

PD Power-Level Control Design for MHTGRs

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Abstract

Due to its inherent safety feature, the modular high temperature gas-cooled reactor (MHTGR) has been seen as one of the best candidates in building next generation nuclear plants (NGNPs). Since the MHTGR dynamics has high nonlinearity, it is necessary to develop nonlinear power-level controller which is not only beneficial to the safe, stable, efficient and autonomous operation of the MHTGR but also easy to be implemented practically. In this paper, based on the concept of shifted-entropy and the physically-based control design approach, it is proved theoretically that the simple proportional-differential (PD) output-feedback power-level control can provide globally asymptotic closed-loop stability. Numerical simulation results verify the theoretical results and show the influence of the controller parameters to the dynamic response.

Keywords

Modular High Temperature Gas-Cooled Reactor (MHTGR); Power-Level Control; Closed-Loop Stability

1. Introduction

Due to its inherent safety performance, the modular high temperature gas-cooled reactor (MHTGR) has been seen as one of the best candidates for building next generation nuclear plants. The MHTGR uses helium as coolant and graphite as moderator and structural material, and its inherent safety is given by the low power density, strong negative temperature feedback effect and slim reactor shape [1]. China began to study the MHTGR at the end of 1970s, and a 10 MW_{th} pebble-bed high temperature gas-cooled test reactor HTR-10 designed by institute of nuclear and new energy technology (INET) of Tsinghua University achieved its criticality in December 2000 and full power in January 2003 [2]. Then, six safety demonstration tests were done on HTR-10, which manifested its inherent safety and self-stabilizing features [3]. Based on the experience of the HTR-10 project, a high temperature gas-cooled reactor pebble-bed module (HTR-PM) project was then proposed [4]. As shown in **Figure 1**, the HTR-PM plant consists of two one-zone MHTGRs with combined thermal power of 2×250 MW_{th}, and has the structure of two nuclear steam supplying systems (NSSSs) driving one steam turbine [4]. Here, the NSSS is composed of an MHTGR, a helical coiled once-through steam generator (OTSG) and some connecting pipes.

Since a MHTGR is essentially a nonlinear dynamical system, it is necessary to develop nonlinear power-level control laws of the MHTGR for safe, stable and efficient operation. Actually, nonlinear power-level control design is a hot field in nuclear engineering, and there have been some promising nonlinear reactor control design methods. Shtessel gave a nonlinear power-level regulator based on sliding mode control and observation tech-

niques for space reactor TOPAZ II [5]. Dong designed a dynamic output feedback dissipation power-level control for the pressurized water reactors (PWRs) [6] by the use of the backstepping technique [7] and dissipation-based high gain filter (DHGF) [8,9]. Etchepareborda and Eliasi proposed the nonlinear MPC (NMPC) method for PWR power-level control design [10-12]. However, the forms of the above nonlinear power-level control laws are too complicated to be implemented practically. Control design by fully using the good natural system dynamics, *i.e.* the physically-based control design method can lead to simple and effective controllers, and is a promising trend of advanced control theory [13-15]. Very recently, based on the physically-based design approach, Dong proposed a nonlinear dynamic output-feedback power-level control for the PWRs [16], and also proved theoretically that the simple proportional-differential (PD) power-level control could guarantee globally asymptotic closed-loop stability for the PWRs [17].

Since the dynamic features of the MHTGR is different from that of the PWR, the power-level control designed for the PWR cannot directly applied to the MHTGR. It is necessary to develop nonlinear power-level controller for the MHTGRs. Dong designed a nonlinear state-feedback power-level control strategy to the MHTGR based on the technique of iterative damping assignment (IDA) [18]. Although this IDA-based control can provide globally asymptotic closed-loop stability, its mathematical form is too complex to be implemented practically. Based upon the physically-based control design approach, Dong also gave a nonlinear dynamic output feedback power-level controller for the MHTGR [19]. However, this control is still complicated in its form. Therefore, it is necessary to design simple power-level control laws for the MHTGR with strong load following capability.

In this paper, based on the concept of shifted-entropy and physically-based control design method, it is proved theoretically that the static output-feedback control with simple PD structure can globally asymptotically stabilize the MHTGR. Numerical simulation results not only verify the theoretical results but also illustrate the relationship between performance and controller parameters.

2. Problem Formulation

2.1. Nonlinear State-Space Model

As shown in **Figure 1**, the MHTGR and OTSG of the NSSS is arranged side by side, and is connected to each other by a horizontal coaxial gas duct. The cold helium enters the main blower mounted on top of the OTSG, and is pressurized before flowing into the cold gas duct. It enters the channels inside the reflector of the core, and then passes through the pebble-bed from top to bottom where it is heated to a high temperature. The hot helium leaves the hot gas chamber at the bottom reflector, and flows into the primary side of the OTSG through the hot gas duct. The primary loop can be nodalized as the elements given in **Figure 2**.

By adopting the point kinetics with one equivalent delayed neutron group and with the temperature reactivity feedback effect of the pebble-bed/reflector community, the dynamical model for control design can be written as

$$\begin{cases} \dot{n}_r = \frac{\rho_r - \beta}{\Lambda} n_r + \frac{\beta}{\Lambda} c_r + \frac{\alpha_R}{\Lambda} n_r (T_R - T_{R,m}), \\ \dot{c}_r = \lambda (n_r - c_r), \\ \dot{T}_R = -\frac{\Omega_p}{\mu_R} (T_R - T_H) + \frac{P_0}{\mu_R} n_r, \\ \dot{T}_H = \frac{\Omega_p}{\mu_H} (T_R - T_H) - \frac{\Omega_s}{\mu_H} (T_H - T_S), \\ \dot{\rho}_r = G_r z_r, \end{cases} \quad (1)$$

where n_r is the relative neutron power, c_r is the relative concentration of delayed neutron precursor, β is the fraction of delayed fission neutrons, Λ is the effective prompt neutron life time, ρ_r is the reactivity provided by the control rods, λ is the effective radioactive decay constant of the precursor, T_R and α_R is the temperature and reactivity feedback coefficient of the community constituted by both the pebble-bed and reflector respectively, $T_{R,m}$ is the initial equilibrium value of T_R , P_0 is the rated reactor thermal power, T_H is the average helium temperature of the primary side, T_S is the average coolant temperature of the secondary side of the OTSG, Ω_p is the heat transfer coefficient between the helium and pebble-bed/reflector community, Ω_s is the heat transfer coefficient

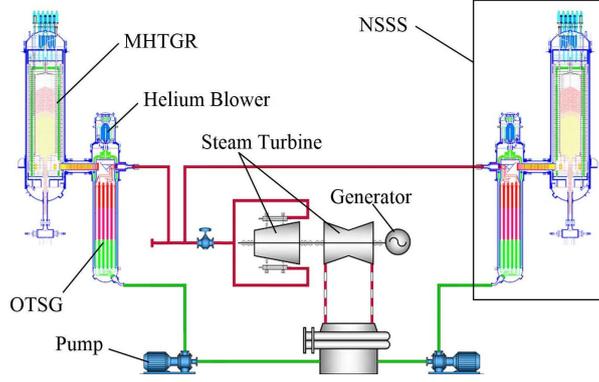


Figure 1. Composition of the HTR-PM plant.

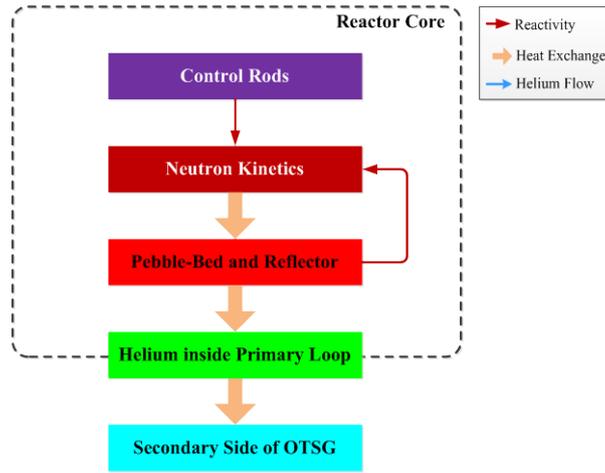


Figure 2. Nodalization of the primary loop.

between the two sides of OTSG, μ_R and μ_H is respectively the total heat capacities of the pebble-bed/reflector community and helium inside the primary loop, G_r is the total differential reactivity worth of the control rod, and z_r is the rod speed signal. Here, note that α_R is guaranteed to be negative by physical design of the MHTGR.

Define the deviations of the actual values of n_r , c_r , T_R , T_H , T_S and ρ_r from their equilibrium values, i.e. n_{r0} , c_{r0} , T_{R0} , T_{H0} and ρ_{r0} as $\delta n_r = n_r - n_{r0}$, $\delta c_r = c_r - c_{r0}$, $\delta T_R = T_R - T_{R0}$, $\delta T_H = T_H - T_{H0}$, $\delta T_S = T_S - T_{S0}$, and $\delta \rho_r = \rho_r - \rho_{r0}$. Here, δT_S reflects the influence of the secondary to primary loop, and can be well suppressed by adjusting the feedwater flow-rate of the OTSG. Therefore, in this paper, the influence of δT_S is omitted. Let

$$\mathbf{x} = [x_1 \ x_2 \ x_3 \ x_4]^T = [\delta n_r \ \delta c_r \ \delta T_R \ \delta T_H]^T, \quad (2)$$

$$\xi = \delta \rho_r, \quad (3)$$

and

$$\mathbf{u} = G_r z_r. \quad (4)$$

Here, \mathbf{x} is the reactor state-vector of the MHTGR. Then, the nonlinear state-space model for control design can be written as

$$\begin{cases} \dot{\mathbf{x}} = \mathbf{f}(\mathbf{x}) + \mathbf{g}(\mathbf{x})\xi, \\ \dot{\xi} = \mathbf{u}, \\ \mathbf{y} = \mathbf{h}(\mathbf{x}), \end{cases} \quad (5)$$

where

$$\mathbf{f}(\mathbf{x}) = \begin{bmatrix} -\frac{\beta}{\Lambda}(x_1 - x_2) + \frac{\alpha_R}{\Lambda}(n_{r0} + x_1)x_3 \\ \lambda(x_1 - x_2) \\ \frac{P_0}{\mu_R}x_1 - \frac{\Omega_p}{\mu_R}(x_3 - x_4) \\ \frac{\Omega_p}{\mu_H}(x_3 - x_4) - \frac{\Omega_s}{\mu_H}x_4 \end{bmatrix}, \quad (6)$$

$$\mathbf{g}(\mathbf{x}) = \begin{bmatrix} \frac{n_{r0} + x_1}{\Lambda} & \mathbf{O}_{1 \times 3} \end{bmatrix}^T, \quad (7)$$

and

$$\mathbf{h}(\mathbf{x}) = [x_1 \quad x_4]^T. \quad (8)$$

2.2. Theoretic Problem

Based on the above modeling, the theoretic problem to be solved in this paper is summarized as follows.

Problem 1. How to design an output-feedback PD control law of nonlinear system (5) taking the form as

$$u = u(\mathbf{y}, \dot{\mathbf{y}}), \quad (9)$$

so that $\mathbf{x} \rightarrow \mathbf{O}$ as $t \rightarrow \infty$? □

3. PD Power-Level Control Design

Following Theorem 1, i.e. the main result of this paper, shows that simple output-feedback PD power-level control law can guarantee asymptotic closed-loop stability of reactor state-variables.

Theorem 1. There exists a PD power-level control law of nonlinear system (5) that provides globally asymptotic closed-loop stability for the reactor state of the MHTGR, i.e. $\mathbf{x} \rightarrow \mathbf{O}$ as $t \rightarrow \infty$.

Proof: Based upon the idea of backstepping, a virtual control input ξ_r is firstly designed for subsystem

$$\begin{cases} \dot{\mathbf{x}} = \mathbf{f}(\mathbf{x}) + \mathbf{g}(\mathbf{x})\xi_r, \\ \mathbf{y} = \mathbf{h}(\mathbf{x}). \end{cases} \quad (10)$$

From [17], the shifted-entropy of neutron kinetics is

$$\zeta_N(x_1, x_2) = n_{r0} \left\{ \Lambda \left[\left(1 + \frac{x_1}{n_{r0}} \right) - \ln \left(1 + \frac{x_1}{n_{r0}} \right) \right] + \frac{\beta}{\lambda} \left[\left(1 + \frac{x_2}{n_{r0}} \right) - \ln \left(1 + \frac{x_2}{n_{r0}} \right) \right] \right\}. \quad (11)$$

Based on (11), let the Lyapunov function for the neutron kinetics be

$$V_N(x_1, x_2) = \zeta_N(x_1, x_2) + \frac{k_1}{2} \left[\int_0^t x_1(s) ds \right]^2. \quad (12)$$

Here, the objective of adding the second term of V_N is to minimize the steady error of n_r by feedback control. Then, differentiate V_N along the trajectory given by neutron kinetics, and we have

$$\dot{V}_N(x_1, x_2) = -\frac{\beta(x_1 - x_2)^2}{(n_{r0} + x_1)(n_{r0} + x_2)} + x_1 \left[\xi_r + \alpha_R x_3 + k_1 \int_0^t x_1(s) ds \right]. \quad (13)$$

Moreover, it is clear that the shifted-entropy of reactor thermal-hydraulics can be written as

$$\zeta_T(x_3, x_4) = \frac{1}{2} (\mu_R x_3^2 + \mu_H x_4^2). \quad (14)$$

Then, based on (14), let the Lyapunov function of reactor thermal-hydraulics be

$$V_T(x_3, x_4) = (1-\gamma)\zeta_T(x_3, x_4) + \gamma\varsigma_T(x_3, x_4), \quad (15)$$

where γ is a positive given constant satisfying $0 < \gamma < 1$, and

$$\varsigma_T = \frac{1}{2\mu_R} \left[\mu_R x_3 + \mu_H x_4 + \Omega_S \int_0^t x_4(s) ds \right]^2 \quad (16)$$

denotes the energy variation of the thermal-hydraulic loops. Differentiate (15) along the trajectory given by the reactor thermal-hydraulics, and we have

$$\begin{aligned} \dot{V}_T = & P_0 x_1 x_3 + \gamma P_0 x_1 \left[\frac{\mu_H}{\mu_R} x_4 + \frac{\Omega_S}{\mu_R} \int_0^t x_4(s) ds \right] - (1-\gamma)(1-\eta) \left[\Omega_p (x_3 - x_4)^2 + \Omega_s x_4^2 \right] \\ & - (1-\gamma)\eta \left[(\Omega_p + \Omega_s) \left(x_4 - \frac{\Omega_p}{\Omega_p + \Omega_s} x_3 \right)^2 + \frac{\Omega_p \Omega_s}{\Omega_p + \Omega_s} x_3^2 \right], \end{aligned} \quad (17)$$

where η is given positive constant satisfying $0 < \eta < 1$.

Choose the Lyapunov function for subsystem (10) as

$$V_1(\mathbf{x}) = V_N(x_1, x_2) + \frac{q_R}{P_0} V_T(x_3, x_4), \quad (18)$$

where q_R is a given positive constant, V_N and V_T is given by (12) and (15) respectively. Differentiate (18) along the trajectory given by subsystem dynamics (10), and we can derive that

$$\begin{aligned} \dot{V}_1 = & -\frac{\beta(x_1 - x_2)^2}{(n_{r0} + x_1)(n_{r0} + x_2)} - (1-\gamma)(1-\eta) \frac{q_R}{P_0} \left[\Omega_p (x_3 - x_4)^2 + \Omega_s x_4^2 \right] \\ & - (1-\gamma)\eta q_R \frac{\Omega_p + \Omega_s}{P_0} \left(x_4 - \frac{\Omega_p}{\Omega_p + \Omega_s} x_3 \right)^2 - \frac{q_R}{2\Sigma_R} \left\{ x_3^2 + \left[x_3 - \Sigma_R \left(1 + \frac{\alpha_R}{q_R} \right) x_1 \right]^2 \right\} \\ & + x_1 \left\{ \xi_r + \frac{q_R \Sigma_R}{2P} \left(1 + \frac{\alpha_R}{q_R} \right)^2 x_1 + k_I \int_0^t x_1(s) ds + \gamma q_R \left[\frac{\mu_H}{\mu_R} x_4 + \frac{\Omega_S}{\mu_R} \int_0^t x_4(s) ds \right] \right\} \end{aligned} \quad (19)$$

where

$$\Sigma_R = \frac{(\Omega_p + \Omega_s) P_0}{(1-\gamma)\eta\Omega_p\Omega_s}. \quad (20)$$

From equation (19), if we design virtual control ξ_r as

$$\xi_r = -k_{ND} x_1 - k_I \int_0^t x_1(s) ds - \gamma q_R \left[\frac{\mu_H}{\mu_R} x_4 + \frac{\Omega_S}{\mu_R} \int_0^t x_4(s) ds \right], \quad (21)$$

where

$$k_{ND} > \frac{q_R (1-\gamma)\eta\Omega_p\Omega_s}{2P_0(\Omega_p + \Omega_s)} \left(1 + \frac{\alpha_R}{q_R} \right)^2, \quad (22)$$

then the closed-loop subsystem constituted by (10) and (21) is globally asymptotically stable.

Now, we design the control law for entire system (5). Choose the Lyapunov function of the entire system as

$$V_2(\mathbf{x}, e_\xi) = V_1(\mathbf{x}) + \frac{e_\xi^2}{2k_\xi}, \quad (23)$$

where k_ξ is a given positive constant, and

$$e_\xi = \xi - \xi_r, \quad (24)$$

Differentiate (23) along the trajectory given by entire system dynamics (5), and we have

$$\begin{aligned} \dot{V}_2 = & -\frac{\beta(x_1 - x_2)^2}{(n_{t0} + x_1)(n_{t0} + x_2)} - (1 - \gamma)(1 - \eta) \frac{q_R}{P_0} \left[\Omega_p (x_3 - x_4)^2 + \Omega_s x_4^2 \right] \\ & - (1 - \gamma) \eta q_R \frac{\Omega_p + \Omega_s}{P_0} \left(x_4 - \frac{\Omega_p}{\Omega_p + \Omega_s} x_3 \right)^2 - \frac{q_R}{2\Sigma_R} \left\{ x_3^2 + \left[x_3 - \Sigma_R \left(1 + \frac{\alpha_R}{q_R} \right) x_1 \right]^2 \right\} \\ & - \tilde{k}_{ND} x_1^2 + x_1 e_\xi + \frac{1}{k_\xi} e_\xi (u - \dot{\xi}_r), \end{aligned} \quad (25)$$

where

$$\tilde{k}_{ND} = k_{ND} - \frac{q_R (1 - \gamma) \eta \Omega_p \Omega_s}{2P_0 (\Omega_p + \Omega_s)} \left(1 + \frac{\alpha_R}{q_R} \right)^2. \quad (26)$$

From (25), if we choose feedback control u as

$$u = -k_\xi x_1 + \dot{\xi}_r = -(k_{NP} x_1 + k_{ND} \dot{x}_1 + k_{TP} x_4 + k_{TD} \dot{x}_4), \quad (27)$$

where

$$k_{NP} = k_\xi + k_1, \quad (28)$$

$$k_{TP} = \gamma q_R \frac{\Omega_s}{\mu_R}, \quad (29)$$

and

$$k_{TD} = \gamma q_R \frac{\mu_H}{\mu_R}, \quad (30)$$

then we have

$$\begin{aligned} \dot{V}_2 = & -\tilde{k}_{ND} x_1^2 - \frac{\beta(x_1 - x_2)^2}{(n_{t0} + x_1)(n_{t0} + x_2)} - (1 - \gamma)(1 - \eta) \frac{q_R}{P_0} \left[\Omega_p (x_3 - x_4)^2 + \Omega_s x_4^2 \right] \\ & - (1 - \gamma) \eta q_R \frac{\Omega_p + \Omega_s}{P_0} \left(x_4 - \frac{\Omega_p}{\Omega_p + \Omega_s} x_3 \right)^2 - \frac{q_R}{2\Sigma_R} \left\{ x_3^2 + \left[x_3 - \Sigma_R \left(1 + \frac{\alpha_R}{q_R} \right) x_1 \right]^2 \right\}. \end{aligned} \quad (31)$$

Based on equation (31), it is clear that there always exists a PD power-level controller (27) so that reactor state x of MHTGR dynamics (5) are globally asymptotically stable. This completes the proof of this theorem. \square

Remark 1. From equation (12) and (28), the steady error of relative nuclear power can be suppressed by enlarging the proportional feedback gain k_{NP} corresponding to δn_r . Also from (18), the dynamic performance of the thermal-hydraulic loop can be strengthened through enlarging q_R , which certainly leads to larger values of feedback gains k_{ND} , k_{TP} and k_{TD} . \square

Remark 2. Since positive constants γ and η can be arbitrarily chosen between 0 and 1, inequality (22) is easy to be satisfied by choosing γ to be close enough to 1 and η to be close enough to 0. However, larger γ also leads to larger k_{TP} and k_{TD} . \square

4. Numerical Simulation with Discussions

4.1. Description of the Numerical Simulation

To verify the stabilization capability of PD control (27), it is applied to the power-level regulation of an MHTGR of the HTR-PM plant. Here, the dynamic model of the MHTGR used in this simulation adopts that one composed of both nodal neutron kinetics and nodal reactor thermal-hydraulics given in [20]. The OTSG adopts the moving boundary model presented in [21]. The model of the steam turbine and that of the electrical genera-

tor are also included in the simulation code [22]. The controller parameters are selected as $k_{NP} = k_{ND} = 0.5$, $\gamma = 0.5$. Here, q_R is set to be variable.

4.2. Simulation Results

In this simulation, the case of large-range power-level maneuver of the MHTGR is studied to show the feasibility of PD power-level control (27). As the power demand signal decreases linearly from 100% full power-level (FP) to 50% FP in 5 minutes, the error signals of the nuclear power and the helium temperature cause the power-level control to generate proper control rod speed to cope with the decrease of power demand. The responses of relative nuclear power, average fuel temperature and outlet helium temperature as well as the designed rod speed with different values of q_R are all shown in **Figure 3**.

4.3. Discussions

From **Figure 3**, we can see that the dynamic performance of reactor thermal-hydraulic loop is higher if q_R is larger. Moreover, a larger q_R results in the deterioration of the response of neutron kinetics. Actually, this phenomenon can be interpreted by the proof of Theorem 1.

From Equation (18), it is clear that the ratio of V_T in V_1 is higher if q_R is larger. Since V_T denotes the Lyapunov function of the thermal-hydraulic loop, larger ratio of V_T results in faster convergence of those thermal-hydraulic state-variables, which can be easily seen from **Figures 3(b)** and **(c)**. On the other hand, from Equation (26), larger q_R leads to smaller \tilde{k}_{ND} , which then weakens the convergence of neutron kinetic states. As we can see from **Figure 3(a)**, the oscillation of the relative nuclear power is tougher if q_R is larger. Thus, from the above discussion,

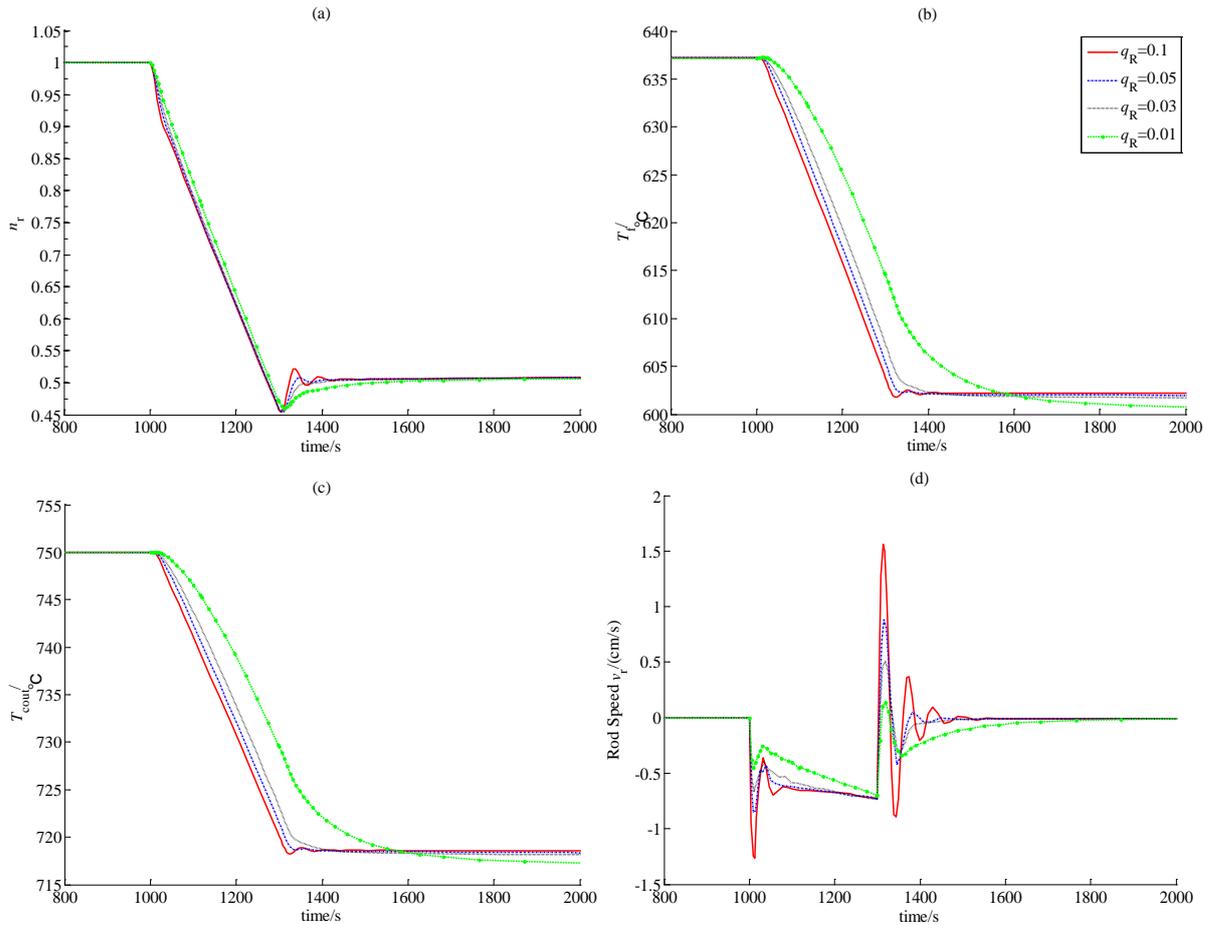


Figure 3. Numerical simulation results: (a) Relative nuclear power, (b) Average fuel temperature, (c) Outlet helium temperature, (d) Designed control rod speed signal.

we can see that the numerical simulation results given in Section 4.2 are in accordance with the theoretical analysis in Section 3. Moreover, the numerical simulation results also illustrate the relationship between dynamic performance and controller parameters.

5. Conclusion

Due to its inherent safety feature and potential economic competitive power, the modular high temperature gas-cooled reactor (MHTGR) has already been seen as one of the best candidates in building SMR-based nuclear power plant. Since power-level control is meaningful in providing safe, stable and efficient reactor operation, and an MHTGR is essentially a nonlinear dynamic system, it is crucial to develop nonlinear power-level control which can be easily implemented. Based upon the shifted-ectropies of both neutron kinetics and reactor thermal-hydraulics, it is proved theoretically that the simple PD power-level control can provide globally asymptotic closed-loop stability for the MHTGR. Numerical simulation results are consistent with the theoretical analysis, and also showed the relationship between the regulating performance and controller parameters.

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