wall, whose leading-edge shape is one half the major axis of an ellipse, was submerged in a position 50 mm above the test section floor. Therefore, a new laminar boundary layer was formed on the ground wall.

The unsteady necklace vortices were produced by the protrusion of a thin square plate with a thickness of 1.5 mm standing perpendicularly to the ground

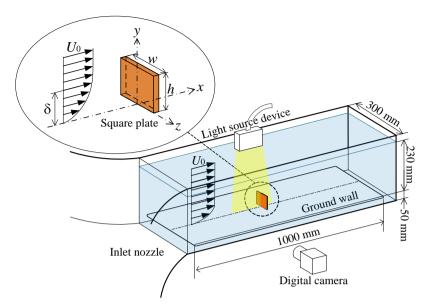


Figure 1. Schematic diagram of experimental apparatus and coordinate system.

wall on which the laminar boundary layer exists. The sizes h or w of the square plate and the blockage ratio are shown in **Table 1**. The blockage effect of the square plate on this flow field will not need to be considered, because the maximum blockage ratio is 4.38%. The square plate was stood at position 100, 150, 320, 500 or 800 mm from the front edge of ground wall.

In the experimental procedure, fine nylon particles, with an average diameter of 50 μ m (specific gravity about 1.03), were mixed in the water channel. The particle-images in the flow field were visualized by a metal-halide light irradiating from a slit of about 2 mm, and were successively taken by a digital camera (1280 \times 1024 pixels) at a frame rate of 100 or 200 fps. Each path lines-image was created by mutually superposing 50 particle-images. The resolution of the pixel was set in the range of 0.07 to 0.09 mm/pixel.

The velocity profile upstream of the square plate protrusion was obtained by using a time-series PIV. The uncertainty of velocity measurement by PIV was estimated to be approximately 6.2%, by comparing the Blasius' boundary layer velocity profile with the velocity profile measured by PIV at 320 mm position downstream from the front edge of the ground wall. On this account, the original boundary layer can be determined by the Blasius' solution. Therefore, the reference laminar boundary layer thickness $\delta (=\delta_{99})$ at the position assumed to stand the square plate was also obtained by the Blasius' solution. The flow visualization and PIV analysis were carried out by varying the relative plate height h/δ between approximately 2.0 and 5.0, and by varying the Reynolds numbers $R_{\rm e}$ (= $U_0 \cdot h/\nu$) between approximately 2000 and 3250.

3. Experimental Results and Discussions

When the Reynolds number was larger than approximately 2000, the waveform of velocity fluctuation became wavy and the turbulent intensity also increased in

Table 1. Square plate size and blockage ratio.	Table 1. Sc	juare plat	te size and	blockage ratio.
--	-------------	------------	-------------	-----------------

Square plate ($w = h$) [mm]	30	31	34	38	55
Blockage ratio [%]	1.30	1.39	1.68	2.09	4.38

the vicinity of the necklace vortices. In addition, the dominant frequency appeared in FFT analysis of the velocity fluctuation. However, the necklace vortices were not always completely steady in $R_{\rm e} < 2000$, because the turbulent intensity in the vortex region increased gradually as the Reynolds number approached 2000. Therefore, the unsteady state in the necklace vortices may be induced by the turbulence inherent in the vortices.

3.1. Oscillation Behavior

Figure 2(a)-(e) shows the path lines-images in about one period in the oscillation behavior of the necklace vortices in the case of $R_e = 2040$ and $h/\delta = 2.0$. Blue, yellow and gray lines indicate the *x*-positions of each vortex center against the time of t = 8.5 s in order to investigate a vortex movement. The velocity measurement positions P₁, P₂ and P₃ were also placed in a slightly upper position on V₁, V₂ and V₃ at the same time. Four clockwise necklace vortices (first vortex V₁, second vortex V₂, third vortex V₃ and fourth vortex V₄) oscillated upstream and downstream, as shown in **Figure 2**. When the first vortex V₁ moved downstream, the second vortex V₂ also moved downstream belatedly. Following this, the third vortex V₃ also moved downstream by following V₂.

Figure 3(a) shows the waveforms of y-direction velocity v measured simultaneously at measurement positions P_1 , P_2 , and P_3 . The waveforms had a phase shift to one another as mentioned above. The positive velocity in the waveform at each measurement position means that each corresponding vortex moves downstream, and negative velocity means vortex moving upstream. The waveforms at P_1 and P_2 were almost opposite phases, and the waveform at P_3 was later about 1/4 the period than that at P₂. Therefore, it is assumed that vortex V₁ and vortex V_2 approach each other in the time zone of A, as shown in Figure 3(a). In addition, V_2 and V_3 approach each other in the time zone of B. Figure 3(b) shows a frequency analysis of the y-direction velocity component measured in a time length of 41 s at the positions P_1 , P_2 , and P_3 . The dominant frequency was f_D = 0.195 Hz at three measurement positions, and corresponds to the Strouhal number $S_t = fh / U_0$ of 0.15. This means that the necklace vortices V_1 , V_2 and V_3 oscillate at the same period of T = 5.1 s. Figure 4(a) shows the waveforms of x-direction velocity u measured at the measurement positions P₁, P₂ and P₃. The phase of the waveforms almost corresponded with the phase of the y-direction velocity waveforms at P₁, P₂ and P₃, respectively. The x-direction velocity waveforms also indicated $f_D = 0.195$ Hz, as shown in **Figure 4(b)**.

3.2. Amalgamation Behavior

Figures 5(a)-(1) shows the path lines-images in about one period in the

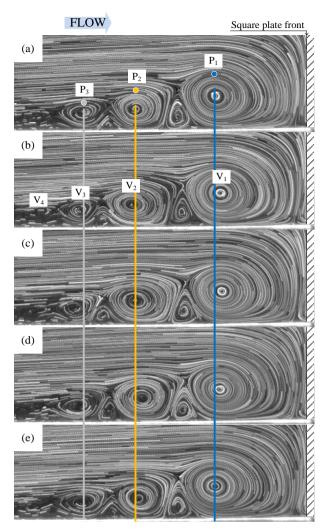


Figure 2. Time-series path lines-images in the oscillation behavior: (a) t = 8.5 s; t/T = 0; (b) t = 9.5 s; t/T = 0.20; (c) t = 10.5 s; t/T = 0.39; (d) t = 11.5 s; t/T = 0.59; (e) t = 12.5 s; t/T = 0.78: Position P₁ (*x/h*; *y/h*) = (-0.48, 0.29); Position P₂ (-0.89, 0.20); Position P₃ (-1.21, 0.13).

amalgamation behavior of the necklace vortices in the case of $R_e = 3020$ and $h/\delta = 3.0$. Blue, yellow and gray lines indicate the *x*-positions of each vortex center against the time of t = 0 s. The first vortex V₁ clearly moved downstream in the time of t = 2 - 5 s and moved upstream in t = 6 - 8 s. The second vortex V₂ lately started to move downstream beyond t = 4 s. After the third vortex V₃ amalgamated the fourth vortex V₄ in the time of t = 4 - 5 s, V₃ lately started to move downstream together with V₂. For that reason, V₂ was enclosed in the narrow region between V₁ and V₃ in the time of t = 5.5 - 6.5 s. When V₂ closely approached V₁, a part of V₂ was amalgamated by V₁ in the time of t = 6.5 - 7 s. After V₃ amalgamated the remaining part of V₂, V₃ and V₄ moved downstream until the original positions of V₂ and V₃, respectively. They became newly V₂ and V₃ at the end time of period.

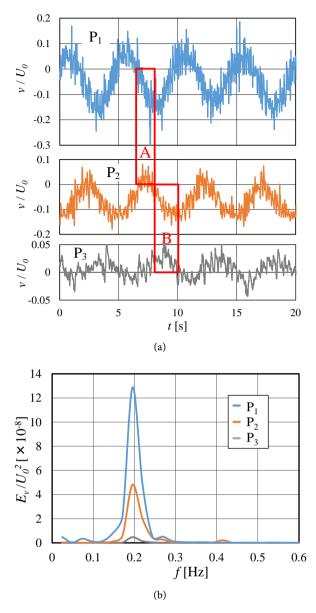


Figure 3. Analysis of *y*-direction velocity *v* in each measurement position in oscillation behavior: (a) Waveforms of velocity fluctuation; (b) FFT analysis of velocity fluctuation.

Figure 6(a) shows the waveforms of *y*-direction velocity *v* measured simultaneously at the measurement positions P_1 , P_2 and P_3 seen in **Figure 5**. Each waveform indicated evidently periodic patterns. The measurement positions P_1 , P_2 and P_3 were placed in a slightly upper position on V_1 , V_2 and V_3 at the time of t = 0 s. When the velocity is negative at P_1 and positive at P_2 , it is expected that the vortices V_1 and V_2 approach each other. In fact, V_1 and V_2 amalgamated in the time zone A, written in **Figure 6(a)**. In similar manner, V_2 and V_3 amalgamated in the time zone B when the velocity was negative at P_2 and positive at P_3 . **Figure 6(b)** shows a frequency analysis of the *y*-direction velocity component measured in a time length of 170 s at the positions P_1 , P_2 , and P_3 . The vortex V_1 would oscillate with 0.11 Hz, because the dominant frequency at the positions P_1

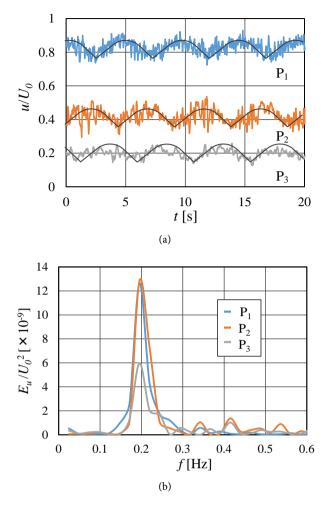


Figure 4. Analysis of *x*-direction velocity *u* in each measurement position in oscillation behavior: (a) Waveforms of velocity fluctuation; (b) FFT analysis of velocity fluctuation.

was $f_D = 0.11$ Hz. The period is T = 9.1 s. The waveform at the positions P₂ was distorted due to V₂ being periodically amalgamated by V₁ and V₃, as mentioned above. On this account, the dominant frequency at the position P₂ became $f_D = 0.11$ and 0.22 Hz, which was equal with the dominant frequency f_D at the position P₃. The Strouhal number for these dominant frequencies were $S_t = 0.11$ and 0.23.

3.3. Breakaway Behavior

Figures 7(a)-(e) shows the path lines-images in about one period in the breakaway behavior of the necklace vortices in the case of $R_e = 3250$ and $h/\delta = 5.0$. Blue and yellow lines indicate the *x*-positions of each vortex center against the time of t = 11 s. The feature of breakaway behavior is that the second vortex V₂ in the upstream position of the first vortex moves downstream with growing and amalgamates with the first vortex V₁, as mentioned by Lin *et al.* The time-series path lines-images in **Figure 7** also support the same feature. In addition, the vortex downstream moving occurred for all vortices except the first vortex. The

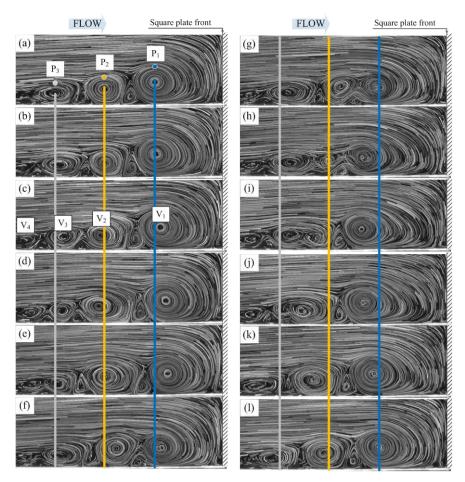


Figure 5. Time-series path lines-images in the amalgamation behavior: (a) t = 0 s; t/T = 0; (b) t = 1.0 s; t/T = 0.11; (c) t = 2.0 s; t/T = 0.22; (d) t = 3.0 s; t/T = 0.33; (e) t = 4.0 s; t/T = 0.44; (f) t = 5.0 s; t/T = 0.55; (g) t = 5.5 s; t/T = 0.60; (h) t = 6.0 s; t/T = 0.66; (i) t = 6.5 s; t/T = 0.71; (j) t = 7.0 s; t/T = 0.77; (k) t = 8.0 s; t/T = 0.88; (l) t = 9.0 s; t/T = 0.99: Position P₁ (*x/h*, *y/h*) = (-0.47, 0.35), Position P₂ (-0.81, 0.18), Position P₃ (-1.01, 0. 16).

first vortex V₁ was pushed downstream by the second vortex V₂ coming downstream at first, and was amalgamated in the time of t = 13 s. Soon after that, the coming vortex V₂ became a new first vortex V₁ in approximately a time of t = 14s in the upstream position of P₁ line.

Figure 8(a) shows the waveforms of *y*-direction velocity v at the measurement positions P₁ and P₂ which were placed in a slightly upper position on V₁ and V₂ at the time of t = 11 s. Though the waveform at the position P₁ seems to mean a oscillation of the same vortex V₁, V₁ is exchanged with the new vortex V₁ in the middle of the period. Similarly, though the waveform at the position P₂ seems to indicate an oscillation of the same vortex V₂, V₂ also is exchanged with a new vortex V₂ in the end of the period. Since V₂ was always moving downstream, V₁ was amalgamated by V₂ when V₁ started to move upstream of the original position P₁, as indicated by the time zone A. **Figure 8(b)** shows a frequency analysis of the *y*-direction velocity component measured in a time length of 82 s at the positions P₁, P₂, and P₃. The breakaway process would be repeated with 0.27 Hz,

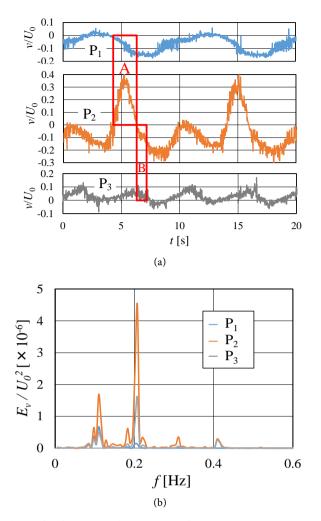


Figure 6. Analysis of *y*-direction velocity v in each measurement position in amalgamation behavior: (a) Waveforms of velocity fluctuation; (b) FFT analysis of velocity fluctuation.

because the dominant frequency at the positions P_1 and P_2 were $f_D = 0.27$ Hz ($S_t = 0.29$). The period is T = 3.7 s.

3.4. Classification of Unsteady Behavior

When the Reynolds number was smaller than 2000, the velocity measured in a slightly upper position on the first vortex V_1 did not provide such wavy waveform and dominant frequency as shown in **Figure 3**. On the other hand, the necklace vortices indicated the obvious oscillation state at $R_e = 2040$, as shown in **Figure 3**. Therefore, the boundary in the Reynolds number between the steady and unsteady necklace vortex states is considered to be approximately 2000.

In the Reynolds number investigated in this experiment, the unsteady necklace vortex was classified to the oscillation state, the amalgamation state and the breakaway state. **Figure 9** shows what kinds of unsteady necklace vortex appear in both conditions of the Reynolds number R_e and the relative plate height h/δ . The necklace vortex indicated the oscillation behavior between $R_e = 2000$ and 2650.

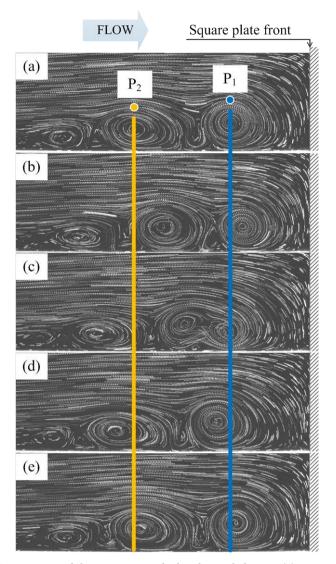


Figure 7. Time-series path lines-images in the breakaway behavior: (a) t = 11.0 s; t/T = 0; (b) t = 12.0 s; t/T = 0.27; (c) t = 13.0 s; t/T = 0.54; (d) t = 14.0 s; t/T = 0.81; (e) t = 15.0 s; t/T = 1.08: Position P₁ (*x/h*, *y/h*) = (-0.35, 0.17), Position P₂ (-0.65, 0.17).

The amalgamation behavior started beyond $R_e = 2650$. The transition boundary from the oscillation state to the amalgamation state depended on only the Reynolds number. In addition, the breakaway state appeared in about $R_e \ge 3200$ and $h/\delta \ge 4.0$. Therefore, only the breakaway state is dependent on the Reynolds number and the relative plate height.

3.5. Feature of Unsteady Necklace Vortex

The Strouhal number was calculated from the dominant frequency of the velocity waveform measured in a slightly upper position on the first vortex V₁ against the various cases of R_e and h/δ . Figure 10 shows the distribution of the Strouhal number. There is only one dominant frequency in the oscillation and breakaway states, as mentioned in Session 3.1 and 3.3. On the other hand, two dominant

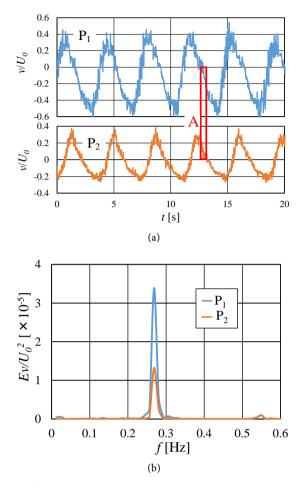


Figure 8. Analysis of *y*-direction velocity *v* in each measurement position in breakaway behavior: (a) Waveforms of velocity fluctuation; (b) FFT analysis of velocity fluctuation.

frequencies appeared in the amalgamation state, as mentioned in Session 3.2. The Strouhal number in the oscillation state was almost constant with each h/δ . Then, the Strouhal number was approximately $S_t = 0.15$ in the case of $h/\delta = 2.0$, and approximately $S_t = 0.19$ in $h/\delta = 3.0$. The first Strouhal number in the amalgamation state gradually increased from $S_t = 0.15$ with increasing the Reynolds number. The second Strouhal number was approximately half of the first Strouhal number. The Strouhal number in the breakaway state was larger than that in the amalgamation state.

4. Conclusions

In this study, the unsteady behavior of necklace vortices formed in front of a square flat plate protrusion was clarified based on the time-series image of path lines and the frequency analysis of velocity fluctuation. The following results were gained:

1) There exist three kinds of unsteady necklace vortices in the range of $2000 \le R_{\rm e} \le 3250$, those are the oscillation state, the amalgamation state, and the breakaway state.

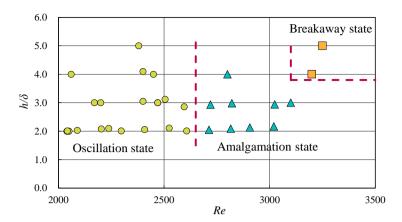


Figure 9. Classification of unsteady necklace vortex state. Red broken lines show the transition boundary of the unsteady necklace vortex state.

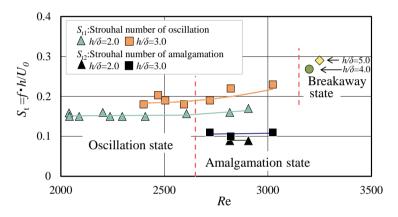


Figure 10. Strouhal number in unsteady necklace vortex movement. Red broken lines show the transition boundary of the unsteady necklace vortex state.

2) In the oscillation state, four clockwise necklace vortices (V₁, V₂, V₃ and V₄ vortices) periodically move downstream and upstream. The Strouhal number is almost constant against the Reynolds number with each h/δ . It is approximately $S_t = 0.15$ in the case of $h/\delta = 2.0$.

3) In the amalgamation state, each vortex periodically moves, and amalgamates when adjacent vortices closely approach each other. Therefore, there are two kinds of dominant frequencies which correspond with the oscillation and amalgamation phenomena. The first Strouhal number gradually increases from $S_t = 0.15$. The second Strouhal number is approximately half of the first Strouhal number.

4) In the breakaway state, all vortices in the upstream position of the first vortex moves downstream with growing. Successively, the second vortex V_2 amalgamates the first vortex V_1 , and replaces V_1 as a new V_1 . The Strouhal number of the breakaway process would be larger than that of the amalgamation state.

5) The necklace vortex starts to oscillate when the Reynolds number increases beyond approximately 2000. Then, an amalgamation behavior takes place when the Reynolds number increases beyond approximately 2650. Furthermore, in the same Reynolds number range, a breakaway behavior appears when the relative height of the square plate is beyond approximately $h/\delta = 4.0$.

References

- [1] Baker, C.J. (1979) The Laminar Horseshoe Vortex. Journal of Fluid Mechanics, 95, 347-367. https://doi.org/10.1017/S0022112079001506
- [2] Seal, C.V., Smith, C.R., Akin, O. and Rockwell, D. (1995) Quantitative Characteristics of a Laminar, Unsteady Necklace Vortex System at a Rectangular Block-Flat Plate Juncture. Journal of Fluid Mechanics, 286, 117-135. https://doi.org/10.1017/S002211209500067X
- [3] Lin, C., Chiu, P.H. and Shieh, S.J. (2002) Characteristics of Horseshoe Vortex System near a Vertical Plate-Base Plate Juncture. Experimental Thermal and Fluid Science, 27, 25-46. https://doi.org/10.1016/S0894-1777(02)00215-7
- [4] Nakahara, Y. and Yamada, H. (2015) Visualization of a Laminar Necklace Vortex System in Front of a Vertical Rectangular Plate on a Ground Wall. Journal of Flow Control, Measurement & Visualization, 3, 75-85. https://doi.org/10.4236/jfcmv.2015.32008
- [5] Baker, C.J. (1991) The Oscillation of Horseshoe Vortex Systems. Journal of Fluids Engineering, 113, 489-495. https://doi.org/10.1115/1.2909523
- [6] Visbal, M.R. (1991) Structure of Laminar Juncture Flows. AIAA Journal, 29, 1273-1282. https://doi.org/10.2514/3.10732
- [7] Seal, C.V., Smith, C.R. and Rockwell, D. (1997) Dynamics of the Vorticity Distribution in Endwall Junctions. AIAA Journal, 35, 1041-1047. https://doi.org/10.2514/2.192
- [8] Tsuruno, S., Matsuguchi, A. and Tsuda, N. (1997) Study on Transition Process of Unsteady Horse-Shoe Vortex by PIV. Journal of the Visualization Society of Japan, 17, 137-140. (In Japanese) https://doi.org/10.3154/jvs.17.Supplement1_137
- [9] Matsuguchi, A., Tsuda, N. and Tsuruno, S. (1999) Study on Transition Process of Unsteady Horse-Shoe Vortex. The Japan Society of Mechanical Engineers, 65, 28-35. (In Japanese)
- [10] Matsuguchi, A., Kagawa, N., Tsuda, N. and Tsuruno, S. (2001) Study on Transition Process of Horse-Shoe Vortex (Examination of the Similarity). National Heat Transfer Symposium of Japan, 667-668. (In Japanese)
- [11] Lin, C., Lai, W. and Chang, K. (2003) Simultaneous Particle Image Velocimetry and Laser Doppler Velocimetry Measurements of Periodical Oscillatory Horseshoe Vortex System near Square Cylinder-Base Plate Juncture. Journal of Engineering Mechanics, 129, 1173-1188.

https://doi.org/10.1061/(ASCE)0733-9399(2003)129:10(1173)

Nomenclatures

(x, y, z): Cartesian coordinate system

h: Height of square flat plate [m]

 δ : Reference laminar boundary layer thickness (= δ_{99}) [m]

*U*₀: Free stream velocity [m/s]

u: x-direction velocity component [m/s]

v: *y*-direction velocity component [m/s]

t: time [s]

T: One period in unsteady necklace vortex behavior

 h/δ : Relative height of square plate

*R*_e: Experimental Reynolds number

 ν : Kinematic viscosity $[m^2/s]$

 S_{t} : Strouhal number (=*f*·*h*/ U_{0})

f. Frequency of velocity fluctuation [Hz]

 $f_{\rm D}$: Dominant frequency [Hz]