

# Modeling and Adaptive Control of an Omni-Mecanum-Wheeled Robot

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Received December 14, 2012; revised February 6, 2013; accepted February 13, 2013

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## ABSTRACT

The complete dynamics model of a four-Mecanum-wheeled robot considering mass eccentricity and friction uncertainty is derived using the Lagrange's equation. Then based on the dynamics model, a nonlinear stable adaptive control law is derived using the backstepping method via Lyapunov stability theory. In order to compensate for the model uncertainty, a nonlinear damping term is included in the control law, and the parameter update law with  $\sigma$ -modification is considered for the uncertainty estimation. Computer simulations are conducted to illustrate the suggested control approach.

**Keywords:** Mecanum-Wheeled Mobile Robot; Dynamics Model; Backstepping Adaptive Control; Lyapunov Stability

## 1. Introduction

In 1973, Bengt Ilon invented the Mecanum wheel (also called Ilon wheel) when he was an engineer with the Swedish company Mecanum A.B. [1,2]. The Mecanum wheel is designed with passive rollers mounted around the wheel circumference at an angle of 45 degrees to the wheel plane, thus it allows for in place rotation with small ground friction and low driving torque. Usually the mobile robots using Mecanum wheels, such as an intelligent wheelchair, a forklift, or the URANUS omni-directional robot, are designed with four wheels to provide agile mobility in any direction without changing its orientation. This omni-directional capability provides greater flexibility in congested environments. Ould-Khessal [3] applied it in a robot soccer team design.

Although the benefit of omni-directionality of a standard Mecanum wheel, it has an unfortunate side effect of reducing the motor effective driving force through the rollers by projecting a portion of the motor force into a force perpendicular or at an angle to that produced by the motor. Thus, it may be inefficient when the platform travels in a straight line, especially when travels diagonally. Diegel *et al.* [4] proposed an improved Mecanum wheel design with a "twist" mechanism for adjusting and locking the angle of the passive rollers to best suit the direction the platform is traveling in. Since a planar mobile robot consisting of four Mecanum wheels has only three degrees of freedom (DOF): two translational motions along  $X$ - and  $Y$ -axes, and one rotation about  $Z$ -axis,

it has one redundant degree of freedom. Asama *et al.* [5] proposed a transmission mechanism such that four wheels can be driven by only three actuators, each of which drives wheels to move the robot for a certain DOF, respectively.

The most popular approach to the control of an omni-directional robot considers the kinematic control relying only on the kinematics model of the platform [e.g. 6,7]. The kinematic control neglects the dynamics effect and thus lowers the effective moving speed that could be obtained. Based on the Newton's second law, Tlale and de Villiers [8] developed and verified a dynamic model for the omni-directional robot with four Mecanum wheels using the resolved force method. In their developed mobile platform, each wheel was fitted with encoder for measuring the wheel rotation/velocity, and 3D gyrometer and accelerometer are also installed in the platform for measuring its orientation and translation motion. Vi-boonchaicheep *et al.* [9] presented a position rectification method including symptomatic and preventive rectifications during the position and orientation control. Their control system is based on kinematics and joint-space linear dynamics model. Recently, Han *et al.* [10], Park *et al.* [11], and Tsai and Wu [12] proposed fuzzy control systems for Mecanum wheeled robots based on kinematic model or joint-space dynamic model.

In this work, we will propose a more complete kinematics and dynamics modeling of an omni-directional mobile robot with four Mecanum wheels considering

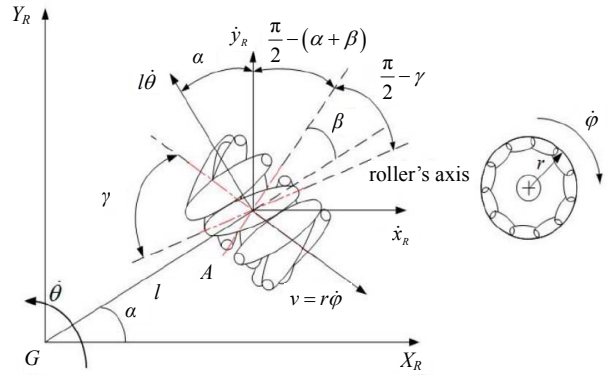
both the friction and load eccentricity effects. After establishing the kinematics model, the 3-DOF dynamics model in the Cartesian space is derived using the Lagrange's equation, which can be used for arbitrary translational and rotational dynamic control. Then an adaptive control is constructed using the backstepping method via Lyapunov stability theory. The derived adaptive controller with uncertainty compensator has excellent three-axis arbitrary trajectory tracking performance, even the platform is encountered a not small eccentricity uncertainty. Finally, simulation results are presented to illustrate the suggested control system performance. The suggested nonlinear controller is more complex than traditional PID control, however it could have more satisfactory and faster arbitrary-trajectory tracking capability.

## 2. Kinematics and Dynamics Modeling of a Mecanum-Wheeled Mobile Robot

As shown in **Figure 1**, a Swedish wheel consists of a fixed standard wheel with passive rollers attached to the wheel circumference. The Mecanum wheel is a type of Swedish wheels with  $\gamma = 45^\circ$ , where  $\gamma$  is the angle between the passive-roller rotation axis and the wheel plane. Complete kinematics and dynamics modeling of an omni-directional robot with four Mecanum wheels will be considered in this section.

### 2.1. Kinematics of a Four-Wheeled Mecanum Robot

Consider a Mecanum wheel mounted on a mobile robot with local coordinate frame  $\{R\}$ :  $X_R Y_R Z_R$ , as shown in **Figure 1**, where point  $A$  is its center and the other geometric parameters are defined as follows.  $\alpha$  is the angle of the vector  $\mathbf{GA}$ , from the robot frame origin  $G$  to the wheel center  $A$ , with respect to the  $X_R$  axis, and  $\beta$



**Figure 1. Parameters of a mecanum wheel.**

is the angle between the vector  $\mathbf{GA}$  and the main wheel axis. The distance from the geometric center  $G$  to the wheel center  $A$  is  $l$ , and the main wheel's radius is  $r$ . And  $\dot{\phi}$  and  $\dot{\phi}_{sw}$  are respectively the rotation speeds of the main wheel and the passive roller contacted with the flat floor.

Assume that the contact point between the Mecanum wheel and the floor is an instantaneous rotation center, that is, the contact is in a pure rolling condition without slipping, then the corresponding velocity of the wheel center  $A$  is  $r\dot{\phi}$  along the tangential direction as shown in **Figure 1**. So the wheel center  $A$ 's velocity component along the contact roller's axis is  $r\dot{\phi} \cos \gamma$ .

Let the robot's instantaneous translation velocity in terms of local frame  $\{R\}$  be  $[\dot{x}_R \ \dot{y}_R]^T$ , and the rotation velocity about  $Z_R$  axis be  $\dot{\theta}$ . Then the wheel center  $A$ 's velocity can also be computed by summing the translational velocity vectors  $\dot{x}_R, \dot{y}_R$ , and the relative velocity  $l\dot{\theta}$  due to the rotation velocity shown in **Figure 1**. Thus, the wheel center  $A$ 's velocity component along the contact roller's axis can be expressed as follows (computed via the platform's velocity):

$$\begin{aligned} & \dot{x}_R \cos \left[ \frac{\pi}{2} - \left( \frac{\pi}{2} - (\alpha + \beta) \right) - \left( \frac{\pi}{2} - \gamma \right) \right] + \dot{y}_R \cos \left[ \left( \frac{\pi}{2} - (\alpha + \beta) \right) + \left( \frac{\pi}{2} - \gamma \right) \right] + l\dot{\theta} \cos \left[ \alpha + \left( \frac{\pi}{2} - (\alpha + \beta) \right) + \left( \frac{\pi}{2} - \gamma \right) \right] \\ &= \dot{x}_R \cos \left[ (\alpha + \beta + \gamma) - \left( \frac{\pi}{2} \right) \right] + \dot{y}_R \cos [\pi - (\alpha + \beta + \gamma)] + l\dot{\theta} \cos [\pi - (\beta + \gamma)] \\ &= \dot{x}_R \sin(\alpha + \beta + \gamma) - \dot{y}_R \cos(\alpha + \beta + \gamma) - \dot{\theta} l \cos(\beta + \gamma) \\ &= [\sin(\alpha + \beta + \gamma) \quad -\cos(\alpha + \beta + \gamma) \quad -l \cos(\beta + \gamma)] \cdot [\dot{x}_R \quad \dot{y}_R \quad \dot{\theta}]^T \end{aligned} \quad (1)$$

If no slipping occurs along the contact roller's axis, the same velocity can also be computed from the wheel's

rotation speed  $\dot{\phi}$ . Hence, we have the following constraint equation for a Swedish wheel:

$$[\sin(\alpha + \beta + \gamma) \quad -\cos(\alpha + \beta + \gamma) \quad -l \cos(\beta + \gamma)] [\dot{x}_R \quad \dot{y}_R \quad \dot{\theta}]^T = r\dot{\phi} \cos \gamma \quad (2)$$

Since the rotation matrix representing the orientation

of the inertia frame  $\{I\}$  with respect to the robot frame

$\{R\}$  can be expressed as

$${}^R R_I(\theta) = \begin{bmatrix} \cos \theta & \sin \theta & 0 \\ -\sin \theta & \cos \theta & 0 \\ 0 & 0 & 1 \end{bmatrix},$$

where  $\theta$  is the angle between axes  $X_R$  and  $X_I$ , and the robot's velocity vector in terms of the robot frame  $\{R\}$ ,

$\dot{\xi}_R = [\dot{x}_R \quad \dot{y}_R \quad \dot{\theta}]^T$  can be computed as:

$$\dot{\xi}_R = {}^R R_I(\theta) \dot{\xi}_I,$$

where  $\dot{\xi}_I = [\dot{x}_I \quad \dot{y}_I \quad \dot{\theta}]^T$  is the robot velocity vector in terms of the inertia frame  $\{I\}$ , Equation (2) can be transformed to as follows [6]:

$$[\sin(\alpha + \beta + \gamma) \quad -\cos(\alpha + \beta + \gamma) \quad -l \cos(\beta + \gamma)] \cdot {}^R R_I(\theta) \dot{\xi}_I = r \dot{\phi} \cos \gamma \quad (3)$$

In the direction orthogonal to the contact roller's axis, the motion is not constrained because of the free rotation

of the passive contact roller, thus we have the following velocity relation:

$$\begin{aligned} & \dot{x}_R \sin \left[ \frac{\pi}{2} - \left( \frac{\pi}{2} - (\alpha + \beta) \right) - \left( \frac{\pi}{2} - \gamma \right) \right] - \dot{y}_R \sin \left[ \left( \frac{\pi}{2} - (\alpha + \beta) \right) + \left( \frac{\pi}{2} - \gamma \right) \right] - l \dot{\theta} \sin \left[ \alpha + \left( \frac{\pi}{2} - (\alpha + \beta) \right) + \left( \frac{\pi}{2} - \gamma \right) \right] \\ &= r \dot{\phi} \sin \gamma + r_{sw} \dot{\phi}_{sw} \end{aligned}$$

$$\therefore [\cos(\alpha + \beta + \gamma) \quad \sin(\alpha + \beta + \gamma) \quad l \sin(\beta + \gamma)] \cdot [\dot{x}_R \quad \dot{y}_R \quad \dot{\theta}]^T + r \dot{\phi} \sin \gamma + r_{sw} \dot{\phi}_{sw} = 0$$

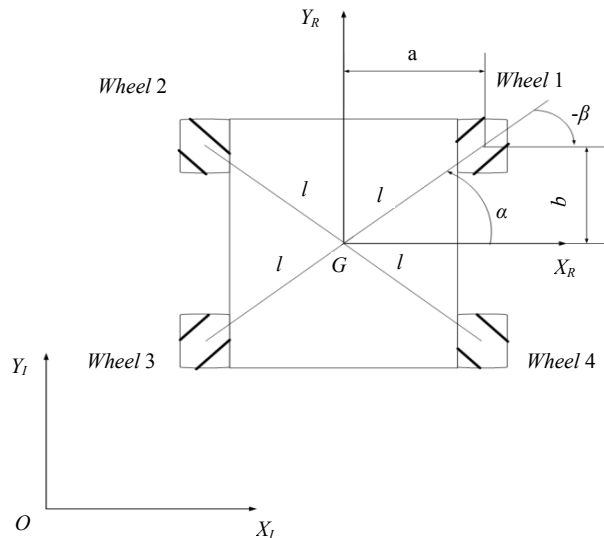
Thus, the above rolling condition can be transformed as:

$$[\cos(\alpha + \beta + \gamma) \quad \sin(\alpha + \beta + \gamma) \quad l \sin(\beta + \gamma)] {}^R R_I(\theta) \dot{\xi}_I + r \dot{\phi} \sin \gamma + r_{sw} \dot{\phi}_{sw} = 0 \quad (4)$$

Consider the omni-directional robot with four Mecanum wheels shown in **Figure 2**. The angles  $\alpha_i, \beta_i$ , and  $\gamma_i$  of the mounted four Mecanum wheels,  $i = 1, 2, 3, 4$ , are

shown in **Table 1**. From Equation (3), we have the following four constraint equations for the centers of the four Mecanum wheels:

$$\begin{bmatrix} \sin(\alpha_1 + \beta_1 + \gamma_1) & -\cos(\alpha_1 + \beta_1 + \gamma_1) & -l_1 \cos(\beta_1 + \gamma_1) \\ \sin(\alpha_2 + \beta_2 + \gamma_2) & -\cos(\alpha_2 + \beta_2 + \gamma_2) & -l_2 \cos(\beta_2 + \gamma_2) \\ \sin(\alpha_3 + \beta_3 + \gamma_3) & -\cos(\alpha_3 + \beta_3 + \gamma_3) & -l_3 \cos(\beta_3 + \gamma_3) \\ \sin(\alpha_4 + \beta_4 + \gamma_4) & -\cos(\alpha_4 + \beta_4 + \gamma_4) & -l_4 \cos(\beta_4 + \gamma_4) \end{bmatrix} \cdot {}^R R_I(\theta) \dot{\xi}_I = \begin{bmatrix} r_1 \dot{\phi}_1 \cos \gamma_1 \\ r_2 \dot{\phi}_2 \cos \gamma_2 \\ r_3 \dot{\phi}_3 \cos \gamma_3 \\ r_4 \dot{\phi}_4 \cos \gamma_4 \end{bmatrix} \quad (5)$$



**Figure 2. A four-Mecanum-wheeled robot.**

Assuming that each Mecanum wheel has equal radius and mounting distances,  $r_i = r, l_i = l, i = 1, 2, 3, 4$ , and by substituting the parameters in **Table 1** into Equation (5), we can obtain the inverse kinematics equation as follows:

$$\begin{bmatrix} \dot{\phi}_1 \\ \dot{\phi}_2 \\ \dot{\phi}_3 \\ \dot{\phi}_4 \end{bmatrix} = -(\sqrt{2}/r) \begin{bmatrix} \sqrt{2}/2 & \sqrt{2}/2 & l \sin(\pi/4 - \alpha) \\ \sqrt{2}/2 & -\sqrt{2}/2 & l \sin(\pi/4 - \alpha) \\ -\sqrt{2}/2 & -\sqrt{2}/2 & l \sin(\pi/4 - \alpha) \\ -\sqrt{2}/2 & \sqrt{2}/2 & l \sin(\pi/4 - \alpha) \end{bmatrix} \begin{bmatrix} \dot{x}_I \\ \dot{y}_I \\ \dot{\theta} \end{bmatrix} \quad (6)$$

where  $\alpha = \tan^{-1}(b/a)$ .

Define the Jacobian matrix as:

**Table 1. Parameters of the Mecanum wheels.**

Wheels	$\alpha_i$	$\beta_i$	$\gamma_i$
1	$\tan^{-1}(b/a)$	$-\tan^{-1}(b/a)$	$(\pi/2 + \pi/4)$
2	$\pi - \tan^{-1}(b/a)$	$\tan^{-1}(b/a)$	$-(\pi/2 + \pi/4)$
3	$\pi + \tan^{-1}(b/a)$	$-\tan^{-1}(b/a)$	$(\pi/2 + \pi/4)$
4	$2\pi - \tan^{-1}(b/a)$	$\tan^{-1}(b/a)$	$-(\pi/2 + \pi/4)$

$$\mathbf{J} = \begin{bmatrix} \sqrt{2}/2 & \sqrt{2}/2 & l \sin(\pi/4 - \alpha) \\ \sqrt{2}/2 & -\sqrt{2}/2 & l \sin(\pi/4 - \alpha) \\ -\sqrt{2}/2 & -\sqrt{2}/2 & l \sin(\pi/4 - \alpha) \\ -\sqrt{2}/2 & \sqrt{2}/2 & l \sin(\pi/4 - \alpha) \end{bmatrix} \cdot \begin{bmatrix} \cos \theta & \sin \theta & 0 \\ -\sin \theta & \cos \theta & 0 \\ 0 & 0 & 1 \end{bmatrix} \quad (7)$$

$${}^R \mathbf{v}_G = \begin{bmatrix} \cos \theta & \sin \theta \\ -\sin \theta & \cos \theta \end{bmatrix} \begin{bmatrix} \dot{x}_I \\ \dot{y}_I \end{bmatrix} = [\dot{x}_I \cos \theta + \dot{y}_I \sin \theta \quad -\dot{x}_I \sin \theta + \dot{y}_I \cos \theta]^T \quad (9)$$

where  $\dot{x}_I$  and  $\dot{y}_I$  are the velocity components of  $G$  along the  $X_I$ - and  $Y_I$ -axes, respectively, and  $\theta$  is the orientation angle of the platform relative to the reference

The forward kinematics equation of the four-wheeled Mecanum mobile robot can be obtained as follows:

$$\begin{bmatrix} \dot{x}_I \\ \dot{y}_I \\ \dot{\theta} \end{bmatrix} = -(\sqrt{2}/2) r \mathbf{J}^+ \begin{bmatrix} \dot{\phi}_1 \\ \dot{\phi}_2 \\ \dot{\phi}_3 \\ \dot{\phi}_4 \end{bmatrix} \quad (8)$$

where  $\mathbf{J}^+ = (\mathbf{J}^T \mathbf{J})^{-1} \mathbf{J}^T$  is the pseudoinverse of  $\mathbf{J}$ .

## 2.2. Dynamics of the Mecanum Robot

Consider the four-wheeled Mecanum mobile robot shown in **Figure 3**, where  $G$  is the geometric center with position vector  ${}^I \mathbf{r}_G = [x_I \quad y_I]^T$  in terms of the inertia frame  $\{I\}$ , and  $G'$  is the mass center of the moving platform with relative position vector  ${}^R \mathbf{r}_{G'/G} = [-d_1 \quad -d_2]^T$  in terms of robot frame  $\{R\}$ . The velocity  ${}^R \mathbf{v}_G$  of point  $G$ , in terms of robot frame  $\{R\}$  can be expressed as

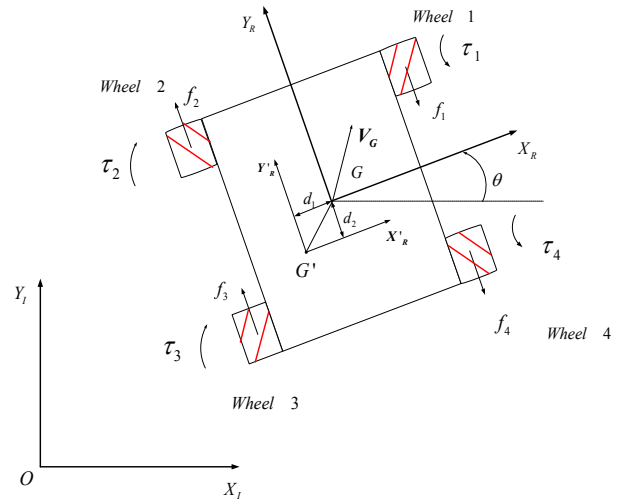
$$\begin{aligned} {}^R \mathbf{v}_{G'} &= {}^R \mathbf{v}_G + \dot{\theta} \mathbf{k}_R \times {}^R \mathbf{r}_{G'/G} \\ &= (\dot{x}_I \cos \theta + \dot{y}_I \sin \theta + \dot{\theta} d_2) \mathbf{i}_R + (-\dot{x}_I \sin \theta + \dot{y}_I \cos \theta - \dot{\theta} d_1) \mathbf{j}_R \end{aligned} \quad (10)$$

The total kinetic energy  $T$  of the mobile robot including those of the platform and four Mecanum wheels can be computed as below:

$$T = \frac{1}{2} \left[ m_b \mathbf{v}_{G'}^T \mathbf{v}_{G'} + I_b \dot{\theta}^2 + \sum_{i=1}^4 m_{wi} (r \dot{\phi}_i)^2 + \sum_{i=1}^4 I_i \dot{\phi}_i^2 \right] \quad (11)$$

where  $m_b$  is the mass of the platform, and  $m_{wi}$  is the mass of the  $i$ th wheel,  $i = 1, 2, 3, 4$ ;  $I_b$  is the moment of inertia of the platform about  $Z'_R$  axis (parallel to  $Z_R$ ) through point  $G'$ , and  $I_i$  is the moment of inertia of the  $i$ th wheel about its main axis;  $\dot{\theta}$  is the rotational speed of the platform, and  $\dot{\phi}_i$  is the rotational speed of the  $i$ th wheel about its main axis; and  $r$  is the radius of each Mecanum wheel. Since the mobile robot is assumed moving in a plane, the total potential energy  $V = 0$ . Assume that the four Mecanum wheels are identical and thus let  $m_{wi} = m_w$ , and  $I_i = I$ ,  $i = 1, 2, 3, 4$ . After substituting Equation (2) and some computations, the La-

grangian  $L = T - V = T$  can be obtained as below:

**Figure 3. Schematic of the Mecanum robot.**

$$\begin{aligned}
L = & \frac{1}{2} I_b \dot{\theta}^2 + \frac{1}{2} m_b \left[ \left( \dot{x}_l \cos \theta + \dot{y}_l \sin \theta + \dot{\theta} d_2 \right)^2 + \left( -\dot{x}_l \sin \theta + \dot{y}_l \cos \theta - \dot{\theta} d_1 \right)^2 \right] \\
& + \frac{1}{2} m_w \left\{ \left[ -\dot{x}_l (\cos \theta - \sin \theta) - \dot{y}_l (\sin \theta + \cos \theta) - \sqrt{2} \dot{\theta} l \sin(\pi/4 - \alpha) \right]^2 \right. \\
& + \left[ -\dot{x}_l (\cos \theta + \sin \theta) - \dot{y}_l (\sin \theta - \cos \theta) - \sqrt{2} \dot{\theta} l \sin(\pi/4 - \alpha) \right]^2 \\
& + \left[ -\dot{x}_l (-\cos \theta + \sin \theta) + \dot{y}_l (\sin \theta + \cos \theta) - \sqrt{2} \dot{\theta} l \sin(\pi/4 - \alpha) \right]^2 \\
& + \left. \left[ \dot{x}_l (\cos \theta + \sin \theta) - \dot{y}_l (-\sin \theta + \cos \theta) - \sqrt{2} \dot{\theta} l \sin(\pi/4 - \alpha) \right]^2 \right\} \\
& + \frac{1}{2} \frac{1}{r^2} I_1 \left[ -\dot{x}_l (\cos \theta - \sin \theta) - \dot{y}_l (\sin \theta + \cos \theta) - \sqrt{2} \dot{\theta} l \sin(\pi/4 - \alpha) \right]^2 \\
& + \frac{1}{2} \frac{1}{r^2} I_2 \left[ -\dot{x}_l (\cos \theta + \sin \theta) - \dot{y}_l (\sin \theta - \cos \theta) - \sqrt{2} \dot{\theta} l \sin(\pi/4 - \alpha) \right]^2 \\
& + \frac{1}{2} \frac{1}{r^2} I_3 \left[ -\dot{x}_l (-\cos \theta + \sin \theta) + \dot{y}_l (\sin \theta + \cos \theta) - \sqrt{2} \dot{\theta} l \sin(\pi/4 - \alpha) \right]^2 \\
& + \frac{1}{2} \frac{1}{r^2} I_4 \left[ \dot{x}_l (\cos \theta + \sin \theta) - \dot{y}_l (-\sin \theta + \cos \theta) - \sqrt{2} \dot{\theta} l \sin(\pi/4 - \alpha) \right]^2
\end{aligned} \tag{12}$$

The dynamics model can then be derived using the Lagrange's equations:

$$\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{q}_i} \right) - \frac{\partial L}{\partial q_i} = F_i, \quad i = 1, 2, 3 \tag{13}$$

where  $q_i$  is the  $i$ th generalized coordinate, and  $F_i$  is

the  $i$ th generalized force/torque. The generalized coordinate vector of the mobile robot can be defined as:  $\mathbf{q} = [q_1 \ q_2 \ q_3]^T = [x_l \ y_l \ \theta]^T$ . Refer to **Figure 3**, where  $f_i$  is the contact friction force of the  $i$ th Mecanum wheel with the floor, the generalized force/torque  $F_i$ ,  $i = 1, 2, 3$ , can be derived as follows [13]:

$$F_i = \sum_{i=1}^4 (\tau_i - r \operatorname{sgn}(\dot{\phi}_i) f_i) \frac{\partial \phi_i}{\partial x_l} = \sum_{i=1}^4 (\tau_i - r \operatorname{sgn}(\dot{\phi}_i) f_i) \frac{\partial \phi_i}{\partial \dot{x}_l} \tag{14}$$

By Equation (6), we can obtain

$$\frac{\partial \phi_1}{\partial \dot{x}_l} = -\frac{1}{r} (\cos \theta - \sin \theta); \quad \frac{\partial \phi_2}{\partial \dot{x}_l} = -\frac{1}{r} (\cos \theta + \sin \theta); \quad \frac{\partial \phi_3}{\partial \dot{x}_l} = \frac{1}{r} (\cos \theta - \sin \theta); \quad \frac{\partial \phi_4}{\partial \dot{x}_l} = \frac{1}{r} (\cos \theta + \sin \theta).$$

Thus,

$$\begin{aligned}
F_1 = & \left[ \tau_1 - r \operatorname{sgn}(\dot{\phi}_1) f_1 \right] \left[ -\frac{1}{r} (\cos \theta - \sin \theta) \right] + \left[ \tau_2 - r \operatorname{sgn}(\dot{\phi}_2) f_2 \right] \left[ -\frac{1}{r} (\cos \theta + \sin \theta) \right] \\
& + \left[ \tau_3 - r \operatorname{sgn}(\dot{\phi}_3) f_3 \right] \left[ \frac{1}{r} (\cos \theta - \sin \theta) \right] + \left[ \tau_4 - r \operatorname{sgn}(\dot{\phi}_4) f_4 \right] \left[ \frac{1}{r} (\cos \theta + \sin \theta) \right]
\end{aligned} \tag{15}$$

Similarly,

$$\begin{aligned}
F_2 = & \sum_{i=1}^4 (\tau_i - r \operatorname{sgn}(\dot{\phi}_i) f_i) \frac{\partial \phi_i}{\partial \dot{y}_l} = \left[ \tau_1 - r \operatorname{sgn}(\dot{\phi}_1) f_1 \right] \left[ -\frac{1}{r} (\sin \theta + \cos \theta) \right] \\
& + \left[ \tau_2 - r \operatorname{sgn}(\dot{\phi}_2) f_2 \right] \left[ -\frac{1}{r} (\sin \theta - \cos \theta) \right] + \left[ \tau_3 - r \operatorname{sgn}(\dot{\phi}_3) f_3 \right] \left[ \frac{1}{r} (\sin \theta + \cos \theta) \right] \\
& + \left[ \tau_4 - r \operatorname{sgn}(\dot{\phi}_4) f_4 \right] \left[ \frac{1}{r} (\sin \theta - \cos \theta) \right]
\end{aligned} \tag{16}$$

$$F_3 = \sum_{i=1}^4 (\tau_i - r \operatorname{sgn}(\dot{\phi}_i) f_i) \frac{\partial \dot{\phi}_i}{\partial \dot{\theta}} = (\tau_1 + \tau_2 + \tau_3 + \tau_4) \left[ -\frac{\sqrt{2}}{r} l \sin(\pi/4 - \alpha) \right] \\ + [\operatorname{sgn}(\dot{\phi}_1) f_1 + \operatorname{sgn}(\dot{\phi}_2) f_2 + \operatorname{sgn}(\dot{\phi}_3) f_3 + \operatorname{sgn}(\dot{\phi}_4) f_4] \left[ \sqrt{2} l \sin(\pi/4 - \alpha) \right] \quad (17)$$

After some straightforward computations, the equations of motion of the mobile robot can be expressed in matrix/vector form as:

$$\mathbf{M}(\mathbf{q})\ddot{\mathbf{q}} + \mathbf{C}(\mathbf{q}, \dot{\mathbf{q}})\dot{\mathbf{q}} + \mathbf{B}^T \mathbf{S} \mathbf{f} = \frac{1}{r} \mathbf{B}^T \boldsymbol{\tau} \quad (18)$$

where

$$\boldsymbol{\tau} = [\tau_1 \quad \tau_2 \quad \tau_3 \quad \tau_4]^T, \mathbf{f} = [f_1 \quad f_2 \quad f_3 \quad f_4]^T,$$

$$\mathbf{S} = \operatorname{diag}[\operatorname{sgn}(\dot{\phi}_1) \quad \operatorname{sgn}(\dot{\phi}_2) \quad \operatorname{sgn}(\dot{\phi}_3) \quad \operatorname{sgn}(\dot{\phi}_4)],$$

$$\mathbf{M} = [m_{ij}]_{3 \times 3}, m_{11} = m_b + 4 \left( m_w + \frac{I}{r^2} \right);$$

$$m_{22} = m_b + 4 \left( m_w + \frac{I}{r^2} \right); m_{12} = m_{21} = 0;$$

$$m_{13} = m_{31} = m_b (d_1 \sin \theta + d_2 \cos \theta);$$

$$m_{23} = m_{32} = m_b (-d_1 \cos \theta + d_2 \sin \theta);$$

$$m_{33} = m_b (d_1^2 + d_2^2) + I_b + 8 \left( m_w + \frac{I}{r^2} \right) l^2 \sin^2(\pi/4 - \alpha);$$

$$\mathbf{C} = \begin{bmatrix} 0 & 0 & m_b \dot{\theta} (d_1 \cos \theta - d_2 \sin \theta) \\ 0 & 0 & m_b \dot{\theta} (d_1 \sin \theta + d_2 \cos \theta) \\ 0 & 0 & 0 \end{bmatrix};$$

$$\mathbf{B} = \begin{bmatrix} -(\cos \theta - \sin \theta) & -(\sin \theta + \cos \theta) & -\sqrt{2} l \sin(\pi/4 - \alpha) \\ -(\cos \theta + \sin \theta) & -(\sin \theta - \cos \theta) & -\sqrt{2} l \sin(\pi/4 - \alpha) \\ \cos \theta - \sin \theta & \sin \theta + \cos \theta & -\sqrt{2} l \sin(\pi/4 - \alpha) \\ \cos \theta + \sin \theta & \sin \theta - \cos \theta & -\sqrt{2} l \sin(\pi/4 - \alpha) \end{bmatrix}.$$

### 3. Stable Adaptive Control of a Mecanum-Wheeled Robot

#### 3.1. Modeling Uncertainty

In practice, the mass of the platform carrying payload, and the contact friction forces may be varied, thus we can model their uncertainty by letting  $m_b = \hat{m}_b + \Delta m_b$ , and  $\mathbf{f} = \hat{\mathbf{f}} + \Delta \mathbf{f}$ , where  $\hat{m}_b$  and

$$\hat{\mathbf{f}} = \frac{1}{4} (\hat{m}_b + 4m_w) \cdot \mathbf{g} [\hat{u}_{rr,1} \quad \hat{u}_{rr,2} \quad \hat{u}_{rr,3} \quad \hat{u}_{rr,4}]^T \quad (19)$$

are the nominal platform mass and friction vector, respectively. Here  $\mathbf{g}$  is the gravitational constant, and  $\hat{u}_{rr,i}$ ,  $i = 1, 2, 3, 4$ , are the nominal rolling friction coefficients of the four wheels. Substituting into Equation (18), the dynamics of the mobile robot considering uncertainty can be summarized as follows:

$$\hat{\mathbf{M}} \ddot{\mathbf{q}} = \frac{1}{r} \mathbf{B}^T \boldsymbol{\tau} - \mathbf{B}^T \mathbf{S} \hat{\mathbf{f}} - \mathbf{D}_1 - \mathbf{D}_2 \quad (20)$$

where

$$\hat{\mathbf{M}} = \operatorname{diag} \left[ \hat{m}_b + 4 \left( m_w + \frac{I}{r^2} \right), \hat{m}_b + 4 \left( m_w + \frac{I}{r^2} \right), \right. \\ \left. I_b + 8l^2 \sin^2(\pi/4 - \alpha) \left( m_w + \frac{I}{r^2} \right) \right],$$

$$\mathbf{H}_m = \begin{bmatrix} \ddot{x}_l & \ddot{\theta} \sin \theta + \dot{\theta}^2 \cos \theta & \ddot{\theta} \cos \theta - \dot{\theta}^2 \sin \theta \\ \ddot{y}_l & -\ddot{\theta} \cos \theta + \dot{\theta}^2 \sin \theta & \ddot{\theta} \sin \theta + \dot{\theta}^2 \cos \theta \\ 0 & \ddot{x}_l \sin \theta - \ddot{y}_l \cos \theta & \ddot{x}_l \cos \theta + \ddot{y}_l \sin \theta \end{bmatrix},$$

$$\mathbf{w}_m = [\Delta m_b \quad m_b d_1 \quad m_b d_2]^T,$$

$$\mathbf{D}_1 = \mathbf{H}_m \mathbf{w}_m, \mathbf{D}_2 = \mathbf{B}^T \mathbf{S} \Delta \mathbf{f}.$$

#### 3.2. Stable Adaptive Control of a Mecanum-Wheeled Robot

##### 3.2.1. Nominal Control Law Derivation

In order to synthesize the adaptive control law, we can first neglect the uncertainty effect, *i.e.*, let  $\mathbf{D}_1 = \mathbf{D}_2 = 0$  in Equation (20), and consider the following nominal dynamics model:

$$\hat{\mathbf{M}} \ddot{\mathbf{q}} = \frac{1}{r} \mathbf{B}^T \boldsymbol{\tau} - \mathbf{B}^T \mathbf{S} \hat{\mathbf{f}} \quad (21)$$

Defining the state vector as:

$$\mathbf{z} = [\mathbf{z}_1^T \quad \mathbf{z}_2^T]^T = [\mathbf{q}^T \quad \dot{\mathbf{q}}^T]^T \\ = [x_l \quad y_l \quad \theta \quad \dot{x}_l \quad \dot{y}_l \quad \dot{\theta}]^T$$

the state equation of the nominal system can be written as

$$\begin{cases} \dot{\mathbf{z}}_1 = \mathbf{z}_2 \\ \dot{\mathbf{z}}_2 = \hat{\mathbf{M}}^{-1} \left( \frac{1}{r} \mathbf{B}^T \boldsymbol{\tau} - \mathbf{B}^T \mathbf{S} \hat{\mathbf{f}} \right) \end{cases} \quad (22)$$

Based on the backstepping method, a stable nonlinear nominal control law can be obtained as follows.

First consider the  $z_1$  subsystem,  $\dot{z}_1 = z_2$ . Let

$$\dot{z}_1 = v_1 \quad (23)$$

where  $v_1$  is a virtual input. Define the tracking error vector as

$$e_1 = z_1 - q_d \quad (24)$$

where  $q_d(t) = [x_{I,d}(t), y_{I,d}(t), \theta_d(t)]^T$  is the desired trajectory vector for the platform. Differentiating Equation (24), we have

$$\dot{e}_1 = \dot{z}_1 - \dot{q}_d = v_1 - \dot{q}_d \quad (25)$$

Considering the Lyapunov function candidate

$$V_1 = \frac{1}{2} e_1^T K_1 e_1 \quad (26)$$

where  $K_1 \in \mathbb{R}^{3 \times 3}$  is symmetric and positive definite, and differentiating Equation (26), we have

$$\begin{aligned} \dot{V}_1 &= \frac{1}{2} \dot{e}_1^T K_1 e_1 + \frac{1}{2} e_1^T K_1 \dot{e}_1 = e_1^T K_1 \dot{e}_1 \\ &= e_1^T K_1 (v_1 - \dot{q}_d) \end{aligned} \quad (27)$$

Thus, we can choose

$$v_1 = \dot{q}_d - e_1 \quad (28)$$

and obtain

$$\dot{V}_1 = -e_1^T K_1 e_1 \leq 0 \quad (29)$$

By Equation (29), we know that  $\lim_{t \rightarrow \infty} e_1(t) = 0$ , that is, the subsystem is asymptotically stable.

Further, the whole nonlinear system (22) is considered. After introducing new error vector

$$e_2 = z_2 - v_1 \quad (30)$$

we can obtain

$$\begin{aligned} \dot{e}_1 &= \dot{z}_1 - \dot{q}_d = z_2 - \dot{q}_d - v_1 + v_1 \\ &= e_2 + v_1 - \dot{q}_d = e_2 - e_1 \\ \dot{e}_2 &= \dot{z}_2 - \dot{v}_1 = \dot{z}_2 - \ddot{q}_d + \dot{e}_1 \\ &= \hat{M}^{-1} \left( \frac{1}{r} B^T \tau - B^T S \hat{f} \right) - \ddot{q}_d + (e_2 - e_1) \end{aligned} \quad (31)$$

Then by considering the Lyapunov function candidate as

$$\begin{aligned} V_2 &= V_1 + \frac{1}{2} e_2^T K_2 e_2 = \frac{1}{2} e_1^T K_1 e_1 + \frac{1}{2} e_2^T K_2 e_2 \\ &= \frac{1}{2} e^T K_a e \end{aligned} \quad (32)$$

where  $K_2 \in \mathbb{R}^{3 \times 3}$  is symmetric and positive definite,  $K_a = \text{diag}[K_1 \quad K_2]$ ,  $e = [e_1^T \quad e_2^T]^T$ , and taking the time

derivative of  $V_2$ , we have

$$\begin{aligned} \dot{V}_2 &= e_1^T K_1 \dot{e}_1 + e_2^T K_2 \dot{e}_2 = e_1^T K_1 (e_2 - e_1) \\ &+ e_2^T K_2 \left[ \frac{1}{r} \hat{M}^{-1} B^T \tau - \hat{M}^{-1} B^T S \hat{f} - \ddot{q}_d + (e_2 - e_1) \right] \end{aligned} \quad (33)$$

Thus, we can consider the nominal control law as

$$\begin{aligned} \tau &= \tau_n = r B (B^T B)^{-1} \hat{M} \\ &\cdot [\ddot{q}_d + \hat{M}^{-1} B^T S \hat{f} - e_2 - (e_2 - e_1) - K_2^{-1} K_1 e_1] \end{aligned} \quad (34)$$

and obtain

$$\dot{V}_2 = -e_1^T K_1 e_1 - e_2^T K_2 e_2 = -e^T K_a e = -2V_2 \leq 0 \quad (35)$$

Since  $\dot{V}_2(t)$  is negative definite, we know that the equilibrium point  $e = 0$  is exponentially stable.

### 3.2.2. Adaptive Control of a Mecanum-Wheeled Robot with Uncertainty

Consider the Mecanum-wheeled robot dynamics model with uncertainty  $D_1, D_2$ :

$$\hat{M} \ddot{q} = \frac{1}{r} B^T \tau - B^T S \hat{f} - D_1 - D_2 \quad (20)$$

Using the error vectors defined before in the nominal control design,

$$\begin{aligned} e_1 &= z_1 - q_d = q - q_d \\ e_2 &= z_2 - v_1 = \dot{q} - v_1 = \dot{q} - \dot{q}_d + e_1 \end{aligned} \quad (36)$$

the system's error dynamics can be obtained by direct differentiating Equation (36) as below:

$$\dot{e} = \alpha(t, q, \dot{q}) + \beta(q) \left( \frac{1}{r} B^T \tau - D_1 - D_2 \right) \quad (37)$$

where

$$\begin{aligned} \alpha(t, q, \dot{q}) &= \begin{bmatrix} e_2 - e_1 \\ -\hat{M}^{-1} B^T S \hat{f} - \ddot{q}_d + (e_2 - e_1) \end{bmatrix}, \\ \beta(q) &= \begin{bmatrix} 0 \\ \hat{M}^{-1} \end{bmatrix}. \end{aligned}$$

Consider the following Lyapunov function for adaptive control design,

$$V_a = V_2 + \frac{1}{2} \tilde{w}_m^T \Gamma^{-1} \tilde{w}_m \quad (38)$$

where  $\tilde{w}_m = \hat{w}_m - w_m$  and  $\Gamma$  is a symmetric and positive definite matrix. And taking the time derivative of  $V_a$ , we have

$$\begin{aligned} \dot{V}_a &= \frac{\partial V_2}{\partial e} \left[ \alpha(t, q, \dot{q}) + \beta(q) \left( \frac{1}{r} B^T \tau - D_1 - D_2 \right) \right] \\ &+ \tilde{w}_m^T \Gamma^{-1} \dot{\tilde{w}}_m \end{aligned} \quad (39)$$

By the definition of  $\mathbf{D}_2$  in Equation (20), we know that it is bounded, i.e.,  $\|\mathbf{D}_2\| \leq \rho_2$ ,  $\rho_2 > 0$ . Since  $\mathbf{D}_1 = \mathbf{H}_m \mathbf{w}_m$  is in linear parametrized form, we can introduce a compensating term  $\boldsymbol{\tau}_d$  and choose the control law as

$$\boldsymbol{\tau} = \boldsymbol{\tau}_n + \boldsymbol{\tau}_d \quad (40)$$

with

$$\boldsymbol{\tau}_d = -r\mathbf{B}(\mathbf{B}^T \mathbf{B})^{-1} \rho_2 \frac{\left(\frac{\partial V_2}{\partial \mathbf{e}} \boldsymbol{\beta}(q)\right)^T}{\left|\frac{\partial V_2}{\partial \mathbf{e}} \boldsymbol{\beta}(q)\right| + c_1} + r\mathbf{B}(\mathbf{B}^T \mathbf{B})^{-1} \mathbf{H}_m \hat{\mathbf{w}}_m \quad (41)$$

where  $c_1 > 0$ . Substituting Equations (40) and (41) into Equation (39), we have

$$\begin{aligned} \dot{V}_a &= \frac{\partial V_2}{\partial \mathbf{e}} \left[ \boldsymbol{\alpha}(t, q, \dot{q}) + \boldsymbol{\beta}(q) \left( \frac{1}{r} \mathbf{B}^T \boldsymbol{\tau}_n \right) \right] + \frac{\partial V_2}{\partial \mathbf{e}} \boldsymbol{\beta}(q) \left[ \frac{1}{r} \mathbf{B}^T \boldsymbol{\tau}_d - \mathbf{D}_1 - \mathbf{D}_2 \right] + \tilde{\mathbf{w}}_m^T \Gamma^{-1} \dot{\tilde{\mathbf{w}}}_m \\ &= -2V_2 + \tilde{\mathbf{w}}_m^T \Gamma^{-1} \dot{\tilde{\mathbf{w}}}_m + \frac{\partial V_2}{\partial \mathbf{e}} \boldsymbol{\beta}(q) \left[ -\rho_2 \frac{\left(\frac{\partial V_2}{\partial \mathbf{e}} \boldsymbol{\beta}(q)\right)^T}{\left|\frac{\partial V_2}{\partial \mathbf{e}} \boldsymbol{\beta}(q)\right| + c_1} + \mathbf{H}_m \hat{\mathbf{w}}_m - \mathbf{H}_m \mathbf{w}_m - \mathbf{D}_2 \right] \\ &= -2V_2 + \frac{\partial V_2}{\partial \mathbf{e}} \boldsymbol{\beta}(q) \left[ -\rho_2 \frac{\left(\frac{\partial V_2}{\partial \mathbf{e}} \boldsymbol{\beta}(q)\right)^T}{\left|\frac{\partial V_2}{\partial \mathbf{e}} \boldsymbol{\beta}(q)\right| + c_1} - \mathbf{D}_2 \right] + \frac{\partial V_2}{\partial \mathbf{e}} \boldsymbol{\beta}(q) \mathbf{H}_m \tilde{\mathbf{w}}_m + \tilde{\mathbf{w}}_m^T \Gamma^{-1} \dot{\tilde{\mathbf{w}}}_m \end{aligned} \quad (42)$$

Choose the parameter adaptation law as

$$\dot{\tilde{\mathbf{w}}}_m = \dot{\hat{\mathbf{w}}}_m = -\Gamma \left[ \left( \frac{\partial V_2}{\partial \mathbf{e}} \boldsymbol{\beta}(q) \mathbf{H}_m \right)^T + \sigma (\hat{\mathbf{w}}_m - \mathbf{w}_m^0) \right] \quad (43)$$

where  $\sigma > 0$ , and  $\mathbf{w}_m^0$  is the best guess for the unknown parameter vector  $\mathbf{w}_m$ . Since

$$\begin{aligned} \frac{\partial V_2}{\partial \mathbf{e}} \boldsymbol{\beta}(q) \left[ -\rho_2 \frac{\left(\frac{\partial V_2}{\partial \mathbf{e}} \boldsymbol{\beta}(q)\right)^T}{\left|\frac{\partial V_2}{\partial \mathbf{e}} \boldsymbol{\beta}(q)\right| + c_1} - \mathbf{D}_2 \right] &= \frac{\partial V_2}{\partial \mathbf{e}} \boldsymbol{\beta}(q) \left[ -\rho_2 \frac{\left(\frac{\partial V_2}{\partial \mathbf{e}} \boldsymbol{\beta}(q)\right)^T}{\left|\frac{\partial V_2}{\partial \mathbf{e}} \boldsymbol{\beta}(q)\right| + c_1} - \mathbf{D}_2 \frac{\left|\frac{\partial V_2}{\partial \mathbf{e}} \boldsymbol{\beta}(q)\right| + c_1}{\left|\frac{\partial V_2}{\partial \mathbf{e}} \boldsymbol{\beta}(q)\right| + c_1} \right] \\ &\leq \frac{\left|\frac{\partial V_2}{\partial \mathbf{e}} \boldsymbol{\beta}(q)\right|}{\left|\frac{\partial V_2}{\partial \mathbf{e}} \boldsymbol{\beta}(q)\right| + c_1} \left[ -\rho_2 \left|\frac{\partial V_2}{\partial \mathbf{e}} \boldsymbol{\beta}(q)\right| + \rho_2 \left( \left|\frac{\partial V_2}{\partial \mathbf{e}} \boldsymbol{\beta}(q)\right| + c_1 \right) \right] = \rho_2 c_1 \frac{\left|\frac{\partial V_2}{\partial \mathbf{e}} \boldsymbol{\beta}(q)\right|}{\left|\frac{\partial V_2}{\partial \mathbf{e}} \boldsymbol{\beta}(q)\right| + c_1} \leq \rho_2 c_1 \end{aligned}$$

we have

$$\dot{V}_a \leq -2V_2 - \sigma \frac{\|\tilde{\mathbf{w}}_m\|^2}{2} + d \quad (44)$$

where

$$d \triangleq \rho_2 c_1 + \sigma \frac{\|\mathbf{w}_m - \mathbf{w}_m^0\|^2}{2} \quad (45)$$

Since  $V_2(\mathbf{e})$  satisfies the following inequality equation [14],

$$\gamma_{e_1}(|\mathbf{e}|) \leq V_2(\mathbf{e}) \leq \gamma_{e_2}(|\mathbf{e}|) \quad (46)$$

where we can choose  $\gamma_{e_1}(|\mathbf{e}|) = \gamma_{e_2}(|\mathbf{e}|) = \frac{1}{2} \mathbf{e}^T \mathbf{K}_a \mathbf{e}$ , thus

$\gamma_{e_1}$  and  $\gamma_{e_2}$  are class- $K_\infty$  function, and  $-V_2(\mathbf{e}) \leq -\gamma_{e_1}(|\mathbf{e}|)$ , we can obtain

$$\dot{V}_a \leq -2\gamma_{e1}(|e|) - \sigma \frac{|\tilde{w}_m|^2}{2} + d \quad (47)$$

Let  $-2\gamma_{e1}(|e|) + d \leq 0$ , then  $\gamma_{e1}(|e|) \geq d/2$ , and thus  $|e| \geq \gamma_{e1}^{-1}(d/2) \triangleq b_e$ . Similarly, while  $-\sigma \frac{|\tilde{w}_m|^2}{2} + d \leq 0$ ,

we have  $|\tilde{w}_m| \geq \sqrt{\frac{2d}{\sigma}} \triangleq b_{\tilde{w}}$ . Hence, if  $|e| \geq b_e$  or  $|\tilde{w}_m| \geq b_{\tilde{w}}$ , then

$$\dot{V}_a \leq 0 \quad (48)$$

Since

$$\begin{aligned} V_a &= \frac{1}{2} e^T K_a e + \frac{1}{2} \tilde{w}_m^T \Gamma^{-1} \tilde{w}_m \\ &\leq \gamma_{e2}(|e|) + \frac{1}{2} \lambda_{\max}(\Gamma^{-1}) |\tilde{w}_m|^2 \end{aligned} \quad (49)$$

by substituting  $|e| = b_e$  and  $|\tilde{w}_m| = b_{\tilde{w}}$  into the right side of Equation (49), we can define

$$V_r = \gamma_{e2} \circ \gamma_{e1}^{-1} \left( \frac{d}{2} \right) + \frac{d \lambda_{\max}(\Gamma^{-1})}{\sigma} \quad (50)$$

and thus,

$$\gamma_{e1}(|e|) \leq V_a \leq \max(V_a(0), V_r)$$

Hence, we have

$$|e| \leq \gamma_{e1}^{-1}(\max(V_a(0), V_r)) \quad (51)$$

Define

$$B_e = \{e \in R^6 : |e| \leq \gamma_{e1}^{-1}(\max(V_a(0), V_r))\} \quad (52)$$

and we thus have  $e(t) \in B_e, \forall t$ .

By properly choosing the parameters:  $\rho_2, c_1, \sigma, \Gamma, K_1$ , and  $K_2$ , constants  $b_e$  and  $b_{\tilde{w}}$  can be made sufficiently small, and the norm of the tracking error  $|e|$  is bounded. And thus the adaptive control system is stable.

## 4. Results and Discussion

In this section, two computer simulation examples are given to illustrate the performance of the proposed adaptive control for the Mecanum-wheeled mobile robot.

The first considers a pure translation along a rectangular desired trajectory in the  $X_I - Y_I$  plane with fixed orientation  $\theta_d(t) = 0$ . The platform's geometric center is planned to move forward from the origin of the inertia frame along  $X_I$  axis 1 m, then leftward along  $Y_I$  axis 1 m, and then backward along  $X_I$  axis 1m, and finally move rightward along  $Y_I$  axis 1 m and return to the origin. The desired  $x_{I,d}(t)$  and  $y_{I,d}(t)$  are obtained using the cubic spline method [15] and shown as the dashed lines in **Figure 4(a)**.

The parameters of the mobile robot are selected as follows:  $m_b = 12$  kg,  $I = 0.5$  kg·m<sup>2</sup>,  $I_w = 4.0378 \times 10^{-4}$  kg·m<sup>2</sup>,  $a = 0.2$  m,  $b = 0.3$  m,  $l = 0.25$  m,  $m_w = 0.313$  kg,  $r = 0.0508$  m,  $\hat{u}_{rr} = 0.25$ , and  $g = 9.8$  m/s<sup>2</sup>. And the adaptive controller parameters are chosen as:

$$K_1 = \text{diag}[9000 \quad 9000 \quad 100],$$

$$K_2 = \text{diag}[2000 \quad 1800 \quad 3.25],$$

$$\rho = 0.002, \quad c_1 = 0.0001,$$

$$\sigma = 0.002I, \quad w^0 = [0 \quad 0 \quad 0]^T,$$

and

$$\Gamma = \text{diag}[8 \quad 2 \quad 4.75].$$

In the simulation, we consider the platform having eccentricity with  $d_1 = d_2 = 0.02$  m, and the mass has variation  $\Delta m_b = 3$  kg. The wheels' contact frictions are assumed with uncertainty

$$\Delta f = [0.05 \quad 0.05 \quad 0.05 \quad 0.05]^T \text{ N}.$$

The simulation results are shown in **Figure 4**. **Figure 4(a)** depicts the tracking performance of the  $X_I$ - and  $Y_I$ -axes translation, and the orientation  $\theta$  variation, and **Figure 4(b)** shows their tracking errors. We know that the tracking errors along the  $X_I$ - and  $Y_I$ -axes are within  $-0.0063$  -  $0.0056$  m and  $-0.0065$  -  $0.0062$  m, respectively, and the orientation error is within  $-0.57^\circ$  -  $0.64^\circ$ . The corresponding control torques of the four Mecanum wheels are shown in **Figure 4(c)**. The adaptation processes of the uncertainty compensation term's parameters vector  $\hat{w}_m(t)$  are shown in **Figure 4(d)**. And the geometric center's moving trajectory in the  $X_I - Y_I$  plane is shown in **Figure 4(e)**.

Considering the same uncertainties as the first case, a second simulation is considered to show the pure rotational control performance. The adaptive control parameters are selected as below:

$$\rho = 0.002, \quad c_1 = 0.0001,$$

$$K_1 = \text{diag}[9000 \quad 9000 \quad 2000],$$

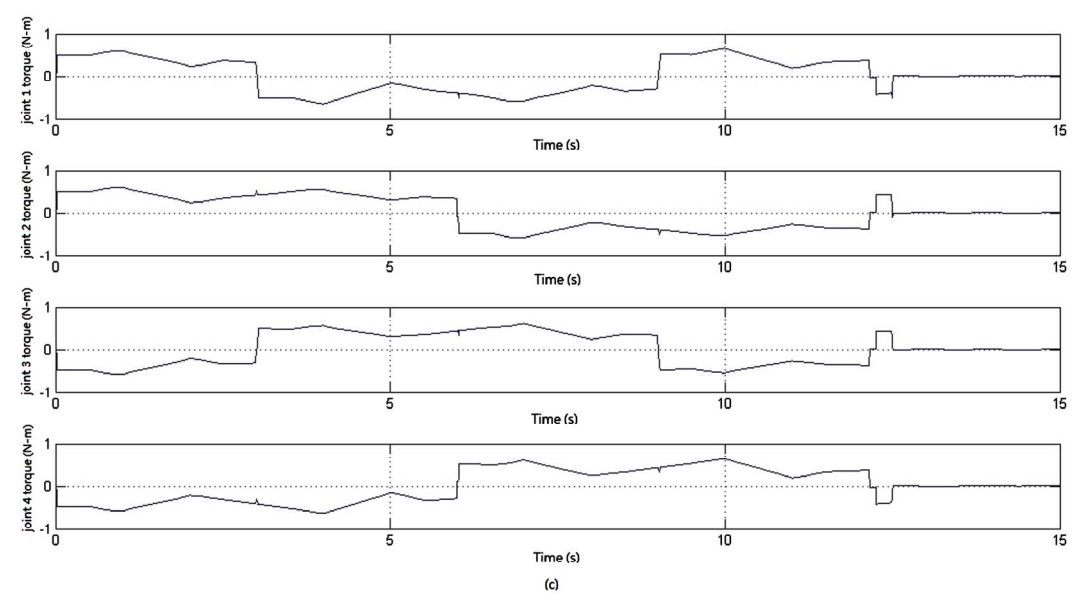
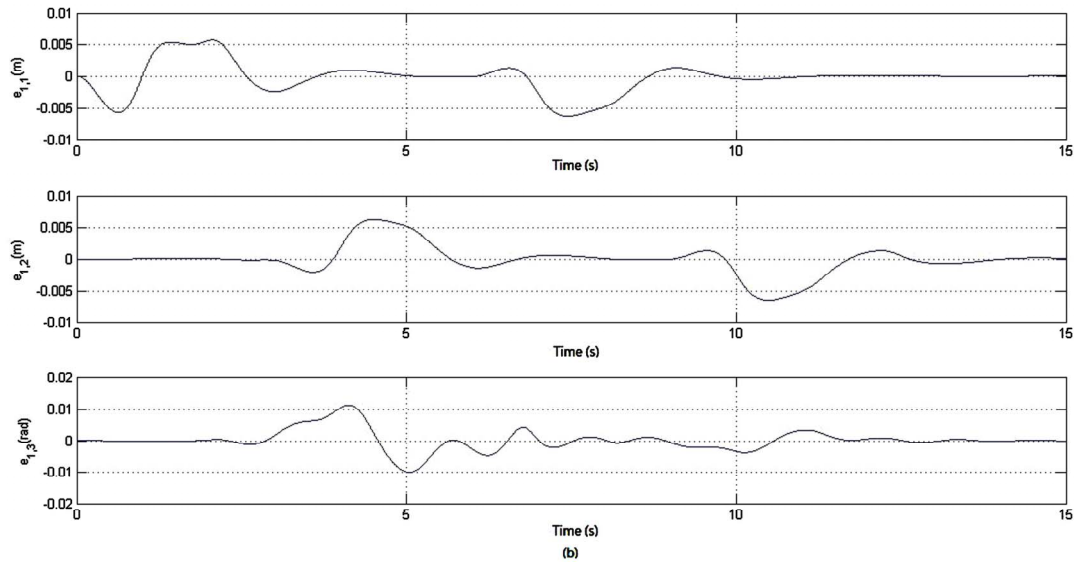
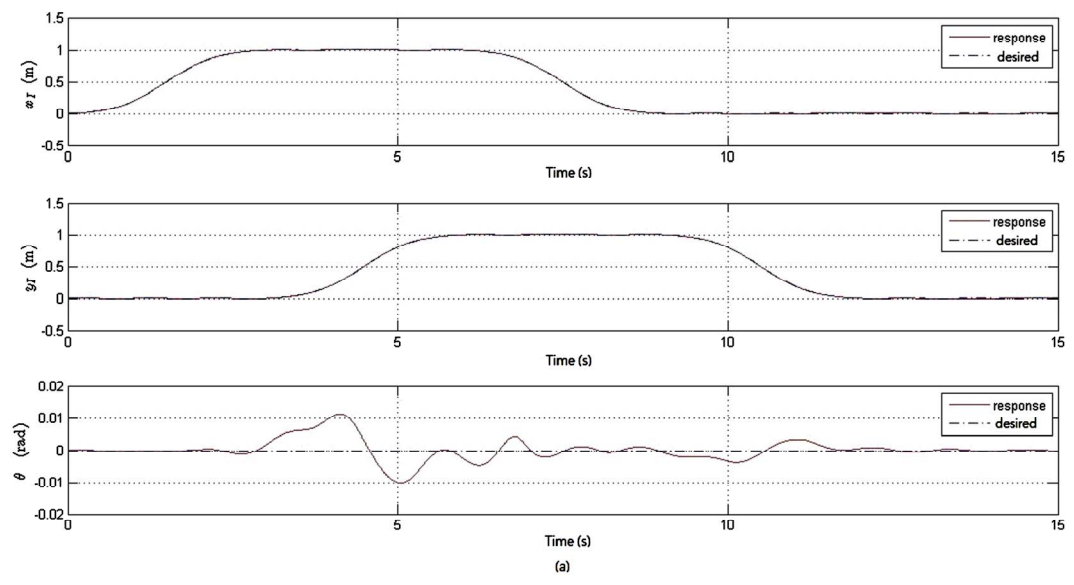
$$K_2 = \text{diag}[2000 \quad 900 \quad 1000],$$

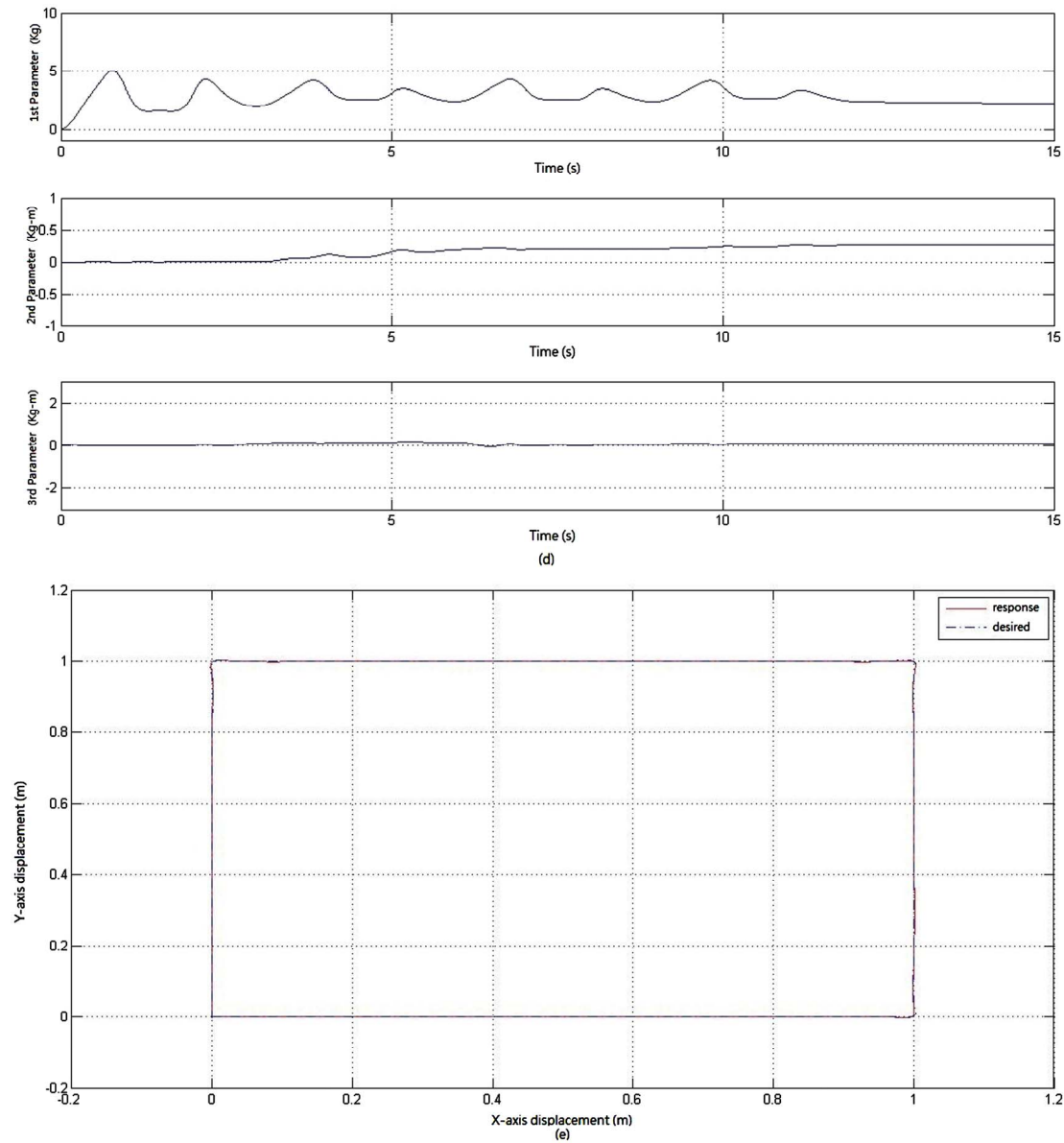
$$\sigma = \text{diag}[0.0001 \quad 0.03 \quad 0.03], \quad w^0 = 0,$$

and

$$\Gamma = \text{diag}[5 \quad 0.001 \quad 0.03].$$

Simulation results of the pure rotation case are shown in **Figure 5**. **Figure 5(a)** depicts the tracking performances of the  $X_I$ - and  $Y_I$ -axes translation, and the orientation  $\theta$ , and **Figure 5(b)** shows their tracking errors. We know that the undesired displacement along the  $X_I$ -





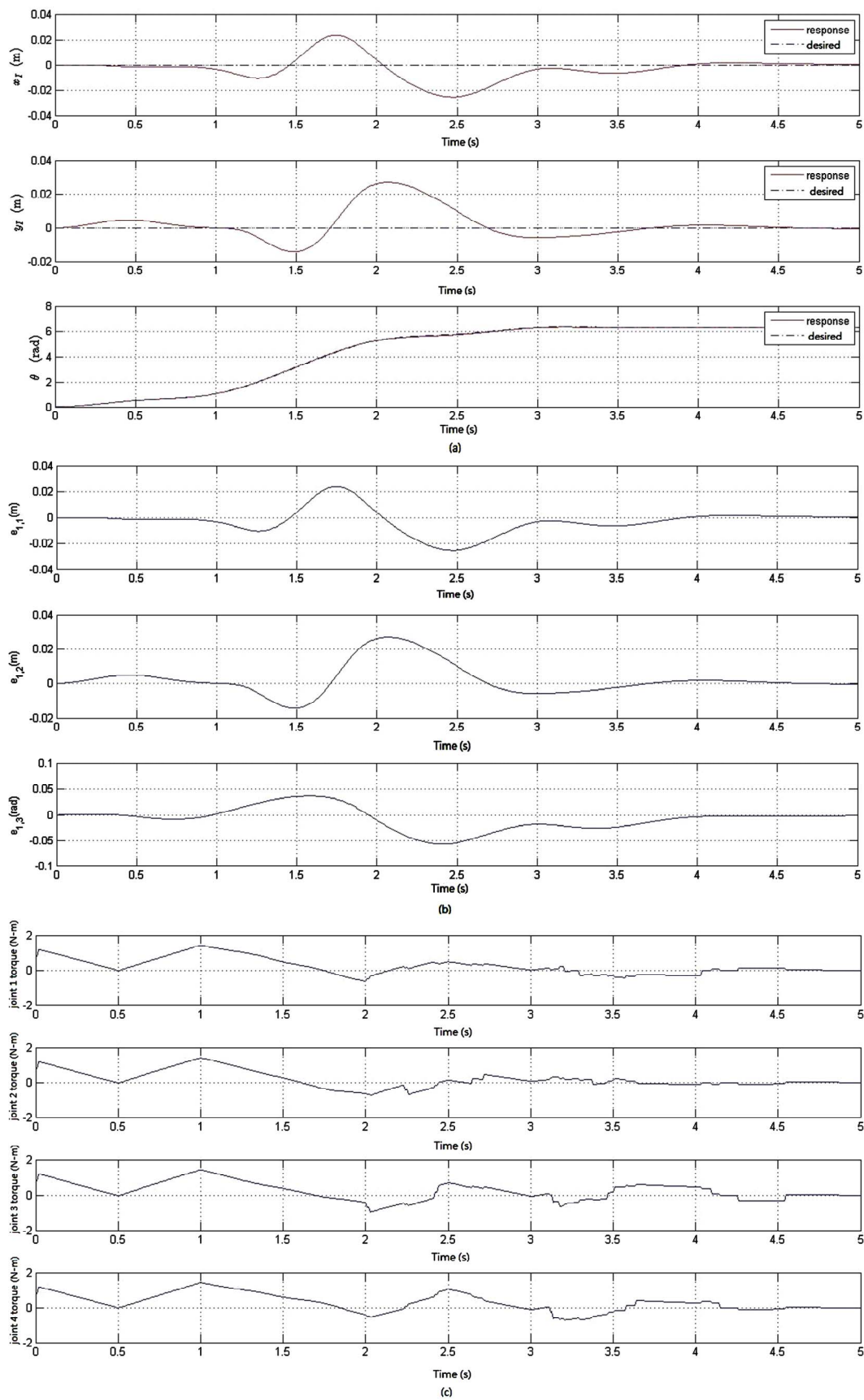
**Figure 4.** Simulation results of the pure translational case. (a)  $x_I, y_I$ , and  $\theta$  responses; (b) Tracking errors; (c) Control torques,  $\tau_1 \sim \tau_4$  (N·m); (d) Compensator parameters adaptation  $\hat{w}_m(t)$ ; (e) Tracking results of geometric center,  $(x_I(t), y_I(t))$ (m).

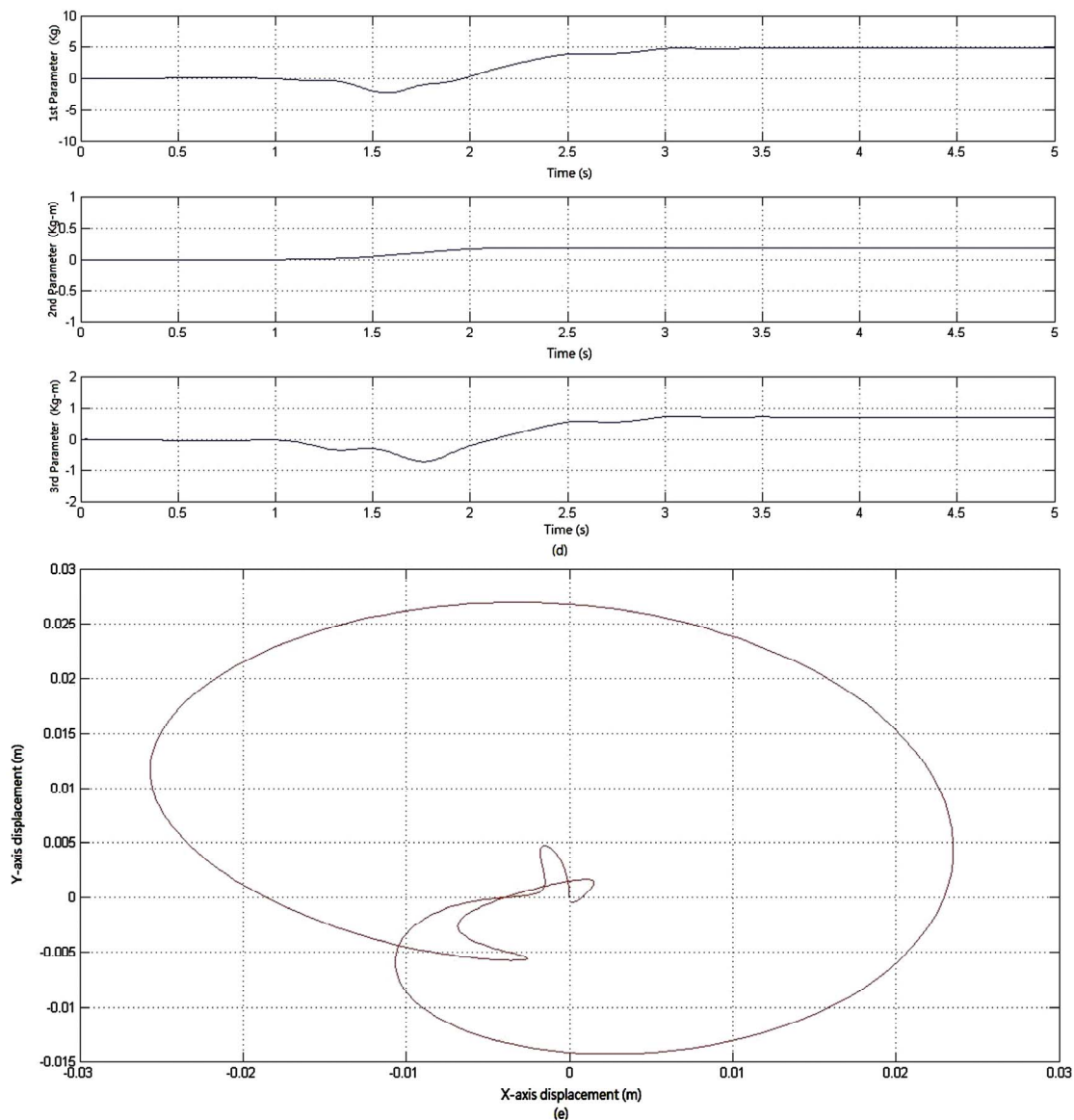
and  $Y_I$ -axes can be kept within  $-0.0257 - 0.0235$  m and  $-0.0143 - 0.0270$  m, respectively, and the orientation  $\theta$  error is within  $-0.0574 - 0.0362$  rad. The corresponding control torques of the four Mecanum wheels are shown in **Figure 5(c)**. The adaptation processes of the uncertainty compensation term's parameters vector  $\hat{w}_m(t)$  are shown in **Figure 5(d)**. And the geometric center's displacement in  $X_I - Y_I$  plane is shown in **Figure 5(e)**.

## 5. Conclusions

In this paper, Cartesian-space dynamics modeling and

stable adaptive control for a four-Mecanum-wheeled robot are considered. Based on the derived 3-DOF dynamics model considering the platform mass variation, eccentricity, and friction uncertainty, a nonlinear stable adaptive control law is derived using the backstepping method via Lyapunov stability theory. A nonlinear damping term is included in the control law to compensate for the estimation error, and the parameters adaptation law with  $\sigma$ -modification is considered for the uncertainty estimation. Computer simulations are presented to illustrate the control system performance. Real





**Figure 5.** Simulation results of the pure rotational case. (a)  $x_I, y_I$ , and  $\theta$  responses; (b) Tracking errors, (c) Control torques,  $\tau_1 \sim \tau_4$  (N·m) ; (d) Compensator parameters adaptation  $\hat{w}_m(t)$  ; (e) Displacement of geometric center  $(x_I(t), y_I(t))$  (m) .

implementation study using the suggested control law with microcontroller deserves future consideration.

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